

# NASA Contractor Report 172214



## MINIVER UPGRADE FOR THE AVID SYSTEM

### VOLUME III: EXITS USER'S AND INPUT GUIDE

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## FOREWORD

This final report presents work which was conducted for Langley Research Center (LaRC) in response to requirements of Contract NAS1-16983. The work presented was performed by REMTECH, Inc., Huntsville, Alabama and is entitled "MINIVER Upgrade For The AVID System". The final report consists of three volumes.

VOLUME 1: LANMIN User's Manual

VOLUME 2: LANMIN Input Guide

VOLUME 3: EXITS User's and Input Guide

The NASA technical coordination for this study was provided by Ms. Kathryn E. Wurster of the Vehicle Analysis Branch of the Space Systems Division.

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## Section 1.0

### INTRODUCTION

The EXITS code described in this document is a thermal analysis tool which allows the user to rapidly predict thermal protection system performance for advanced space transportation vehicle reentry. Design and weights optimization can be accomplished by repeated analysis within the constraints and guidelines of system performance.

EXITS is designed to run interactively on a small or mainframe computing system in conjunction with the LANMIN code. LANMIN, as described in Volume I and II is run, given a trajectory, geometry, heating rate methods, etc., to provide an input file containing the thermal environment information needed for the body points specified. Information flow for this process is depicted in Figure 1.1.

The user then calls EXITS in an interactive mode and sets certain input parameters, start time, end time, print time, etc. and defines the TPS structure by selecting structure types from a menu presented to him. In its present form, the menu contains seven different structure types, including an ablator, slab, radiation gap, etc. By stacking these structures, the entire TPS is defined and a nodal mesh is automatically generated. EXITS thus uses the LANMIN generated input file and calculates the temperature history of each node through the structure.

During the calculation, all of the heats are integrated and printed out. These include the convected, reradiated, sensible heat, ablated heat, and advected heat. A total energy balance is made to determine the method's conservation.

EXITS uses an explicit (Euler) integration of the energy equation using equivalent radiation conductors where internal or reradiation exists. An abla-

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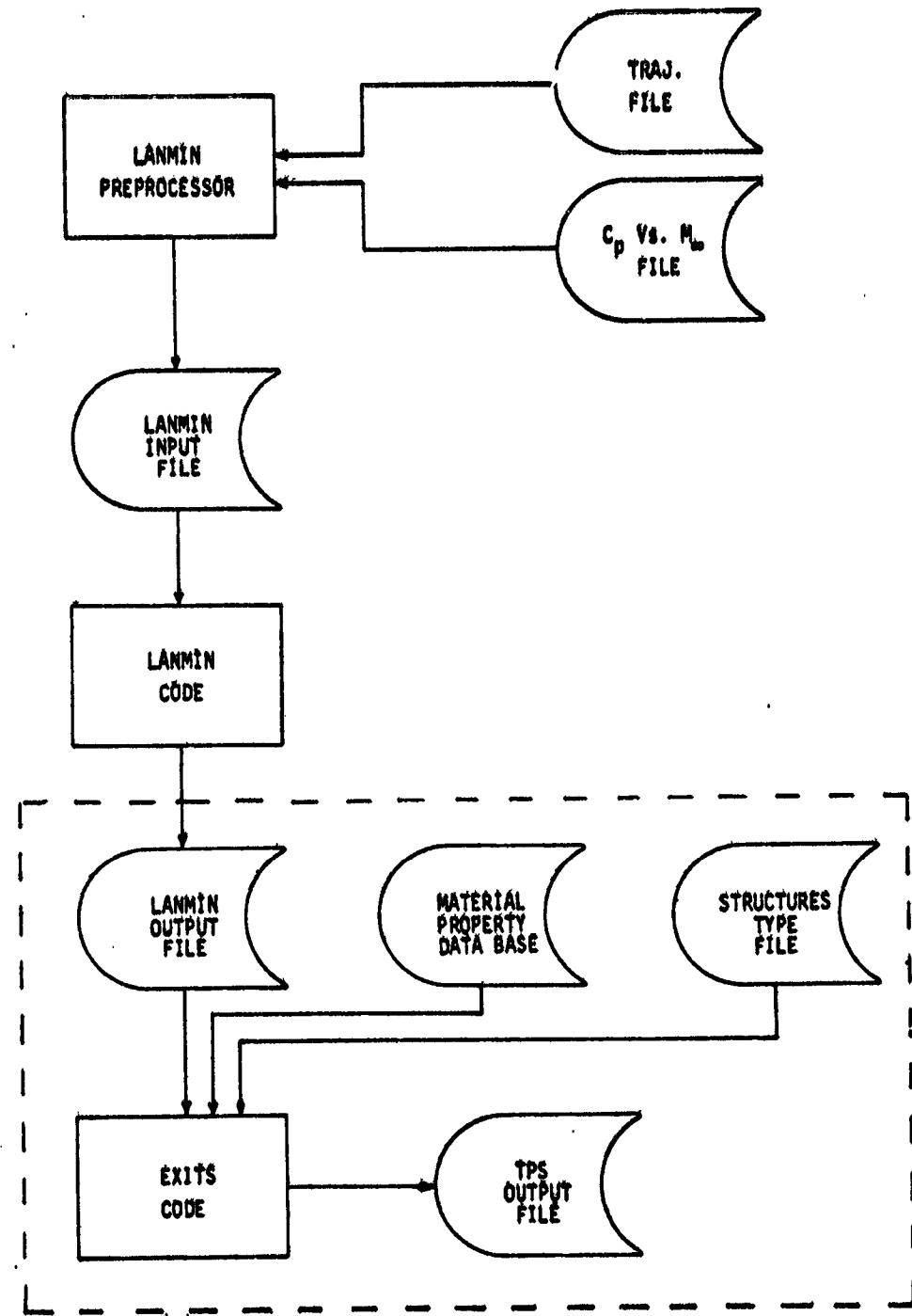


Fig. 1.1 Information Flow For Thermal Protection System Analysis

tion routine has been included using a simple ablator-sublimer model which includes the latent heat and heat required for the moving interface (advected). For materials with high thermal conductivity (aluminum, copper, etc.), a thermally "thin" structure has been included to avoid the time step problems explicit methods have with these materials. Details of these methods are described in the Technical Discussion, Section II.

The program structure, flow charts, etc. are given in Section III, Description of Program. Each subroutine is described in Section IV.

Input and Output data is described in Sections V and VI respectively. Finally, conclusions and recommendations are presented in Section VII. A listing of the code is presented in the Appendix.

## Section 2.0

### TECHNICAL DISCUSSION

This section describes the methods used in calculating the thermal response of the TPS structure. A basic energy balance performed at each node during the time marching using an explicit Euler integration forms the basis of the code. Special methods are used to describe the response of the ablator-sublimer and thermally thin structures. However, when complicated structures are used, the program logic branches off and constructs equivalent thermal networks, and from solutions of these networks, equivalent thermal conductance is computed and placed into the primary thermal network.

Presently, EXITS contains the capability to analyze seven different structure types. These are listed below as follows:

STRUCTURE TYPE	- - -	NUMBER
SLAB		1
RADIATION GAP		2
HONEYCOMB		3
CORRUGATED		4
Z STANDOFF		5
THIN SKIN		6
ABLATOR SUBLIMER		7

The methods used for analysis of the slab, thin skin, and ablator-sublimer are contained in this section. The methods for the radiation gap, honeycomb, corrugated, and Z-standoff structure are actually the same as the slab so therefore the logic used to compute the equivalent conductivity is not presented here but is given in Section IV, Description of Subroutines.

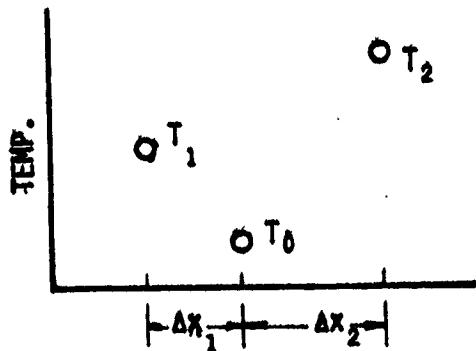
## 2.1 SLAB MODEL

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The thermal slab model is the finite difference representation of the heat conduction equation and is used to obtain the temperature response of the slab and ablator structure types. To obtain a finite difference numerical solution to the heat conduction equation, the derivatives in space and time are replaced by finite difference analogs. The heat conduction equation for an isotropic material with one spatial dimension is:

$$\rho C \frac{\partial T}{\partial t} = k \frac{\partial^2 T}{\partial x^2}$$

The space derivatives can be represented in the following manner referring to the temperature distribution depicted below.



Finite Difference Temperature Distribution

Expanding about point 0 using Taylor's series for the temperature at 2

$$T_2 = T_0 + \Delta x_2 \frac{\partial T}{\partial x} \Big|_0 + \frac{\Delta x_2^2}{2!} \frac{\partial^2 T}{\partial x^2} \Big|_0 + O(\Delta x_2^3) + \dots$$

In a like manner, expanding about 0 for the temperature at 1

$$T_1 = T_0 - \Delta x_1 \frac{\partial T}{\partial x} \Big|_0 + \frac{\Delta x_1^2}{2!} \frac{\partial^2 T}{\partial x^2} \Big|_0 - O(\Delta x_1^3) + \dots$$

Combining these two expressions, ignoring higher order terms and solving for

$$\frac{\partial^2 T}{\partial x^2} \Big|_0 \text{ at time step } n \text{ we find}$$

$$\frac{\partial^2 T}{\partial x^2} \Big|_0^n = \frac{2}{(\Delta x_1 + \Delta x_2)} \left[ \frac{T_1^n \Delta x_2 + T_2^n \Delta x_1 - T_0^n (\Delta x_1 + \Delta x_2)}{\Delta x_1 \Delta x_2} \right].$$

If we take a forward difference approximation for the time derivative as shown below

$$\frac{\partial T}{\partial \theta} \approx \frac{T_0^{n+1} - T_0^n}{\Delta \theta}$$

and substitute into the heat conduction equation we find

$$\rho C \left( \frac{\Delta x_1 + \Delta x_2}{2} \right) \frac{T_0^{n+1} - T_0^n}{2} = \frac{k}{\Delta x_1} (T_1^n - T_0^n) + \frac{k}{\Delta x_2} (T_2^n - T_0^n)$$

If the thermal capacitance is defined as

$$C_1 = \rho C V_1 = \rho C \left( \frac{\Delta x_1 + \Delta x_2}{2} \right)$$

and the conductors as

$$K_{1j} = \frac{k}{\Delta x_{1j}}$$

we have, upon substitution and some algebra

$$T_1^{n+1} = T_j^n + \frac{\Delta \theta}{C_1} \sum_{j=1}^2 K_{1j} (T_j^n - T_1^n).$$

This expression is the basis of the thermal balance at each node in the conduction network. However, for nodes adjacent to a radiation gap or structure in which the heat transfer mechanism is not by pure conduction, we can form equivalent conductors.

The maximum stable time step,  $\Delta\theta$ , which can be taken can be found by rearranging our finite difference algorithm as follows

$$T_1^{n+1} = T_1^n \left( 1 - \frac{\Delta\theta}{C_1} \sum_j K_{1j} \right) + \frac{\Delta\theta}{C_1} \sum_j K_{1j} T_j^n$$

and noting that the coefficient of  $T_1^n$  must remain positive for all  $\Delta\theta$ . A negative coefficient would mean that the greater the temperature at time step n, the less the temperature at time step  $n + 1$  which would not make sense. We now have

$$1 - \frac{\Delta\theta}{C_1} \sum_j K_{1j} \geq 0$$

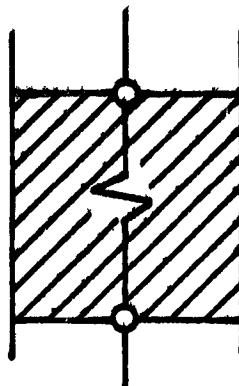
or

$$\Delta\theta \leq \frac{C_1}{\sum_j K_{1j}} .$$

This criteria is used for all cases except the thermally thin sections. To insure stability especially when the  $K_{ij}$ 's are nonlinear radiative conductors, the time step is divided by the input parameter STAB.

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A major feature in the development of this code was the thermal element, which consists of a thermal mass and a heat transfer path between two nodes located at the ends of the element shown below



Typical Thermal Element

Half of the thermal mass of an element is assigned to each node. The thermal elements are then stacked to define the complete thermal protection system. The slab and ablator materials are divided into several elements by the program. All other structure types consist of a single thermal element which are stacked upon each other sharing their common node points.

Constructing the network in this manner introduces slight errors where structures of varying capacitance are adjacent to one another and also at the surface node. In these cases, the node is not placed in the exact center of the thermal mass, however, energy is conserved.

## 2.2 COMPARISON WITH ANALYTICAL SOLUTION

As a check on the accuracy of the numerical algorithm, the solution was compared to an analytical solution, Ref. 1, of the partial differential equation. The convective heating of a plate of thickness  $2\delta_1$  from both sides is analogous to heating of a slab of thickness  $\delta_1$  from one side with an adiabatic

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backwall boundary condition. If the convective heat transfer coefficient,  $h$ , is held constant then the following boundary conditions will apply, where  $T = t - t_f$ .

$$T = T_1 \text{ at } \theta = 0$$

$$\frac{\partial T}{\partial x} = 0 \text{ at } x = 0 \text{ (center of slab)}$$

$$+ \frac{\partial T}{\partial x} = \frac{h}{k} T \text{ at } x = \pm \delta_1 \text{ (surface).}$$

The product solution is found to be

$$\frac{T}{T_1} = \frac{t - t_f}{t_1 - t_f} = 4 \sum_{n=1}^{\infty} \left( \frac{\sin M_n}{2M_n + \sin 2M_n} \right) e^{-M_n^2 \theta} \cos M_n \left( \frac{x}{\delta_1} \right)$$

where  $M_n$  are the roots of the transcendental equation

$$N_u = M_n \tan M_n$$

$N_u$  being the Nusselt number given as

$$N_u = \frac{h\delta_1}{k} .$$

Nomenclature for this case and a graphical representation of the transcendental equation are shown in Fig. 2.1.

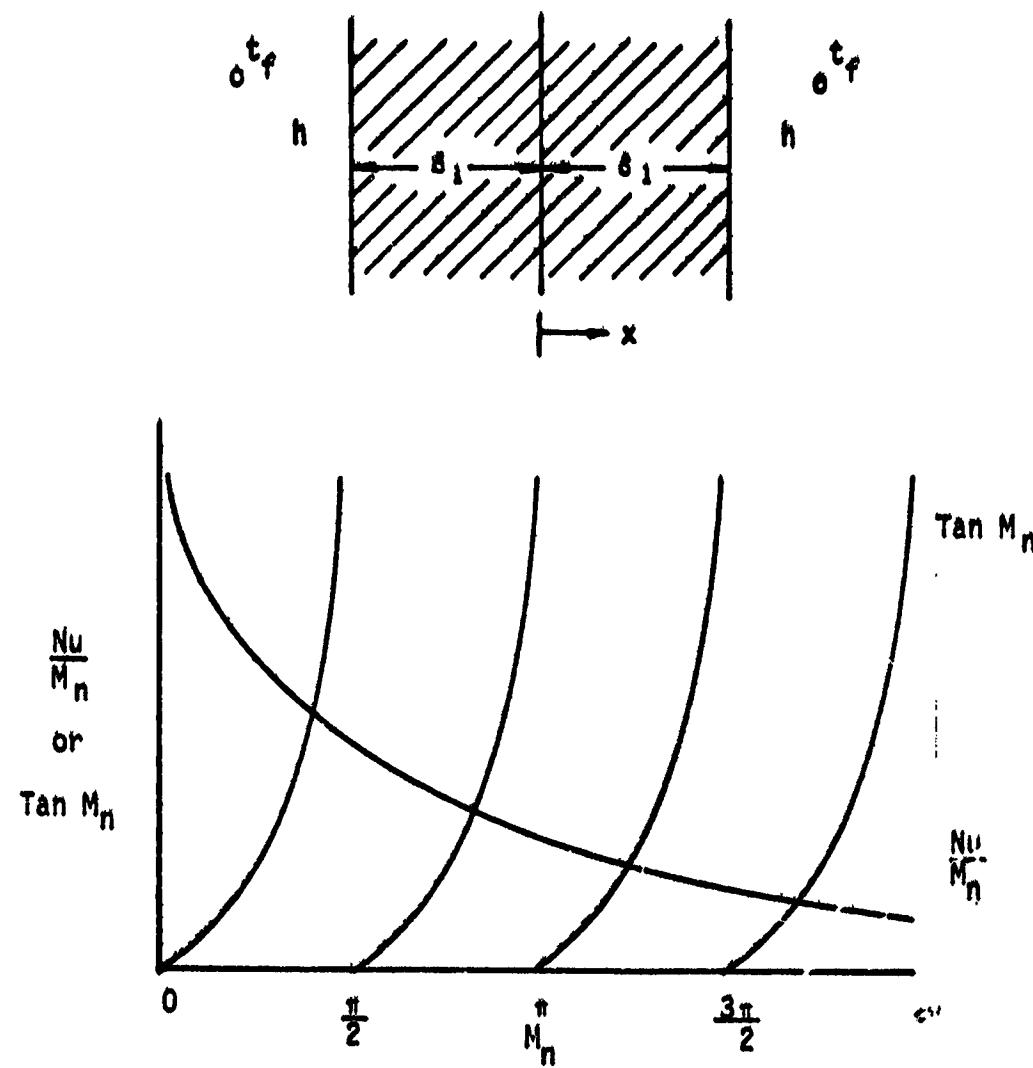
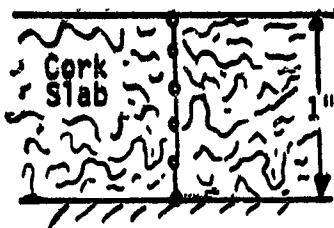


Fig. 2.1 Infinite Slab Heated From Both Sides  
With Graphical Solution For  $M_n$

A digital computer program was written to find the roots  $M_n$  and evaluate the analytical solution. Various numbers of terms were taken in the infinite series to check for convergence. A satisfactory solution was found after 50 terms were used.

The test case consisted of a layer of cork one inch thick with an adiabatic backside model using six nodes through the thickness. A comparison of this case with the analytical solution is presented in Fig. 2.2. Agreement appears to be quite good.



## NOTES:

1. 6 Node Model, Cork Slab
2. Insulated Backside
3. Adiabatic Wall Temp = 10,416.6°R
4. Film Coefficient = .001276 Btu/ft<sup>2</sup>-°R

PROPERTIES  
Density 10 lbm/ft<sup>3</sup>  
Sp. Heat .04 Btu/lbm-°R  
Conductivity  $6.9 \times 10^{-6}$  Btu/lbm-ft-°R

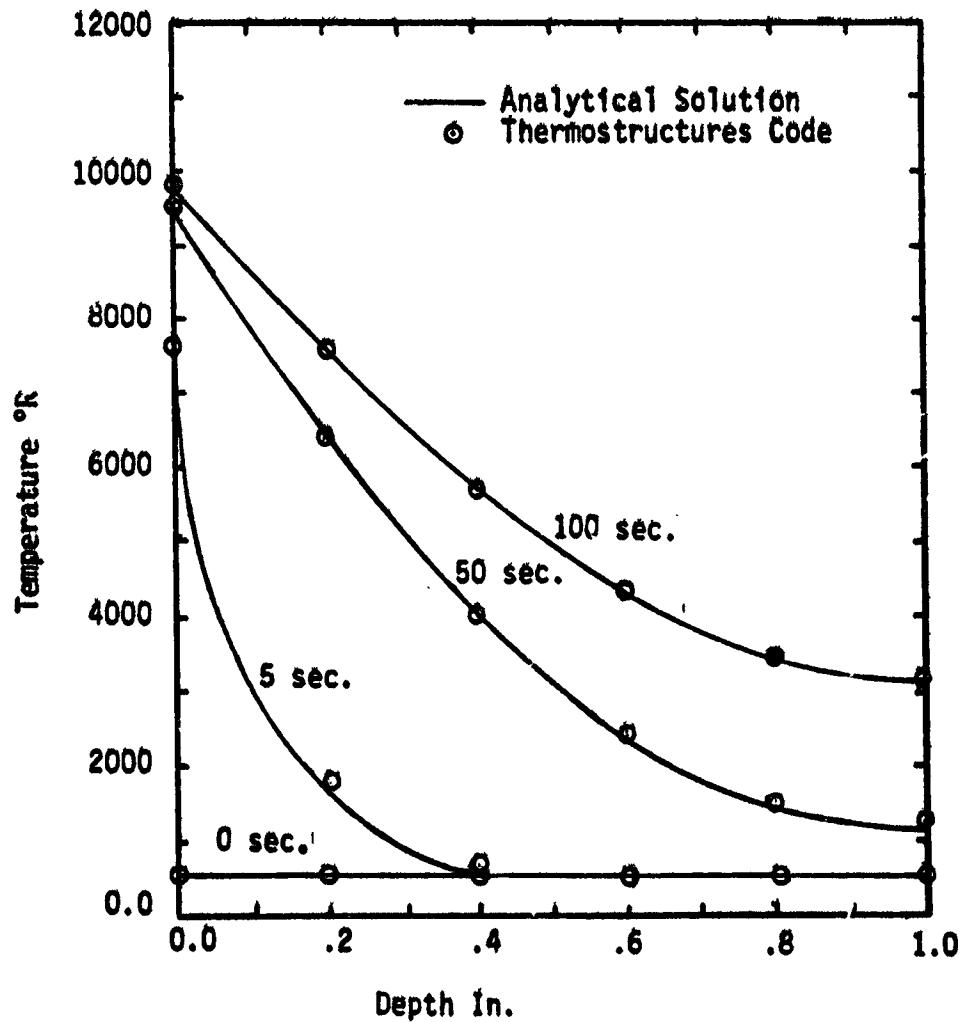


Fig. 2.2 Comparison of Thermostructures Code with Analytical Solution

### 2.3 THIN SKIN MODEL

For a slab type structure which is made of a material which has a high thermal conductivity, the temperature gradient through the material can be expected to be small for relatively thin sections and the heat fluxes encountered during reentry. If this gradient is to be modeled using the slab option, we see that the time step required to resolve this gradient will be very small since in general,  $\Delta\theta_1 \sim \frac{C_1}{\Sigma K_{ij}}$ , where one or more of the  $K_{ij}$ 's will be large. The small time step will result in long run times with very little increase in accuracy of the analysis.

If, however, we assume that the temperature gradient through the thin skin type structure is zero while still allowing heat to be stored in the structure, we can circumvent this time step restriction since we have effectively taken the conductors in the high thermal conductivity material out of the network. The resulting slab of material now becomes thermally "thin", i.e. no temperature gradient and long run times can be avoided.

Consider the generalized slab of material and model network in Figure 2.3

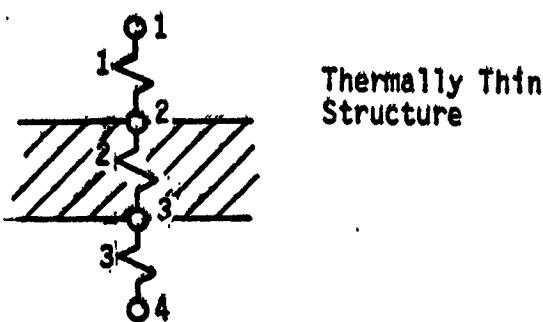


Fig. 2.3 Typical Thermally Thin Structure

If we write the equations for a heat balance at time step  $n + 1$  at nodes 2 and 3, we have

$$C_2 T_2^{n+1} = T_2^n C_2 + (T_1 K_1 + T_3 K_2 - T_2 K_1 - T_2 K_2) \Delta \theta$$

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and

$$C_3 T_3^{n+1} = T_3^n C_3 + (T_2 K_2 + T_4 K_3 - T_3 K_2 - T_3 K_3)^n \Delta \theta .$$

Now if we assume there is no temperature gradient between node 2 and 3 and add the two equations together to find the total energy stored at the end of time step  $n + 1$ , we have

$$C_2 T_2^{n+1} + C_3 T_3^{n+1} = T_2^n C_2 + T_3^n C_3 + (T_1 K_1 - T_2 K_1 + T_4 K_3 - T_3 K_3)^n \Delta \theta$$

Solving for the temperature of the thin section,  $T_{2-3}$ , we arrive at the algorithm for the thin skin section temperature below

$$T_{2-3}^{n+1} = T_{2-3}^n + \frac{\Delta \theta}{C_2 + C_3} (T_1^n K_1 - T_2^n K_1 + T_4^n K_3 - T_3^n K_3) .$$

Looking at this expression, we note that the conductor  $K_2$  has been eliminated and will no longer cause the small time step problem.

Considering the second law and finding a stable time step criteria can be accomplished as follows. Factoring  $T_{2-3}^n$ , we have

$$T_{2-3}^{n+1} = T_{2-3}^n \left[ 1 - \frac{\Delta \theta}{C_2 + C_3} (K_1 + K_3) \right] + \frac{\Delta \theta}{C_2 + C_3} \left[ T_1^n K_1 - T_4^n K_3 \right] .$$

The first term in brackets must remain positive for any stable time step so it follows that

$$\frac{\Delta \theta}{C_2 + C_3} (K_1 + K_3) \leq 1$$

or

$$\Delta \theta \leq \frac{C_2 + C_3}{K_1 + K_3} .$$

We see that the stable time step expression is in the familiar form  $\frac{C_j}{\Sigma K_{ij}}$  but does not have large conduction values which will cause the small time step problems.

The previous discussion considers the thin skin section to be a general case. For the case where the slab is on the surface exposed to the reentry environment or is located on the backside where the adiabatic boundary condition is used or where it exists by itself where both conditions exist, special logic is imposed.

#### 2.4 Ablator-Sublimer Model

The logic used to compute sublimer-ablator performance takes into account the energy management requirement at the material surface as follows:

1. The energy conducted away from the surface
2. The sensible energy stored in the material
3. The latent heat required to sublime the material
4. The convected or advected energy required due to the receding surface.

The numerical scheme devised to account for these effects is incorporated into the program's network by special logic which considers the moving boundary and the latent heat required to sublime the material using the slab logic. When the temperature of the surface remains below the temperature of sublimation, the thermal balance is performed just as it would be done in any nonablator material. If, however, at the end of any time step we see that the temperature has exceeded the sublimation temperature, the amount of energy that was required to exceed the sublimation temperature is computed and the surface node temperature is set to the sublimation temperature. The excess energy is then used to compute the amount of material which is sublimed.

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Considering the four node network shown in Figure 2.4, we see that the surface has

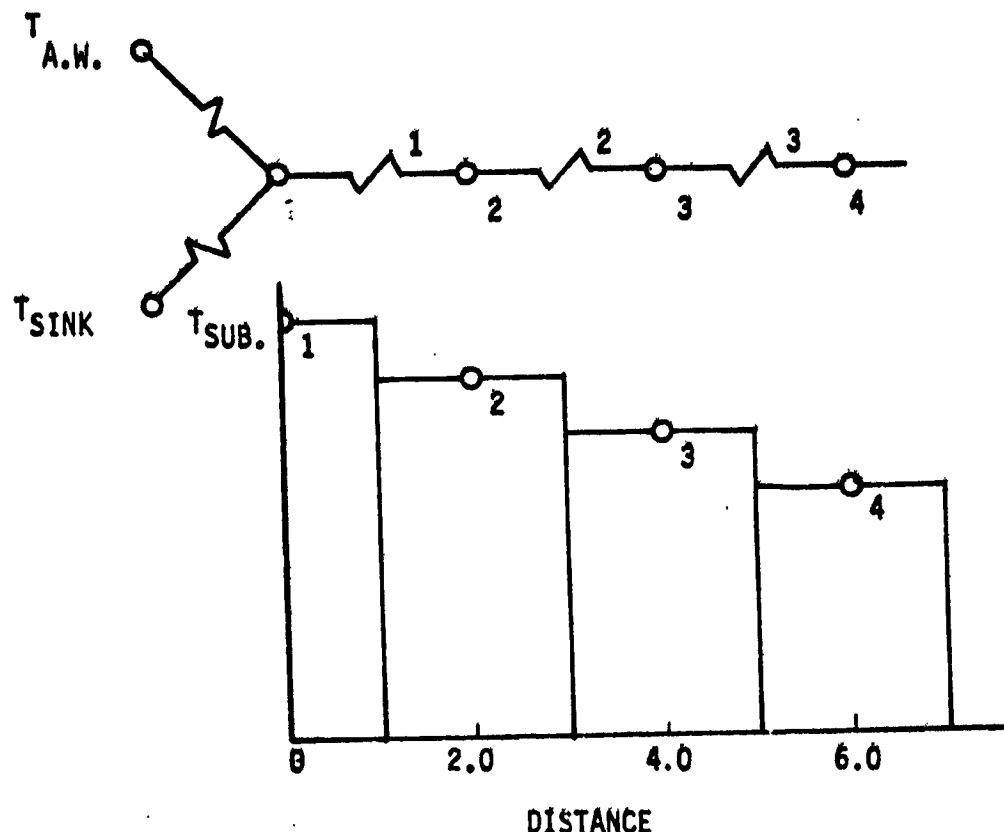


Fig. 2.4 Nodal Mesh And Temperature Distribution  
For Ablator-Sublimer

reached the sublimation temperature. Additional heat added to the surface which is not radiated or conducted away is heat which sublimes the surface material and advances the surface into the cooler material.

We first compute an excess amount of heat which was used to take the surface node temperature over the sublimation temperature with the following expression

$$\Delta q = C_1 (T_1 - T_{SUB.})$$

and then set

$$T_1 = T_{SUB.}$$

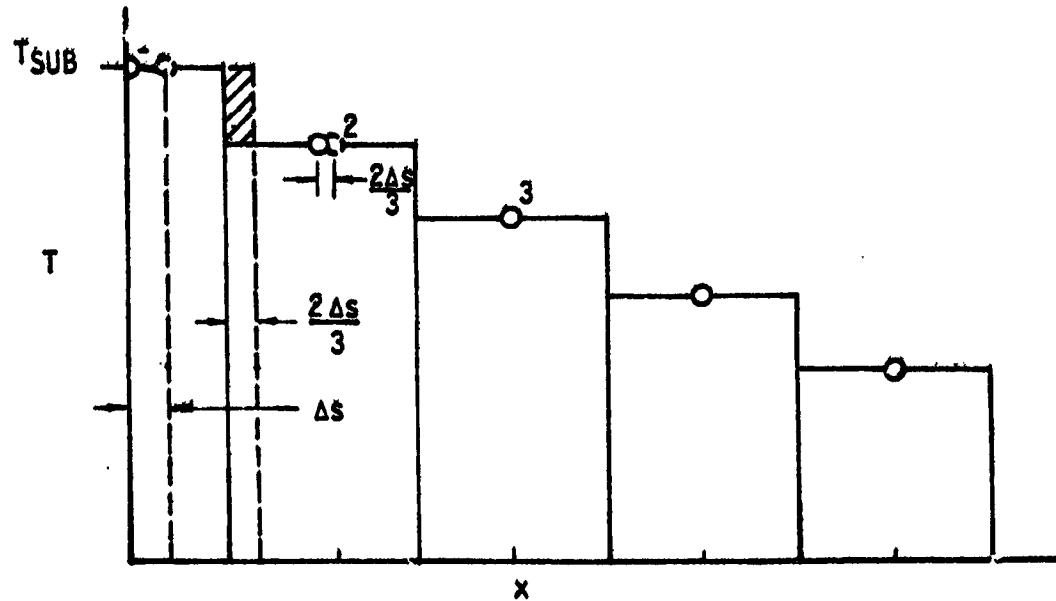
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Next, we compute the surface recession distance from the latent heat of sublimation,  $L$ , and the density as follows

$$\Delta S = \frac{\Delta q}{\rho L_{eff}} .$$

As the surface recedes, the melt line must also recede, so we move the boundary between node 1 and 2. This results in the mass in node 2 at temperature  $T_s$  being brought to the sublimation temperature  $T_{\text{sub}}$ . Thus, the energy added to the system must be taken into account.

If we look at Figure 2.5 below where the temperature through the ablator is shown and



**Fig. 2.5 Node Movement For Ablator-Sublimer**

one time step is  $\Delta S$ , we see that if the node boundary between 1 and 2 is moved  $2/3 \Delta S$  and node 2 is moved  $1/3 \Delta S$ , the material in node 1 and node 2 will be completely eliminated after a given number of steps. However, before we completely eliminate node 1 and 2, we stop when a prescribed amount of material is left in node 2 and raise its temperature to  $T_{\text{sub}}$ . Node 3 now becomes node

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number 2, and the remaining nodes are renumbered. The process now continues until node 2 is eliminated again.

If we now consider the original numbering scheme, we see that the node boundary between node 1 and node 2 is a moving boundary or looking at it in another way, node 1 is fixed in space (Eulerian) and nodes 2 and greater are fixed (LaGrangian) to a moving material. In this sense, we see that energy is convected or advected into node 1 and this energy must be supplied by the aerodynamic heating environment. Referring to Fig. 2.5, we can see that this amounts to

$$\rho C_p \frac{2\Delta S}{3} (T_{SUB} - T_2).$$

Since this energy must be supplied by the aerodynamic heating and is only required when ablation occurs, we adjust the latent heat of sublimation to account for this. We then compute an effective latent heat of ablation from the following expression

$$L_{eff}^{n+1} = \frac{\left( L_{eff}^n V_1^n + \frac{2}{3} \Delta S^{n+1} (L + C_p (T_{SUB} - T_2^n)) \right)}{V_1^n + \frac{2}{3} \Delta S^n}$$

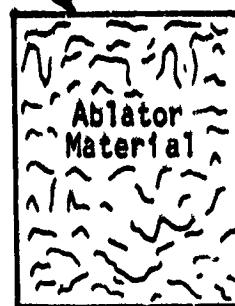
In the expression above,  $L$  is the actual heat of ablation  $L_{eff}^n$  is the effective heat of ablation from the last step and  $V_1^n$  is the volume of node 1 at the last time step.

In applying this method, heat conducted from node 1 to node 2 and 2 to 3 etc. is accounted for in the same manner as the slab described in Section 2.1.

An example of this procedure is shown in Figure 2.6 for a hypothetical ablator. Results are compared to a steady state analytical solution from Ref. 2.

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$$\dot{Q} = 300 \frac{\text{Btu}}{\text{ft}^2 \cdot \text{sec.}}$$



$$\rho = 100 \text{ lbm./ft}^3$$

$$C_p = .30 \text{ Btu/lbm. - } ^\circ\text{R}$$

$$K = 1.0 \times 10^{-4} \text{ Btu/ft. - sec. - } ^\circ\text{R}$$

$$L = 1276 \text{ Btu/lbm.}$$

$$T_{melt} = 5460 \text{ } ^\circ\text{R}$$

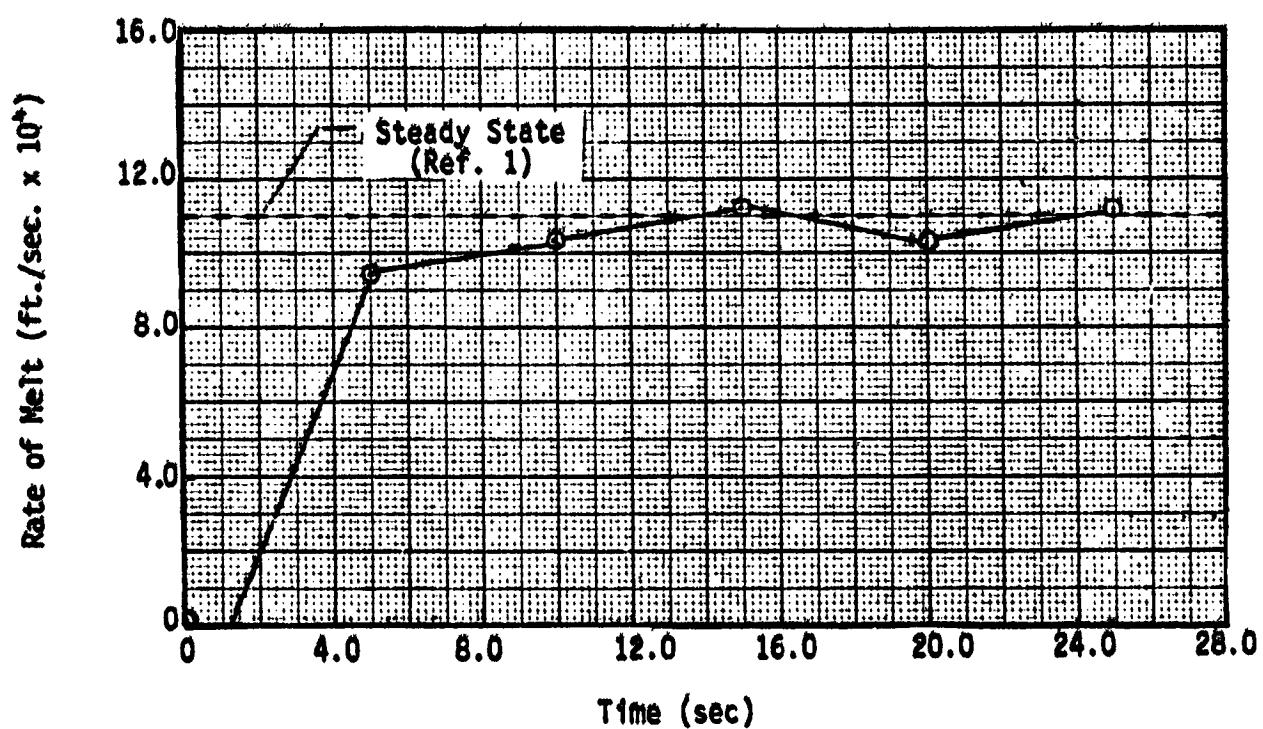


Fig. 2.6 Comparison of Results for the Recession Rate

## Section 3.0

### DESCRIPTION OF THE PROGRAM

An effort was made throughout the development of the EXITS code to keep the structure of the code as modular as possible and to define specific functions which could be broken off into subroutines. By in large, this has been accomplished, and as a result, the program capability can be expanded without extensive reprogramming.

The method of defining thermal structure types, i. e. slab, honeycomb, corrugated etc. facilitates the organization of the program since each structure type, with the exception of the slab and the ablator, consists of a conductor connecting two nodes located at the ends of the structure and capacitance, one half of which is assigned to each node. The slab and ablator are similarly defined with the exception being several nodes are placed within the structure.

The main driver contains calls to the primary functions or primary subroutines. These primary functions in turn call secondary routines which supply required information. The structure of the EXITS code is shown in Table 3.1. The routines are arranged so that the MAIN controls the program flow, calls input routines, contains the time marching iterative loop and creates the output file.

A more detailed flow chart and arrangement of the subroutines is shown in Figure 3.1. Each subroutine's calling structure is shown in Table 3.2. A full description of each subroutine is given in the next section.

No blank common is used, only named common and it's location is shown in Table 3.3.

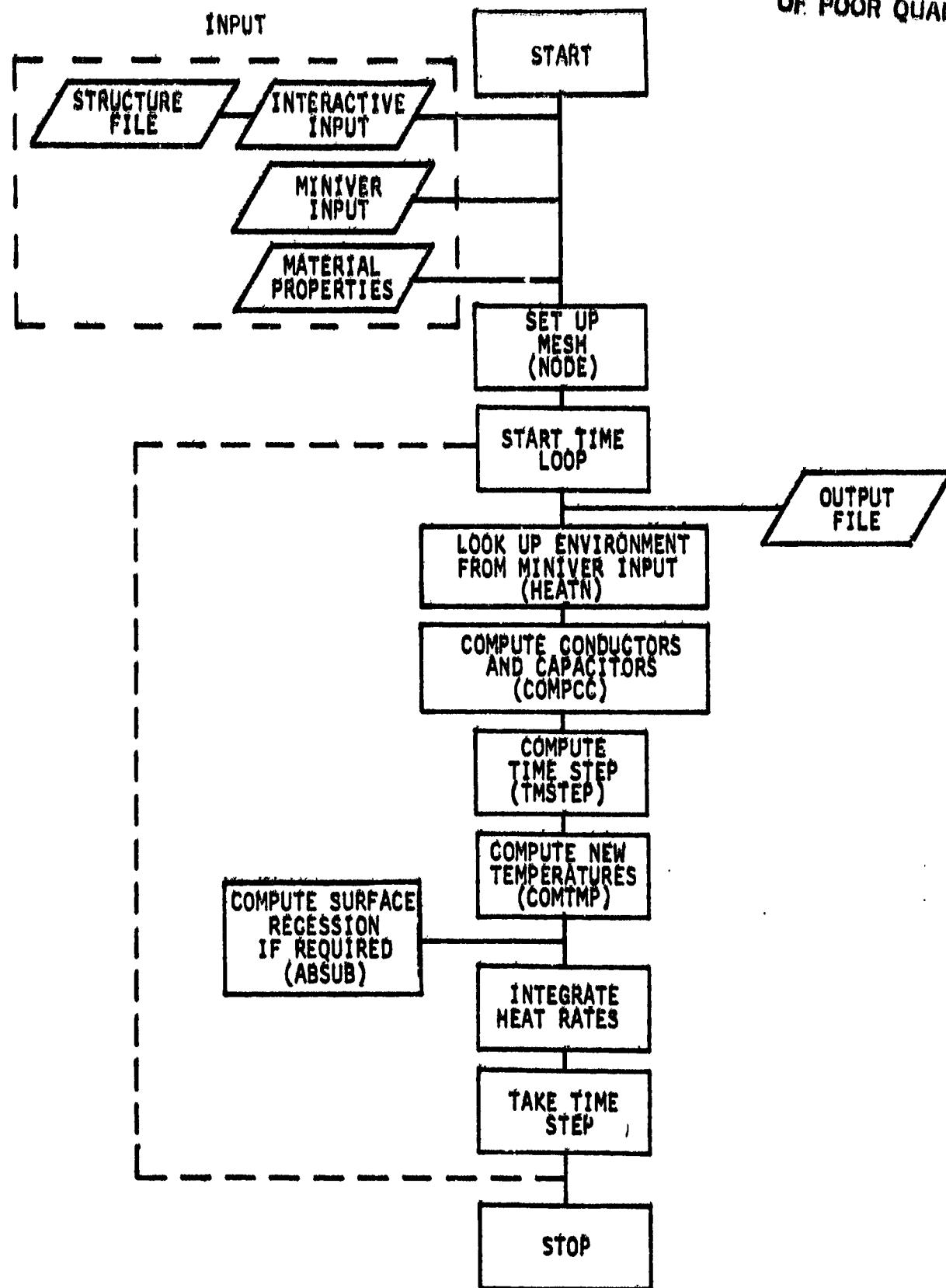


Table 3.1 Simplified Functional Structure

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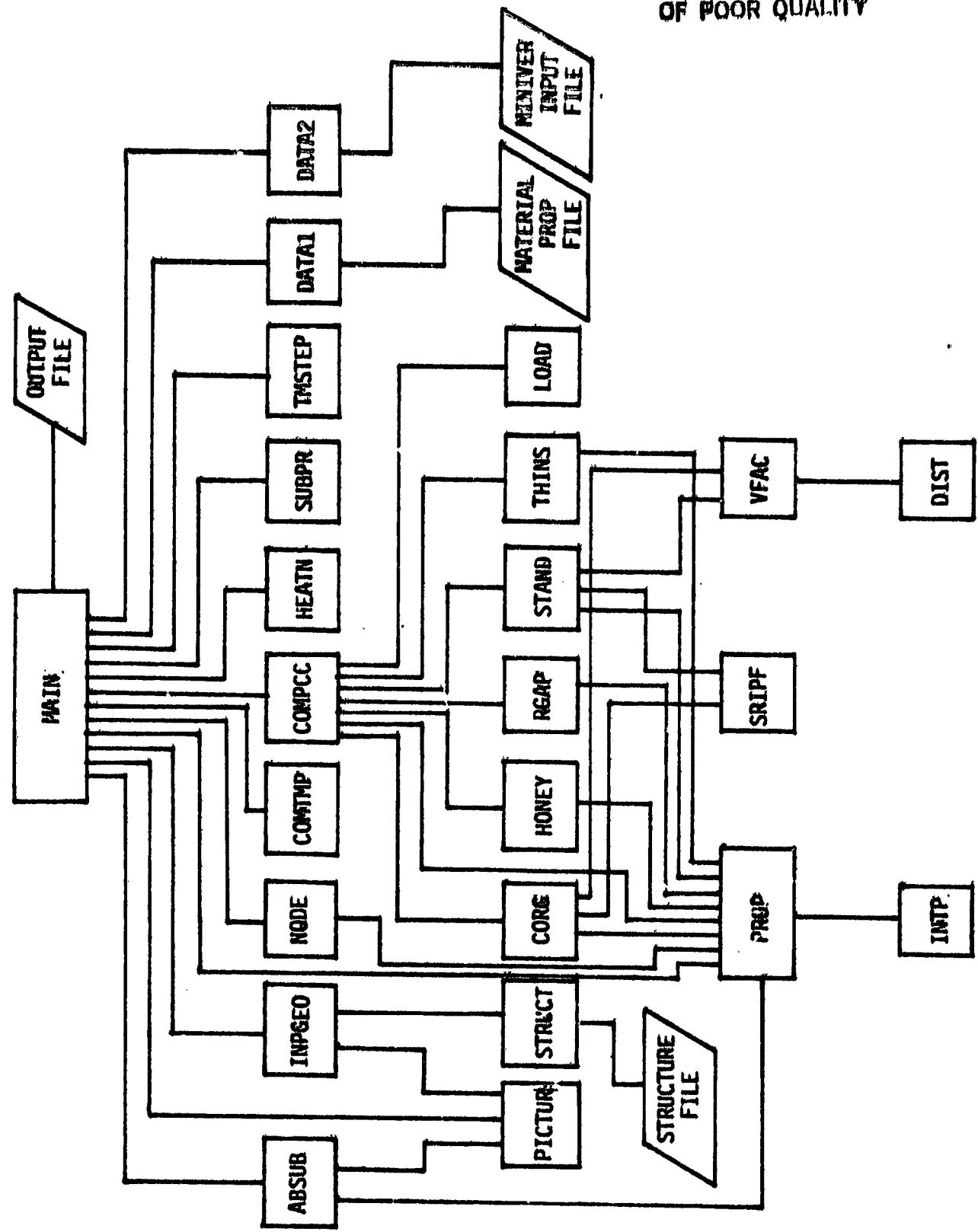


Fig. 3.1 EXIIS Subroutine Flow Chart

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CALLS SUBROUTINE	MAIN	PROP	INTP	DATA1	COMPCC	STRUCT	LOAD	ABSUB	DATA2	PICTUR	COMPP	NODE	HEATN	VFAC	DIST	TMSTEP	INPGE0	SUBPR	SRIPF	CORG	HONEY	RGAP	STAND	THINS
MAIN	x								x	x	x	x	x	x	x	x	x	x	x	x	x	x	x	
PROP		x																						
INTP			x																					
DATA1				x																				
COMPCC				x																				
STRUCT					x																			
LOAD					x										x									
ABSUB						x																		
DATA2							x																	
PICTUR								x																
COMPP									x															
NODE						x																		
HEATN													x											
VFAC														x										
DIST															x									
TMSTEP										x	x													
INPGE0												x												
SUBPR													x											
SRIPF														x										
CORG								x							x									
HONEY								x								x								
RGAP								x							x									
STAND								x								x								
THINS								x									x							

TABLE 3.2 Subroutine Calling Structure

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ROUTINE COMMON \	MAIN	PROP	INTP	DATA1	COMPEC	STRUCT	LOAD	ABSUR	DATA2	PICTUR	CONTMP	NODE	HEATN	VFAC	DISTI	TRSTEP	INREGO	SUBPR	SRITP	CORG	HONEY	RGAP	STAND	THINS		
ENVIR	x							x				x														
GAP	x			x		x	x					x				x				x	x	x	x	x		
INIT	x		x	x	x	x	x	x	x	x	x	x			x											
TAX	x		x	x	x	x	x	x	x	x	x	x														
TIME	x		x	x	x	x	x	x	x	x	x	x			x	x										
ARA	x		x	x	x	x	x	x	x	x	x	x			x											
LD	x		x	x	x	x	x	x	x	x	x	x			x	x										
NODES	x		x	x	x	x	x	x	x	x	x	x														
CTMP	x											x														
CAC	x				x																					
PICT	x					x				x	x	x				x										
SUBLN	x						x		x	x	x	x					x									
TITLE	x		x	x	x	x	x	x	x	x	x	x				x										
PRESS	x		x	x	x	x	x	x	x	x	x	x					x			x	x	x	x	x		
SAVE	x						x													x	x	x	x	x	x	
DTA		x	x																							
CSUB		x																		x						
TITL2		x											x				x									
FACT														x		x	x	x	x	x	x	x	x	x	x	
SF															x					x	x	x	x	x	x	x

Subroutine PROP Contains No Common Statement.

TABLE 3.3 Named Common Statements And Subroutine Locations

## Section 4.0

### DESCRIPTION OF SUBROUTINES

This section describes the main driver and the twenty three subroutines that comprise the EXITS code. The description presented is an overall description which may include the subroutine function, method and program logic.

#### 4.1 MAIN

The main driver of the EXITS code controls the major functions of the program and also contains the output calls. Logic for calling the major subroutines is found in this part of the program. Constants and control flags and initial values of integrated heat loads are first set to their prescribed values. The interactive input routine INPGEO is called. Then initial temperatures are set and subroutines DATA1 and DATA2 are called to obtain the material property and environment data. A call to subroutine NODE sets up the thermal network of nodes, capacitors and conductors for each body point. A call to PICTUR sends a depiction of the structure and node location to the line printer.

The temperature integration starts with the time loop after further initialization. Output and units conversion take place within the time loop which is controlled by the print flag NPEG. The adiabatic wall enthalpy, film coefficient and pressure is found from a call to HEATN. If an ablator-sublimer is used, the ablator properties are found from two calls to SUBPR. Values for the conductors and capacitors are computed from COMPCC. The time step DTSM is calculated from stability criteria and the user supplied parameter STAB. Temperatures for all nodes in the structure are computed at the end of the time step by subroutine COMTMP. If an ablator is called for, the recession and renumbering of the node and conductor sequence is done in ABSUB. Finally, at

the end of the time integration loop, the heat loads, sensible heat, advected heat, and sublimed heat are integrated and time is increased by the amount DTSM. A check is made to see if the number of steps or time has exceeded the input values and if not control is returned to the top of the integration loop.

#### 4.2 SUBROUTINE PROP

This subroutine returns thermophysical properties for the material specified by the variable MAT as a function of temperature, T1, and pressure, P. Subroutine INTP is called with T1 and P as the independent arguments after the property table numbers are computed for the density, specific heat, conductivity and emissivity. Properties for ablator material, heat of ablation and temperature of ablation, are not computed by PROP. These properties are found by the subroutine SUBPR.

#### 4.3 SUBROUTINE INTP

This subroutine linearly interpolates in either two or three dimensional arrays for material properties as a function of temperature or as a function of temperature and pressure. The arguments X, P, N, Y, are respectively, temperature, pressure, table number and the returned property. The subroutine interpolates in both monovariate and bivariate tables. Ablator-sublimer properties, sublimation temperature and heat of sublimation, are not found by INTP but are found by SUBPR. Data for the properties are stored in the array CC(N,J) for the monovariate arrays and CC(N,J) and BSV(N,JT,IL) arrays for the Bivariate tables. The arrangement of data in the arrays for the two types of tables are shown in the following examples. Data for the monovariate tables are shown in Table 4.1.

CC(N,1) —— 5.	NYLON PHEN CONDUCTIVITY
CC(N,2) —— 0.0	1.39E-5 — CC(N,3)
CC(N,4) —— 460.0	1.39E-5 — CC(N,5)
	660.0    1.94E-5
	910.0    2.50E-5
CC(N,10) —— 1000.0	2.50E-5 — CC(N,11)

TABLE 4.1 Arrangement of Data For Monovariate Properties

Data for the bivariate table are shown in Table 4.2.

Initially a check is made on the sign of CC(N,2) to determine if the data for table N is monovariate or bivariate. If the sign is positive, the data is searched to find the two temperatures to interpolate between and a straight line interpolation is used.

If the sign on the variable CC(N,2) is negative, then a bivariate table is assumed and the independent variable array, pressure stored in CC(N,3) to CC(N,(-CC(N,2)+2)), is searched to find the increment in the pressure direction. The temperatures are then searched to find the two temperatures between which the interpolation is to be performed and the pressure increment applied. Finally, with two interpolated values found in the pressure direction, the temperature increment is applied and the final value is computed and returned through Y in the argument.

#### 4.4 SUBROUTINE DATA1

Subroutine DATA1 finds the material property data for the materials given, renumbers the material identifiers, MATS(I,LT,IM), and stores the material property data in arrays to be used later in the thermal analysis. This routine first reads through the data and picks out the data from the materials used in the model. Material identification numbers are then changed to the order in which they appear to minimize the storage requirement in the CC(I,J) and BSU(I,J) arrays.

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CC(N <sup>**</sup> , 1)	15	LI-900	CONDUCTIVITY	CC(N, 5)	.....	CC(N, K*)
CC(N, 2)	-6.0	0.0	1.389E-6	1.389E-6-J->2.083E-6	2.12	21.16
CC(N, K+1)	0.0		1.389E-6	1.389E-6	2.083E-6	4.166E-6
CC(N, K+2)	-210.0		1.389E-6	1.389E-6	2.083E-6	6.060E-6
460.0			2.083E-6	2.083E-6	2.777E-6	6.060E-6
710.0			2.555E-6	2.555E-6	3.472E-6	6.944E-6
960.0			3.472E-6	3.472E-6	4.638E-6	8.777E-6
1210.0			4.861E-6	4.861E-6	7.666E-6	1.111E-5
1460.0	1		6.472E-6	6.472E-6	9.027E-6	1.366E-5
1710.0			8.555E-6	8.555E-6	1.088E-5	1.667E-5
1960.0			1.155E-5	1.155E-5	1.366E-5	2.014E-5
2210.0			1.575E-5	1.575E-5	1.275E-5	1.174E-5
2460.0			2.039E-5	2.039E-5	1.694E-5	2.130E-5
2760.0			2.683E-5	2.683E-5	2.172E-5	2.616E-5
2960.0			3.222E-5	3.222E-5	2.833E-5	3.222E-5
3260.0			4.277E-5	4.277E-5	3.416E-5	4.305E-5
CC(N, L <sup>**</sup> )	3460.0		5.277E-5	5.277E-5	4.500E-5	5.000E-5
					5.444E-5	6.080E-5
					7.277E-5	8.055E-5

TABLE 4.2 Arrangement Of Data For Bivariate Properties

- \*K = -CC(N, 1) + 2
- \*\*L = K + CC(N, 1)
- \*\*\*N = TABLE NUMBER

CC(N, 1) = NUMBER OF TEMPERATURE ENTRIES  
 CC(N, 2) = NUMBER OF PRESSURE ENTRIES

When a material is found or matched to the material specified, MAT(I,LT,IM), this routine reads the title card and next set of data cards according to the number specified. The next three sets are then read for a total of four tables. If the first entry in the independent array of any table is negative, the table is assumed to be bivariate and a different set of logic is used to store the data. Data for the monovariate tables are stored in an array, CC(N,J), where N is the table number. If the data is found to be bivariate, then the independent variables are stored in CC(N,J) array and the dependent variables are stored in the array BSV(N,J,K), Table 4.6. Each material property set in the property file must appear in the prescribed order, density, specific heat, conductivity, and emissivity. Units for these properties must be entered in BTU's, feet, seconds, pounds mass, pounds force, and degrees Rankine.

For ablator-sublimer material, the material property number of the fifth and sixth property is entered on the title card. When the same material identifier number is found, the temperature of sublimation and the heat of sublimation as a function of pressure is given as the fifth and sixth property and stored in the array CCS(I,J).

#### 4.5 SUBROUTINE COMPCC

Subroutine COMPCC computes the values of the conductors and capacitors for the network. The capacitor C(I) and conductor CD(I) values for the slab and ablator structure types are computed directly in COMPCC. Capacitor and conductor values for the other structures are found from routines called from COMPCC. Values for the conductors between the nodes for the slab and the ablator are found by calculating the distance between the nodes from the node position array, XX(I), and then finding the conductivity from the average temperature between the nodes. Conductor values are found from the expression

$$K_f = \frac{k}{\Delta x}$$

In the same loop that the conductors are computed, the capacitors are found. The mass of the material between the nodes is computed and multiplied by the specific heat. Since one half the mass is associated with each node, the capacitance value is divided by two and half of it is summed at each node. Capacitance value for each node is found from

$$C_i = \sum_{j=1}^2 \frac{\rho_j V C}{2.0}$$

where the summation on  $j$  is on the thermal mass adjacent to the node  $i$ . At this time, the mass of the structure is also computed and stored in XMAS.

For structure other than slab type, JN = 1, or ablator, JN = 7, COMPCC branches off to the following routines, Table 4.3.

JN	Structure Type	Subroutine
2	RADIATION GAP	RGAP
3	HONEYCOMB	HONEY
4	CORRUGATED	CORG
5	Z-STANDOFF	STAND
6	'THIN' SKIN	THINS

TABLE 4.3 Routines For Computing Effective Conductance

Before branching off however, subroutine LOAD is called. This routine takes information, (i.e. geometry, materials etc.) from named common, LD, and loads it into the named common, GAP. The subroutines called when JN = 2 through 6 compute the effective thermal conductance, XK, mass XM, and capacitance values CAP1 and CAP2 and returns these values through the named common LD. Subroutine COMPCC then sums the capacitor values in the C(I)s and the mass XMAS. The conductor is then defined in CD(I).

#### 4.6 SUBROUTINE STRUCT

Subroutine STRUCT is the routine that handles the structure files.

STRUCT opens the structure file and either locates a specific structure that already exists in the file or adds an additional structure to the file. Every structure has a corresponding structure number, and a two line description, along with the structure variables.

#### 4.7 SUBROUTINE LOAD

This routine takes data from the named common LD which describes the geometry and materials of the following structure types

RADIATION GAP  
HONEYCOMB  
CORRUGATED  
Z-STANDOFF  
THIN SKIN

and loads it into the named common GAP. In addition, the temperature of the upper and lower surfaces are set for the material property lookups and the radiation conductance. The material identifier numbers MATS(MP,IS,I) are loaded into the M(I) array and the six geometric parameters, XP(MP,IS,J), are loaded into the X(I) array.

#### 4.8 SUBROUTINE ABSUB

Subroutine ABSUB provides the logic to predict ablator-sublimer recession, the node spacing and the effective heat of ablation. This routine is called by main after the sublimation temperature is reached. Ablator variables are passed through the named common SUBLM where the following variables are significant.

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T <sub>SUB</sub>	- Sublimation temperature
X <sub>L</sub>	- Latent heat of sublimation
X <sub>LP</sub>	- Effective heat of sublimation
E <sub>XCHT</sub>	- Excess heat over time step
E <sub>XCHSV</sub>	- Excess heat from last time step
Q <sub>ADVS</sub>	- Advected heat from previous time step
T <sub>MSV</sub>	- Time at last iteration

First, this routine computes the distance of the surface recession,  $\Delta S$ , by the following expression

$$\Delta S = \frac{q_{\text{excess}}}{L' \rho}$$

From this value of  $\Delta S$  the surface node is moved a distance of  $\Delta S$ , the second node is moved  $\Delta S/3.0$  and the boundary between the two nodes is moved  $2 \Delta S/3$ . If the distance between the first and second node,  $XLT\$$ , becomes less than  $XMIN$ , the second node is dropped from the network and nodes, capacitors, locations, and conductors for the rest of the network are renumbered. Figure 4.1 shown below describes some of the nomenclature used in this routine.

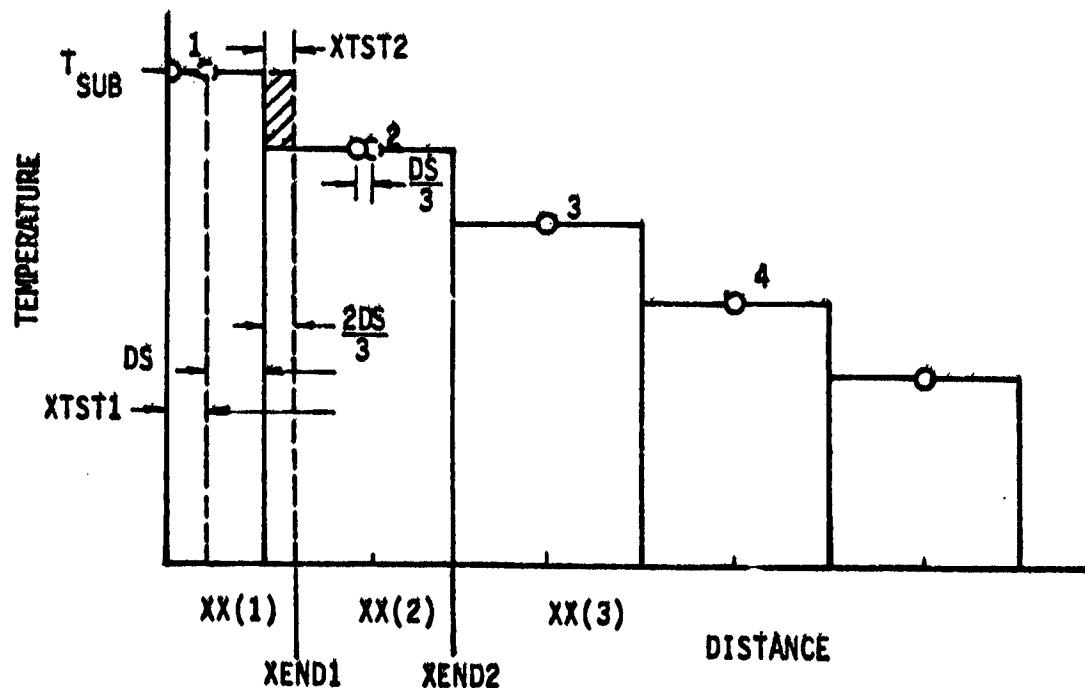


Fig. 4.1 Nomenclature For Ablation Logic

After the nodes and the node boundaries have been moved as shown in this figure, the total energy required to sublime the mass in node one is computed as follows

$$QMEL = \rho \cdot \left( L' \cdot XTST1 + L \cdot XTST2 + C_p \cdot XTST2 (T_{SUB} - T_2) \right)$$

where  $L'$  is the effective heat of sublimation from the previous step and  $L$  is the physical heat of sublimation. The new effective heat of sublimation now is computed from the expression below

$$L' = \frac{QMEL}{\rho \cdot (XTST1 + XTST2)}.$$

The sensible heat added to node one by moving the melt line  $2^{\circ}DS/3$  is the heat advected across the moving boundary and is compensated for by increasing the heat of sublimation to form an effective heat of sublimation. This increase amounts to

$$QADV = \rho C_p XTST2 * (T_{SUB} - T_2)$$

and is shown in the figure as the cross hatched area.

Dropping node two and renumbering the network is accomplished in the last part of the routine, after which subroutine PICTUR is called and a new schematic of the structure is printed on the line printer.

#### 4.9 SUBROUTINE DATA2

Subroutine DATA2 reads data from Unit 7 which defines the environment and was created by LANMIN. Data on the LANMIN output file are shown in Section 5.1. The body point number for the particular point in question, LBP is passed to DATA2 through the argument. The file is then searched for the correct body point and, once found, the following quantities are read from the file and

stored using the following variable names

TIME	TMI (IC)
FILM COEFFICIENT	HC1 (IC)
ADIABATIC WALL ENTHALPY	HAW1 (IC)
PRESSURE	PRES1 (IC)

The largest number of entries in these tables is dimensioned by the common variable NMEN and currently set to 50. If a search of the file does not reveal a match of the body point number, a message is printed, CANNOT FIND BODY POINT.

#### 4.10 SUBROUTINE PICTUR

Subroutine PICTUR displays a description of the structure for a specific Body Point in the form of a picture. PICTUR is called in the EXITS program in two ways. The first way is from INPGEO, right after the structure of a Body Point is defined. This is a quick look picture, that appears on the interactive device, and is used for determining if the structure defined is really the structure desired. If not, an opportunity is allowed to redefine the structure for the Body Point correctly.

The other way that PICTUR is called is from MAIN after the node structure has been defined. This picture is written to the Output file and corresponds to the specific body point that is being executed at that time.

If an ablator-sublimer structure is chosen, then additional calls of PICTUR will occur each time a node is dropped from the structure. For each node that is dropped, a picture will be written to the Output file that describes the structure of the body point after dropping the node. An example of a picture made by PICTUR is shown in Fig. 4.2. It includes a picture representation of the structure of each layer stacked together and also information like the materials used, the structure type and some of the dimensions.

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-THIS IS THE CONFIGURATION FOR BODY PT.

**Fig. 4.2 Line Printer Representation Of Sample Structure From Subroutine PICTUR**

#### 4.11 SUBROUTINE COMTMP

Subroutine COMTHP is the subroutine which calculates the new temperatures  $T(I)$  at the end of the time step given the old temperatures  $T0(I)$ , capacitors  $C(I)$ , and conductors  $CD(I)$  using the heat balance, Section 2.0, at each of the nodes in the structure. Two types of logic are used to compute the temperatures. The first part of the subroutine is devoted to setting up the heat balances and the boundary conditions for the thin skin structure type if a thin skin section is adjacent to the node in question. The second part performs heat balances for all other structural types.

At the beginning of the routine a check on the flag ISBFG is made to determine if thin skin logic is to be used anywhere within the structure. If not, the logic flow goes directly to the standard heat balance for each node. To determine if a node is adjacent to a thin skin element, a check is made on the conductors on either side of the node. If a conductor value is greater than  $10^6$ , then the thin skin heat balance is to be used. (A conductor value of  $10^{10}$  is set in subroutine THINS).

Logic is included to determine if the node is above or below the thin skin

section or if the thin skin section lies on the surface or is the last structural type and requires the adiabatic boundary condition.

The standard heat balance, Section 2.0, is used for all nodes other than nodes adjacent to the thin skin sections. A check is made to see if the node in question is a surface node or if it is the last node. Finally, the temperature of the surface node is checked to see if it has exceeded the sublimation temperature for a ablator-sublimor structure. If this is the case, the excess heat is calculated.

$$\text{EXCHT} = (T(1) - T_{\text{SUB}}) \cdot C_1,$$

the flag NAB set equal to one and the surface temperature set to  $T_{\text{SUB}}$ .

#### 4.12 SUBROUTINE NODE

Subroutine NODE is called from MAIN to set up the model network and to initialize temperatures.

The logic starts by taking one structure type at a time beginning at the surface and working down. A check is made on the structure type, IST, to see if it is a slab or ablator-sublimor. If a slab or ablator-sublimor is found, it is divided into layers and nodes assigned as follows. The layer thickness is controlled by the input parameter DTIM which divides the total thickness into layers to give a stable value of DTIM shown below

$$DX = \sqrt{\frac{DTIM \cdot 2 \cdot k}{\rho C_p}}$$

The number of layers, conductors, are found from

$$NX = \frac{H}{DX} + 1$$

where H is the thickness of the slab or ablator-sublimer. Finally, the length of each conductor in the slab is found from

$$TK = \frac{H}{NX}.$$

At each conductor, the node number at the upper and lower end of the conductor is stored in the L(IC,2) array, the initial temperatures T and TO are set and the node positions XX(IC) are assigned. Finally, at the end of this subroutine, the network information is written out which shows node spacing, structure type, material type, conductor number, and node numbers.

#### 4.13 SUBROUTINE HEATN

This routine linearly interpolates the heating and pressure environment generated by LANMIN and stored in the named common ENVIR. Time (TIME), is the independent argument while the film coefficient (HC), adiabatic wall enthalpy (HAW), pressure (PRES), are returned to MAIN. The counter ISU allows the code to start the interpolation search at the same place in the arrays the last time this routine was called.

#### 4.14 SUBROUTINE VFAC

Subroutine VFAC computes the geometric view factors of NN two dimensional surfaces using the crossed strings method. The named common SF contains the area, AR(I), emissivity, EPP(I), view factors F(I,J), and area view factor products ASF(I,J), of up to ten surfaces which may see each other within an enclosure. Coordinates of the end points of straight line surfaces are contained in the XX and YY arrays in the named common FACT. Two nested DO loops, I and J, cycle through each surface. The area of surface I is found from the subroutine

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DIST which finds the distance between the end points assuming that the surface is a straight line surface.

The view factor using the crossed strings method is shown below, Ref. 9. Given two surfaces shown in Fig. 4.3.

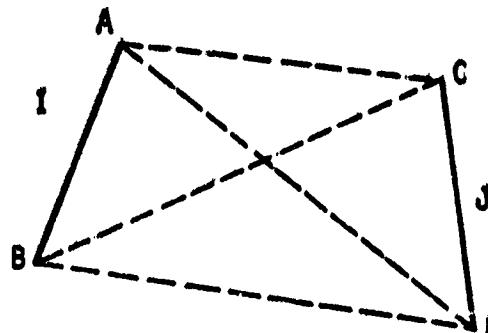


Fig. 4.3 Crossed Strings Nomenclature

$$F_{i-j} = \frac{(\overline{AD} + \overline{BC}) - (\overline{AC} + \overline{BD})}{2 A_i}$$

In other words, the view factor from surface I to J is equal to the lengths of the crossed strings minus the uncrossed strings divided by twice the area of surface I.

Subroutine DIST is called to find the lengths of the crossed and uncrossed strings. The variable SUMF is the sum of the view factors of one surface to all other surfaces which should equal 1 but is not now printed out.

#### 4.15 SUBROUTINE DIST

Subroutine DIST finds the distance between two points given their two dimensional coordinates. The named common FACT contains the coordinates of the end points of the line segments which make up the radiation enclosure. The coordinates are contained in the XX(I,J) and YY(I,J) arrays where the I subscript is the surface number and J is equal to 1 or 2 depending upon which end of the surface is considered. The distance formula

$$D = \sqrt{(X_1 - X_2)^2 + (Y_1 - Y_2)^2}$$

is used to compute the distance.

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#### 4.16 SUBROUTINE TMSTEP

This subroutine determines the stable time step for the explicit time integration of the energy balance at each node. For the general case the maximum stable time step for the nodal network shown in Figure 4.4 is

$$\Delta\theta \leq \frac{C(I)}{CD(J) + CD(J-1)} \quad | \text{MIN.}$$

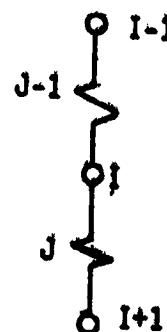


Fig. 4.4 Nomenclature For Slab Time Step Calculation

For the case where the node lies on the surface, the conductor  $CD(J-1)$  is replaced by  $CONV + CRAD$ , the sum of the convective and radiative conductors. For the adiabatic backwall, the conductor  $CD(J)$  is set to zero.

The thin skin stability requirement for the configuration shown in Figure 4.5

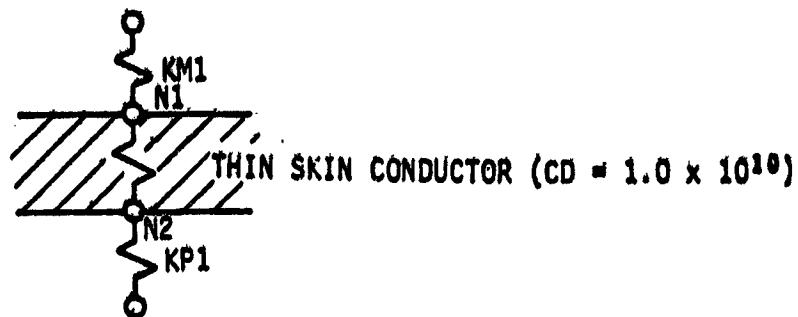


Fig. 4.5 Nomenclature For Thin Skin Time Step Calculation

is found by ignoring the very high conductor value of the thin skin section. In general, the stability requirement for node N1 or N2 is

$$\Delta\theta \leq \frac{C(N1) + C(N2)}{CD(KM1) + CD(KP1)} \quad | \text{ MIN}$$

For a thin skin surface node, the convection and radiation conductors are included in  $CD(KM1)$ . For the adiabatic backwall  $CD(KP1) = 0$ .

After the minimum  $\Delta\theta$  is found, the resulting time step is divided by the input parameter STAB to insure stability.

#### 4.17 SUBROUTINE INPGEO

Subroutine INPGEO is the interactive routine that sets up the initial conditions and defines the structure for each Body Point to be run, asks for the FILE NAME of the file that contains the LANMIN (MINIVER) data for each body point, and asks for the FILE NAME of the file that contains all previously defined structures. (NOTE: Use of this file is optional). This subroutine also

asks for the FILE NAME of the file that is to contain the EXITS output.

The initial, final and delta-times for the output print are also set here. The control parameters may also be changed at this point if the user desires. Otherwise, default values are used.

The number of body points to be run is then defined and the conditions for each body point are defined. For each body point, an initial and sink temperature is defined as well as the structure of that body point. The structure for a body point may be chosen from the structure file by structure number or by creating a new structure definition.

Creating a new structure is done by layers using structure types, material numbers, and dimensions. After the structure of a body point is defined, a simple picture of that structure is displayed and the structure of the next body point is defined. After the structure and initial condition for all the body points are defined, INPGBO returns to MAIN.

#### 4.18 SUBROUTINE SUBPR

Subroutine SUBPR interpolates in an array of data to find the temperature and latent heat of sublimation as a function of surface static pressure as supplied by LANMIN. These properties are stored in the CCS (N, NMB9) array found in the named common CSUB. The data is stored in the following manner

CCS (N,1) = Number of X-Y pairs in array  
CCS (N,2) = First independent variable (Pressure)  
CCS (N,3) = First dependent variable  
CCS (N,4) = Second independent variable  
CCS (N,5) = Second dependent variable  
etc.  
↓

The routine first checks to see if the value of the argument, X, is out of range

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of the independent variables in the table, and if so, assigns the correct subscripts to extrapolate off the end of the table. If the value is not out of range, a search is conducted to find the two independent variables which bound the value and a simple straight line interpolation is performed.

#### 4.19 SUBROUTINE SRIPP

Subroutine SRIPP finds the radiation interchange factor,  $\bar{F}$ , in an enclosure given the areas, emissivities and the geometric view factors found in VFAC. The named common SF contains the information in AR(I) (areas), EPP(I) (emissivity), and F(I,J) (geometric view factors). The product of the area and radiant interchange factor is stored in ASF(I,J).

The method used in SRIPP is a network method which is solved by an iterative technique for the radiosity between each of the surfaces. If we consider the network in Figure 4.6 for an enclosure with three surfaces

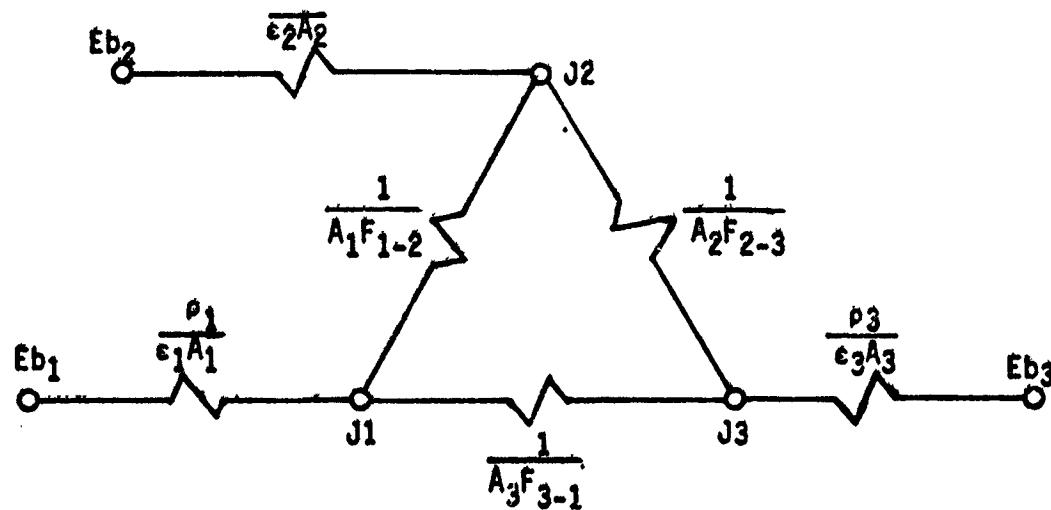


Fig. 4.6 Network for Typical Three Surface Enclosure

we see that

$$q_{NET j} = \sum_{k=1}^n F_{j k} A_j (J_k - J_j)$$

or

$$q_{NET-j} = \frac{\epsilon_j A_j}{\rho_j} (J_j - E_{bj}) .$$

Equating these two expressions and by use of some algebra, we see that

$$J_j = \left[ \frac{\epsilon_j}{1 - \rho_j F_{jj}} \right] E_{bj} + \frac{\rho_j}{1 - \rho_j F_{jj}} \sum_{k=1, k \neq j}^n J_k F_{ik}$$

is the final expression for the radiosity at node j. Using the following iterative relaxation procedure

$$J_j^{n+1} = (1 - \beta) J_j^n + \beta J_j ,$$

where the relaxation parameter,  $\beta$ , is typically .5, convergence,

$$\left| \frac{J_j^n - J_j^{n-1}}{J_j^n} \right| \leq .001$$

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is found within ten iterations for the structures described in this code. After the radiosities are found, the radiant interchange area factors are found from

$$ASF(i,j) = \sum_{i \neq j} A_i = \frac{A_i F_{i-j} (J_i - J_j)}{(E_{bi} - E_{bj})}$$

where the black body emissive power  $E_b$  is assigned arbitrarily.

#### 4.20 SUBROUTINE CORC

This routine determines the effective thermal conductivity, thermal capacity and mass of a corrugated panel section considering heat transfer by conduction and radiation through the panel. We first consider a small section of the corrugated panel shown in Figure 4.7

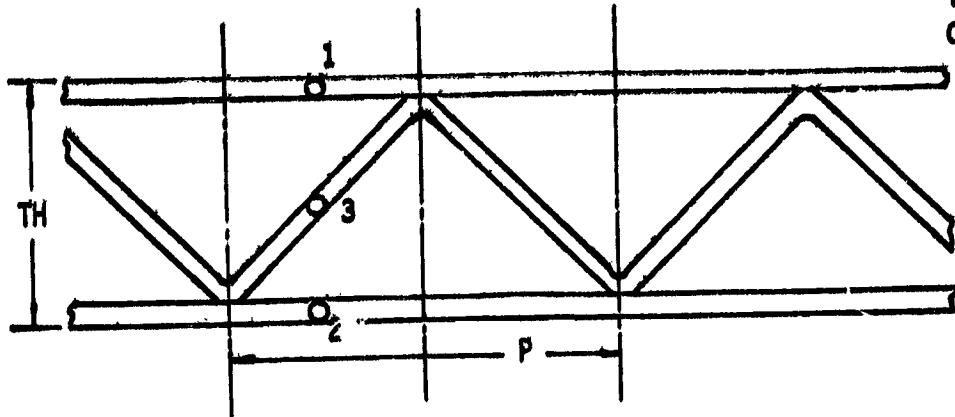


Fig. 4.7 Corrugated Panel Configuration

and choose two planes of symmetry and assign the three nodes 1, 2, and 3. The geometric and material parameters are contained in the named common GAP. The three material thicknesses are  $TH_1$ ,  $TH_2$ ,  $TH_3$ , while material identifiers are contained in  $M_1$ ,  $M_2$ ,  $M_3$ . Overall height is  $TH$  and the pitch is  $P$ . Temperatures at 1 and 2 are given as  $T_1$  and  $T_2$ . These parameters are assigned their respective variable names in subroutine LOAD which is called immediately before CORG is called. The temperature at 3 is unknown, but given the temperatures at 1 and 2, geometric and thermophysical properties, the temperature at 3 can be solved by iteration. The equivalent network for this system consists of a conduction path from node 1 to node 2 passing through node 3. In addition, there is radiative heat transfer from node 1 to 3 and reradiative heat transfer from 3 to 1. The radiative enclosure for the upper, lower, and corrugated structure is modeled using three planes shown below

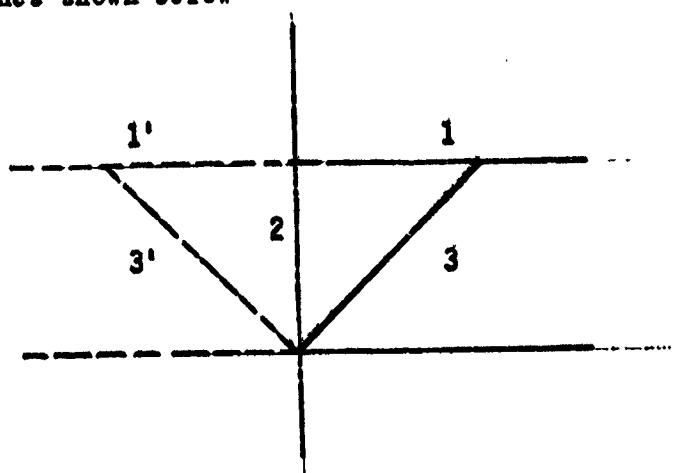


Fig. 4.8 Radiation Enclosure for Corrugated Panel

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Plane 2 is a plane of symmetry in which the emissivity is zero and all incident radiation is reflected back into the enclosure which is the emitted and reflected radiation from the reflected enclosure 1', 3', 2. The coordinates of the planes 1, 2, 3 are set and stored in the XX (I,J) and YY (I,J) arrays where I is the plane number and J is 1 or 2 representing the end points. The subroutines VFAC and SRIPF are called to define geometric view factors and area-radiative interchange factors, ASF (I,J). From these factor radiation conductors are formed from the following expression

$$K_{ij} = A_i \sigma F_{ij} \circ (T_i^2 + T_j^2) (T_i + T_j)$$

where  $\sigma$  = Stefan Boltzman Constant

The equivalent network for the total heat transfer from surface 1 to 2 is shown in Figure 4.9.

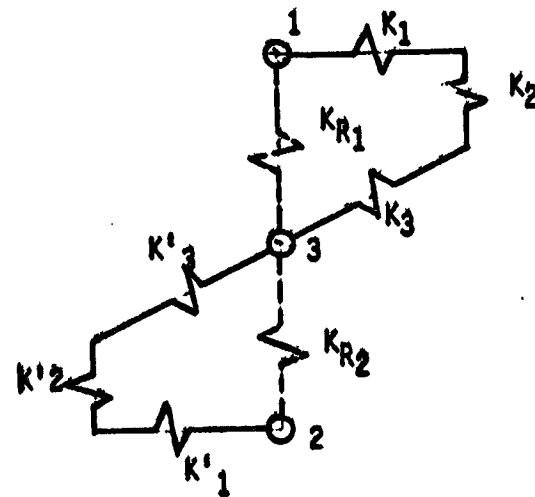


Fig. 4.9 Corrugated Panel Equivalent Network

The three conductors in series  $K_1$ ,  $K_3$ ,  $K_2$  and  $K'_1$ ,  $K'_3$ ,  $K''_2$  represent the paths

through the upper and lower surfaces, a contact conductance which for the present time is set to  $10^6$  times the area, and the conduction path through the corrugated section. The expression for the three conductors in series is used to form the conductors from node 1 to 3 and 2 to 3 as shown below

$$C_1 = \frac{K_1 K_2 K_3}{K_1 K_2 + K_1 K_3 + K_2 K_3}$$

and

$$C_2 = \frac{K'_1 K'_2 K'_3}{K'_1 K'_2 + K'_1 K'_3 + K'_2 K'_3}$$

If we represent the radiation paths from 1 to 3 as

$$C_4 = A_1 \sigma_{1-3} (T_1^2 + T_3^2) (T_1 + T_3)$$

and 2 to 3 as

$$C_5 = A_1 \sigma_{1-3} (T_2^2 + T_3^2) (T_2 + T_3)$$

We can iterate on the temperature at 3 using the following expression.

$$T_3^{n+1} = (1 - \beta) T_3^n + \beta \cdot \left( \frac{T_1 C_1 + T_2 C_2 + T_1 C_4 + T_2 C_5}{C_1 + C_2 + C_4 + C_5} \right)$$

Convergence is obtained when

$$\frac{T_3^{n+1} - T_3^n}{T_3^{n+1}} < \epsilon \sim .001$$

where  $\epsilon$  is the input parameter TOL set to a default value of .001.

When the temperature  $T_s$  is solved for the total heat transfer per unit area of panel is computed

$$Q = \frac{|T_1 - T_s| \cdot (C1 + C4)}{P2}$$

and the equivalent conductivity is found from

$$XK = \frac{Q}{T_1 - T_2}.$$

An example case for an aluminum corrugated panel and the equivalent thermal conductance is shown in Figure 4.10.

The thermal capacitance is computed from the mass of the structure and split in two equal parts assigned to CAP1 and CAP2. Total mass is found and stored in XM.

#### 4.21 SUBROUTINE HONEY

Subroutine HONEY determines the effective thermal conductivity, capacity and mass of a honeycomb core sandwiched between two layers. The geometric definition and material identifiers are contained in the named common GAP. Temperatures of the outer layers  $T_1$  and  $T_2$  are also found in GAP. Cell dimensions are given by TH, the overall height, and H the distance from one flat side to the other for a hexagonal cell. The distance D is the pitch distance shown in Figure 4.11.

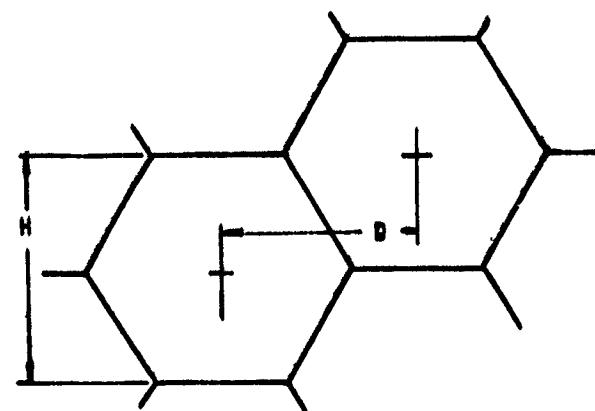


Fig. 4.11 Honeycomb Cell Dimensions

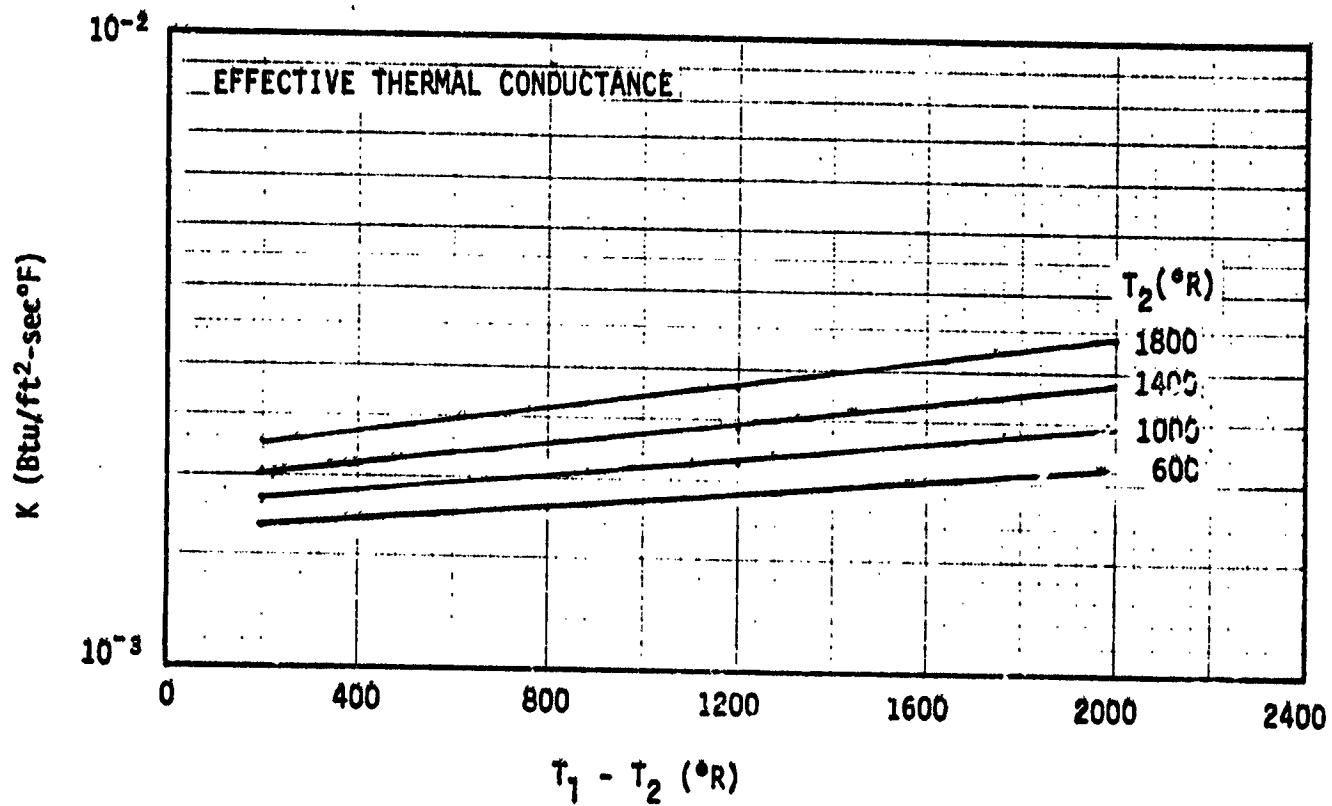
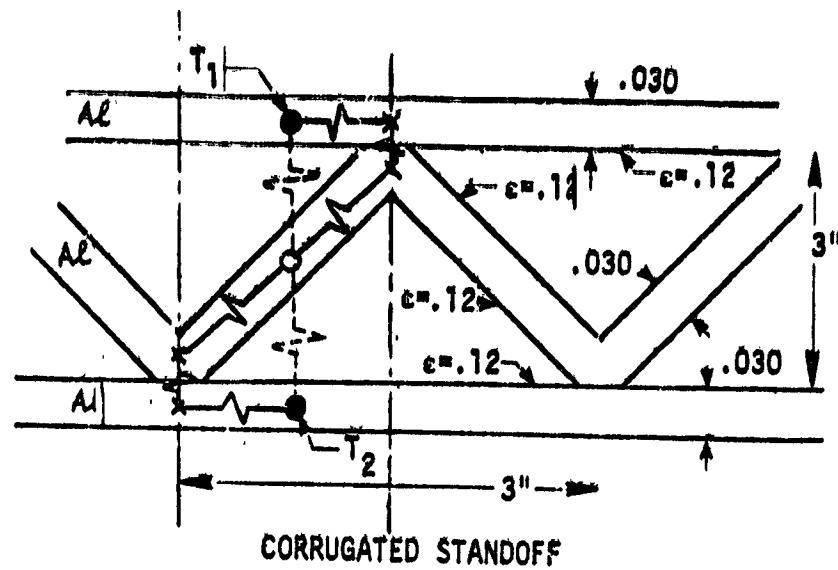


Fig. 4.10 Corrugated Structure Effective Thermal Conductance

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The number of cells per unit area, the number of cell walls, and the volume of the material making up the structure are computed.

Heat transfer through the honeycomb is assumed to be by conduction through the core and radiation within each cell. Each cell is assumed to have six equal walls which is shown in Figure 4.12.

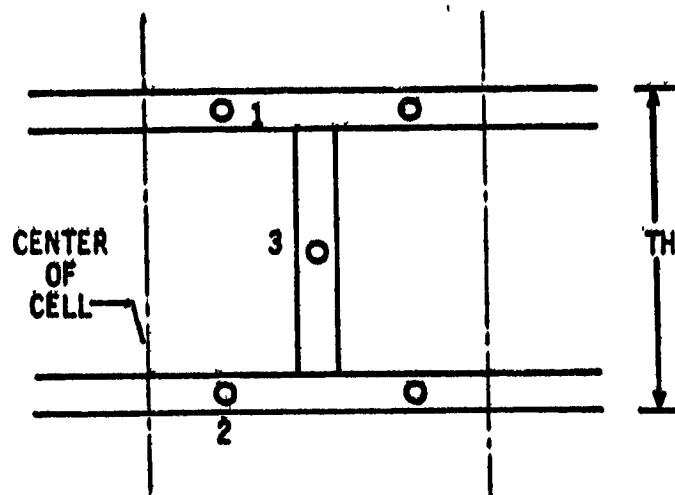
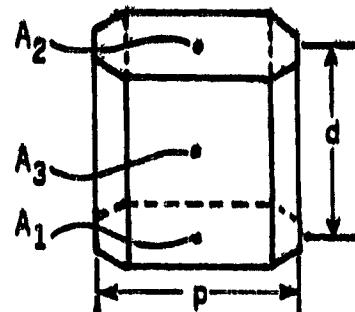


Fig. 4.12 Honeycomb Cell Model

Temperatures at 1 and 2,  $T_1$  and  $T_2$  respectively, are given. Temperature at 3 is solved by relaxation. Radiation from nodes 1 to 2, 1 to 3 and 3 to 2 is computed by assuming a view factor of .1 from surface 1 to 2 and .9 from 1 to 3. This is done in lieu of using the crossed string method (subroutine VFAC) since this is a three dimensional configuration. Changes in the view factors can be made easily to reflect cell size and honeycomb thickness. Typical view factors as a function of cell size and honeycomb thickness are provided in Table 4.4.

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### VIEW FACTORS

$$F_{1-2} \text{ (Base To Top)} \\ F_{1-3} \text{ (Base To Sides)}$$

$\frac{p}{d}$	.2	.3	.4	.5	.6	.8	1.0
.25	.130 .870	.230 .770	.300 .700	.380 .620	.450 .550	.540 .460	.610 .390
.50	.035 .965	.075 .925	.125 .875	.175 .825	.220 .780	.300 .700	.380 .620
1.0	.025 .975	.035 .965	.040 .960	.060 .940	.075 .925	.125 .875	.170 .830
2.0	.012 .988	.020 .980	.025 .975	.030 .970	.035 .965	.040 .960	.060 .940
3.0	.008 .992	.010 .990	.017 .983	.023 .975	.027 .973	.028 .972	.030 .970
4.0	.006 .994	.012 .908	.016 .984	.020 .980	.021 .979	.023 .977	.025 .975

Table 4.4 View Factors From Top And Sides To Bottom Of Honeycomb Cell For Use In Subroutine HONEY

The equivalent electrical network is shown in Figure 4.13

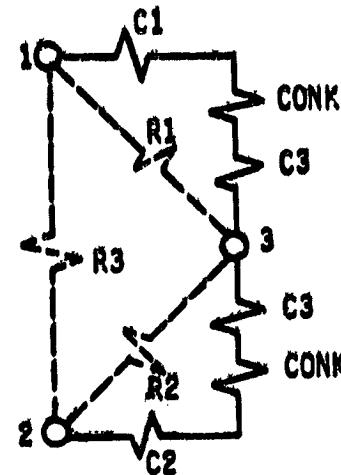


Fig. 4.13      Equivalent Network For Honeycomb

This network describes the heat transfer path between the upper and lower surfaces. Equivalent conductors for the conduction paths are determined from

$$X_{C1} = \frac{C1 \cdot \text{CONK} \cdot C3}{C1 \cdot \text{CONK} + C3 \cdot \text{CONK} + C1 \cdot C3}$$

and

$$X_{C2} = \frac{C2 \cdot \text{CONK} \cdot C3}{C2 \cdot \text{CONK} + C3 \cdot \text{CONK} + C2 \cdot C3}$$

where CONK is the contact conductance between the core and the outer surfaces, currently set to  $10^6$  BTU/Ft<sup>2</sup>-Sec<sup>-0.5</sup>. R<sub>1</sub>, R<sub>2</sub>, and R<sub>3</sub> are computed using the three temperatures and the view factors. Finally, the temperature at 3 is found by relaxation using the formula

$$T_3^{n+1} = (1 - \beta) T_3^n + \beta \frac{(T_1 \cdot (X_{C1} + R_1) + T_2 \cdot (X_{C2} + R_2))}{X_{C1} + R_1 + X_{C2} + R_2}$$

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In the expression above,  $\beta$  is the relaxation factor usually set to .5 in the input. Convergence is rapid and occurs when

$$\frac{T_g^{n+1} - T_g^n}{T_g^n} < \text{TOL} \sim .001 .$$

Total heat transfer is computed from the conductor values and three known temperatures. Equivalent conductance is then found by dividing by the temperature difference  $T_i - T_s$ . An example case is shown in Figure 4.14 for an all aluminum honeycomb structure. Capacitance CAP1 and CAP2 is found in addition to the mass of the structure and stored in XM.

#### 4.22 SUBROUTINE RGAP

Subroutine RGAP computes the equivalent conductor value through a radiation gap. This model also includes the thermal conductance of the upper and lower surfaces. The thermal model of the radiation gap used is shown in Figure 4.15.

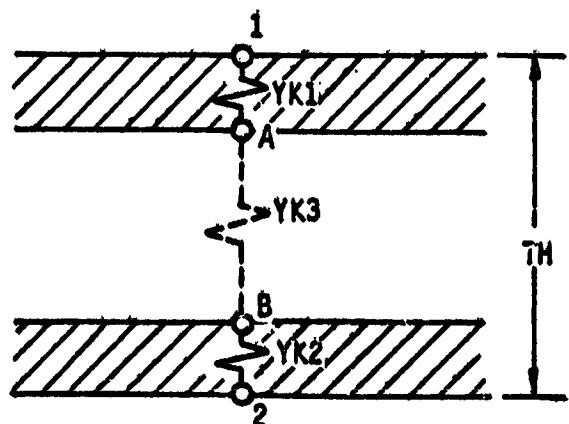


Fig. 4.15 Network For Radiation Gap Calculation

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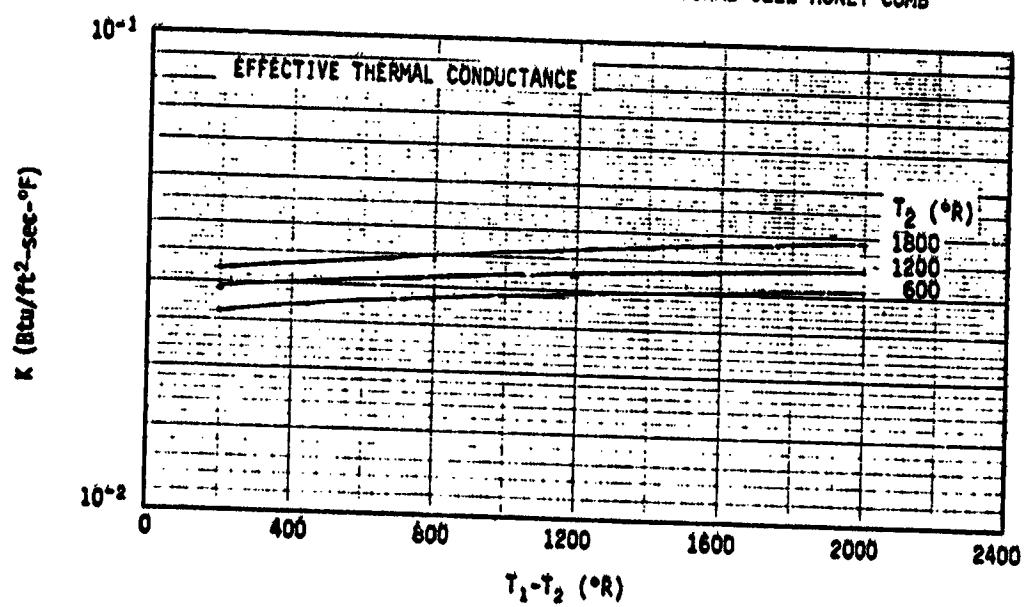
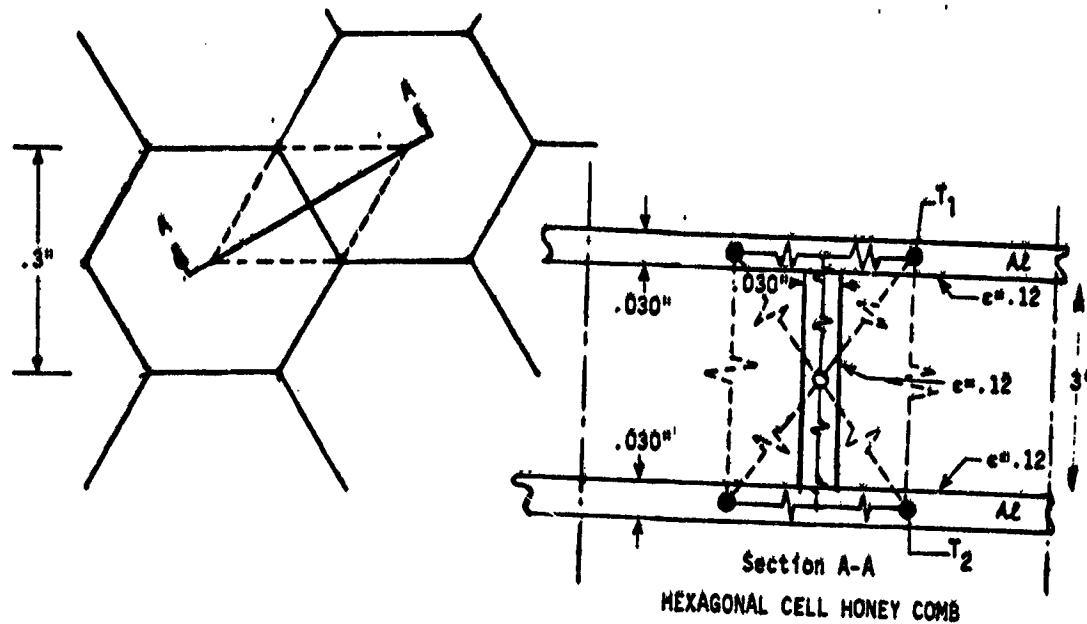


Fig. 4.14 Honeycomb Effective Thermal Conductance

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The temperatures at A and B are computed from the known temperatures at 1 and 2. All geometric and material parameters are passed to RGAP through the named common GAP. The radiant interchange factor between the two surfaces is assumed to be that of two infinite plates and is found from the expression below

$$\sigma_{A-B} = \frac{1}{\frac{1}{\epsilon_1} + \frac{1}{\epsilon_2} - 1}$$

An iteration procedure is used to find the temperatures at A and B given the temperatures at 1 and 2, T1 and T2 respectively. The conductors YK1 and YK2 are found from the conductivity and the thicknesses TH1 and TH2. YK3 is a radiation conductor found from the following expression:

$$YK3 = \sigma_{A-B} (T_A^2 + T_B^2) (T_A + T_B)$$

The relaxation algorithm used to find TA and TB is as follows

$$T_A^{n+1} = (1 - \beta) T_A^n + \beta \frac{T_1 \cdot YK1 + T_B^n \cdot YK3^n}{YK1 + YK3}$$

and

$$T_B^{n+1} = (1 - \beta) T_B^n + \beta \frac{T_2 \cdot YK2 + T_A^n \cdot YK3^n}{YK2 + YK3}$$

Convergence is found after

$$\frac{T_A^{n+1} - T_A^n}{T_A^n} \text{ AND } \frac{T_B^{n+1} - T_B^n}{T_B^n} < \epsilon ".001$$

Equivalent thermal conductivity is found once  $T_A$  and  $T_B$  are solved by

$$XK = \frac{YK_1 + YK_2 + YK_3}{YK_1 + YK_2 + YK_1 + YK_3 + YK_2 + YK_3}$$

An example case is shown in Figure 4.16.

Finally the mass,  $\bar{M}$ , and capacitance,  $CAP_1$  and  $CAP_2$ , are computed.

#### 4.23 SUBROUTINE STAND

This subroutine computes the equivalent thermal conductance, capacitance and mass of a structure consisting of a standoff section and two outer surfaces. Heat transfer is assumed to be by conduction and radiation through the panel. Consider the small section of the Z-standoff panel shown in Figure 4.17.

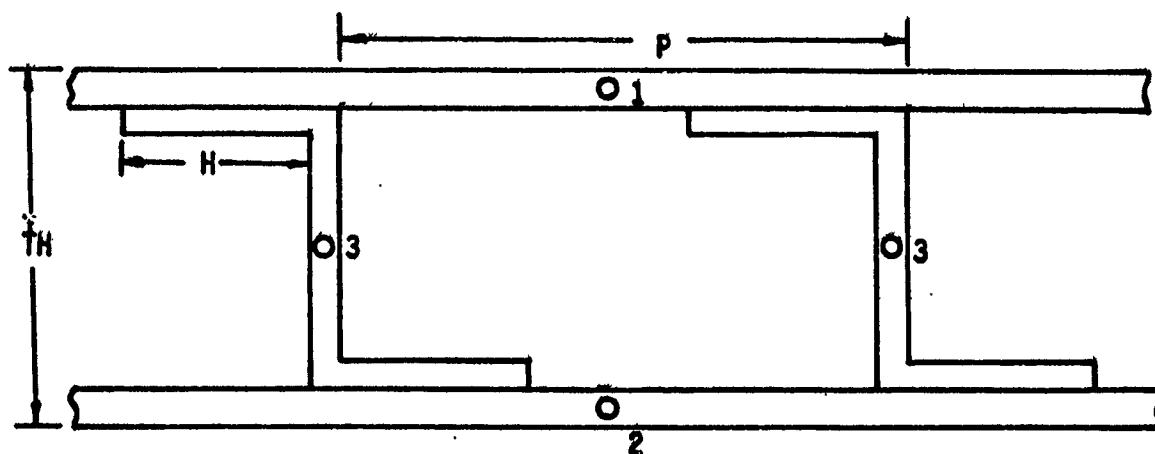


Fig. 4.17 Z-Standoff Configuration

We choose a single enclosure bounded by two standoffs' mid-plane and assign three nodes to the four surfaces, node 3 being common to the standoffs. The geometric and material parameters are contained in the named common, GAP. The three material thicknesses are  $TH_1$ ,  $TH_2$ , and  $TH_3$  while the material identifiers are  $M_1$ ,  $M_2$ , and  $M_3$ . Overall height is  $TH$ , the pitch is  $P$  and the flange width

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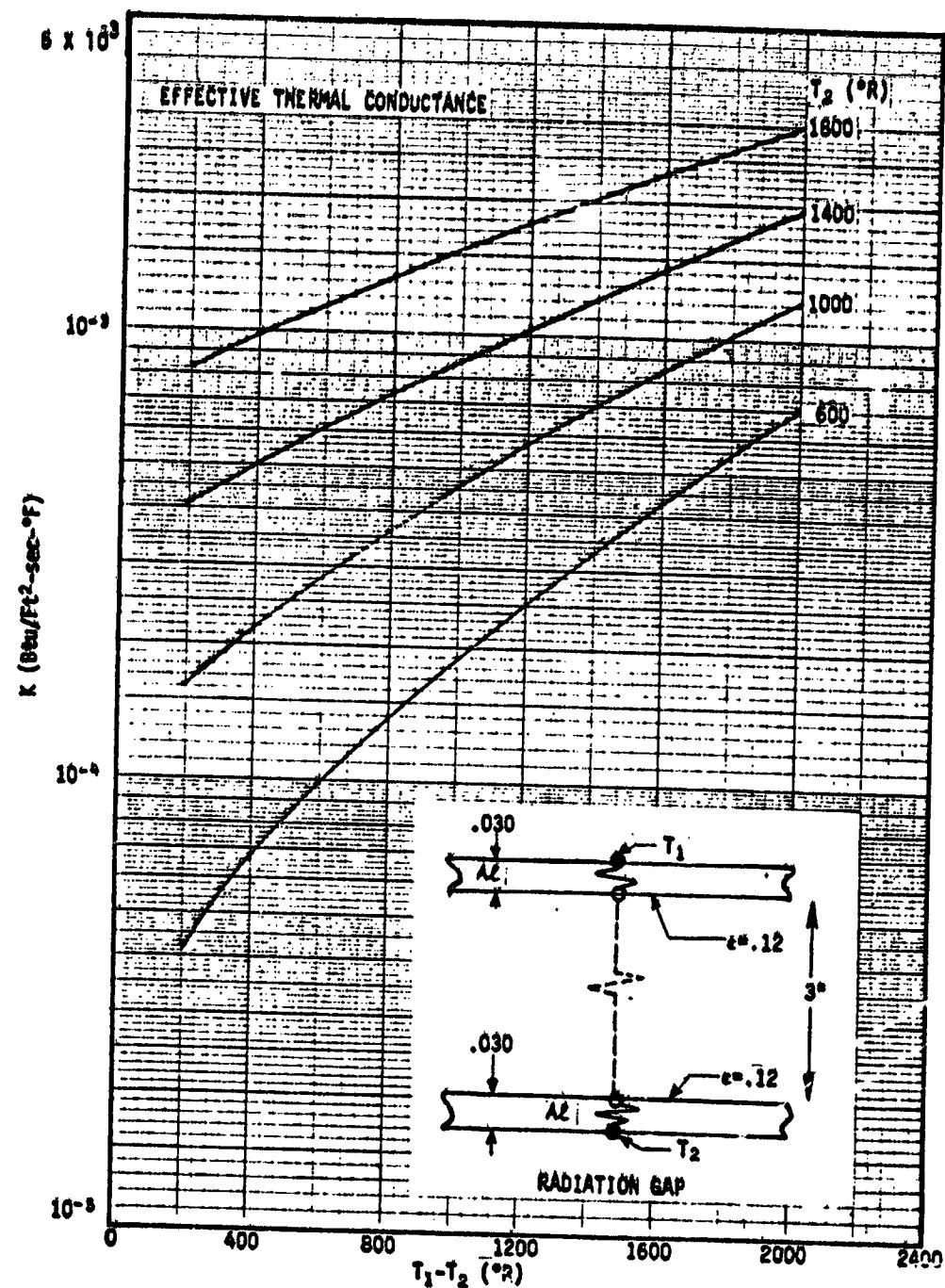


Fig. 4.16 Radiation Gap Effective Thermal Conductance

is H. Temperatures at 1 and 2 are  $T_1$  and  $T_2$ . These parameters are assigned their respective variable names in subroutine LOAD which is called immediately before STAND is called. Temperature at 3 is unknown, but given the temperatures at 1 and 2, geometric and property data, the temperature at 3 can be solved for by iteration. The heat transfer paths are conduction from the upper and lower surfaces through each of the standoffs, since the model is asymmetrical about a midplane, and radiation from node 1 to 2, and from 1 to 3 to 2. The radiative enclosure for the upper lower surfaces, and standoffs consists of four planes shown in Figure 4.18.

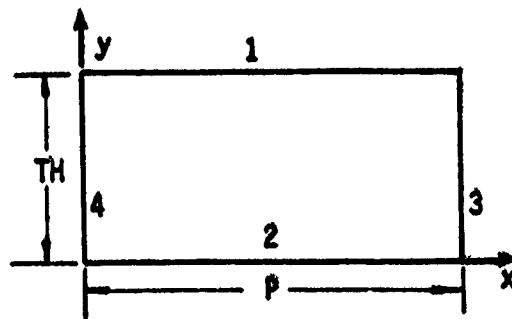


Fig. 4.18 Radiation Enclosure Model For Z-Standoff

The coordinates of the planes making up the enclosure are computed and stored in the XX(I,J) and YY(I,J) arrays where I is the plane number and J is 1 or 2 representing the end points. The subroutines VFAC and SRIPF are called to define geometric view factors and area-interchange factors ASF(I,J).

From these factors, radiation conductors are formed from the following expression

$$K_{ij} = A_i \sigma F_{ij} \alpha (T_i^2 + T_j^2) (T_i + T_j)$$

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The equivalent network for the total heat transfer from surface 1 to 2 is shown in Figure 4.19.

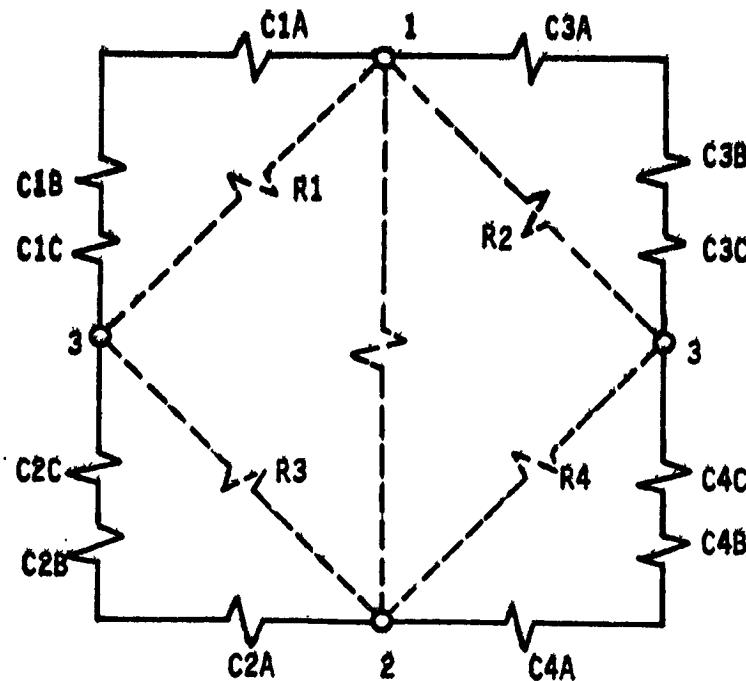


Fig. 4.19      Equivalent Network For Z-Standoff

Equivalent series conductors are formed,  $C_1$ ,  $C_2$ ,  $C_3$  and  $C_4$  which include a contact conductance and the conductance of the plate and standoffs. Radiation conductors  $R_1$ ,  $R_2$ ,  $R_3$ ,  $R_4$ , and  $R_5$  complete the network. The expression for the equivalent conductor  $C_1$  is

$$C_1 = \frac{C_{1A} \cdot C_{1B} \cdot C_{1C}}{C_{1A} \cdot C_{1B} + C_{1B} \cdot C_{1C} + C_{1A} \cdot C_{1C}}$$

Similar expressions are used for  $C_2$ ,  $C_3$  and  $C_4$ . The temperature at 3 is found by relaxation using the following expression

$$T_3^{n+1} = (1 - \beta) T_3^n + \beta \left( \frac{T_1 (R_1 + R_2 + C_1 + C_3) + T_2 (C_2 + R_3 + C_4 + R_4)}{R_1 + R_2 + R_3 + R_4 + C_1 + C_2 + C_3 + C_4} \right)$$

Convergence is obtained when

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$$\frac{T_g^{n+1} - T_g^n}{T_g^{n+1}} < \epsilon \sim .001$$

where  $\epsilon$  is the input parameter TOL set to a default value of .001.

The total heat transfer per unit of panel is then found from

$$Q = \frac{(T_1 - T_2) \cdot R_5 + (T_1 - T_3) \cdot (C_2 + C_4 + R_3 + R_4)}{P}$$

and the equivalent thermal conductance is found from

$$XK = \frac{Q}{T_1 - T_2}$$

An example of the equivalent thermal conductance calculation is shown in Figure 4.20.

The thermal capacitance is found and stored in CAP1 and CAP2. Total mass is found and stored in XM.

#### 4.24 SUBROUTINE THINS

Subroutine THINS computes the capacitance and mass of a material with infinite thermal conductance. CAP1 and CAP2 each contain one half the total thermal capacitance of the plate and XM contains the mass. The equivalent thermal conductance is set to  $10^{10}$ , while never used in computing temperatures, it is used as a flag to indicate presence of infinitely conducting plate.

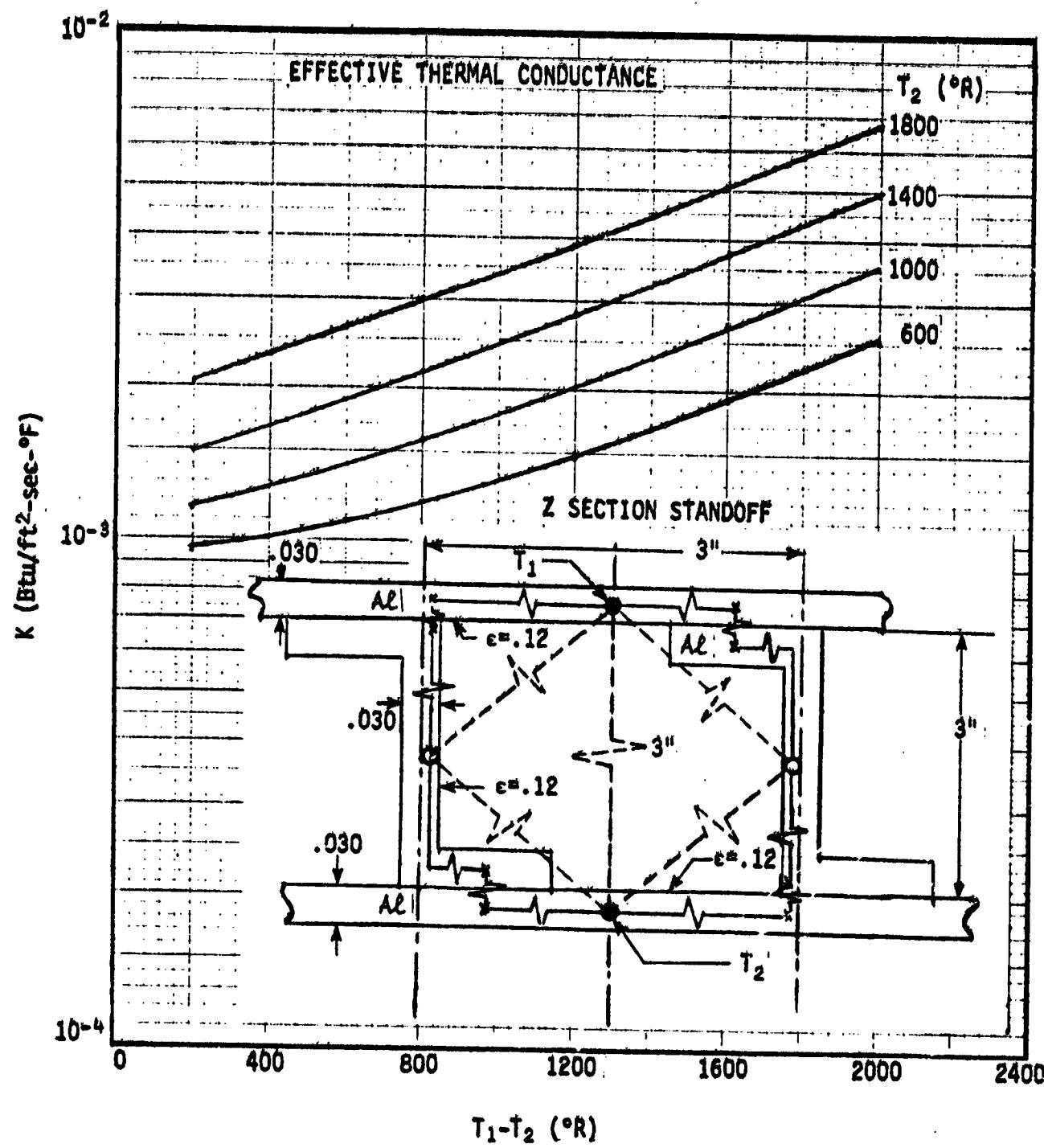
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Fig. 4.20 Z-Standoff Effective Thermal Conductance

## Section 5.0

### INPUT

Data required for operation of EXITS comes from several sources. Data for the particular case, control parameters, geometry etc. comes from the interactive input. Material properties are on a property file and identified with a material number. The environment comes from a file created by LANMIN which is compatible with the EXITS input format. If the structural configuration has been modeled previously and saved on the structure file, the user can skip the structural modeling questions during the interactive input, and call in data from the structure file to describe the detail of the thermal protection system being investigated. A user may wish to study the effects of changing certain trajectory parameters, in such case he would create several LANMIN input files. He would then run the EXITS code at each body point being investigated saving the structure by assigning it a structure number and saving it on a structure file. The input for the subsequent trajectory cases would be greatly simplified since the geometry and materials have been defined and stored on a structures file. Several of these structures files may be created, each file defining the thermal protection system at selected body point locations on a particular vehicle. With these data defined, one can easily compare for thermal protection systems candidate vehicles'.

The following discussion presents examples and descriptions of the input data required for input in the EXITS code. First we have the data defining the environment generated and stored by LANMIN. Next the material properties file, which presently contains some twenty eight materials and can be added to or edited as the user chooses, is presented. Thirdly, an example of the file generated by EXITS which saves the thermal protection system structural and geo-

tric definition and is called the structures file.

Two examples of the interactive input which demonstrates use of all of the options and the seven structure types now available in EXITS are presented. A description of the output for these cases is given in Section VI.

### 5.1 LANMIN GENERATED ENVIRONMENT FILE

The environment for the body point under investigation is generated by the LANMIN code and stored as a data file. The EXITS code reads this data from Unit 7 in subroutine DATA2 and stores it in arrays. A body point description and body point number is read from this file by the following statement

```
READ(7,700,END=1000)DESCRP,IBP  
700 FORMAT(A72,IS)
```

until the body point specified in the interactive input is found. The environment is then read immediately following this record by the read statement shown below

```
READ(7,701)TM1(IC),HC1(IC),HAW1(IC),PRES1(IC)  
701 FORMAT(2X,FG. 1,39X,E10.3,2X,E10.3,3GX,E10.3).
```

An example of the LANMIN input file is shown in Table 5.1. As can be seen, only the time, enthalpy based heat transfer coefficient, adiabatic wall enthalpy, and pressure is read. Data may be read in either the English units or Metric units shown in Table 5.2. Trajectory points are read until a negative time point is encountered.

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**SFS-1 REENTRY TRAJ. (FORSTER) VANG. REF.**

# TIME	ALT SEC	VEL FT/SEC	PITCH NO. ATTACK	PERIODS HO./FT	MEAT COEFF REC ENTHALPY PAD EQUIL			HEAT RATE BTU/SFT-S	HEAT RATE BTU/SFT-S	PRESSURE LB/SFT	FLOW TYPE	
					LBS/SF	ST	DESF					
0.0	396.3	26565.1	19.24	*1.13	*1.13	0.113-005	198.9	*620-001	-2038-001	*125-101	WAVI	
25.0	384.0	26580.4	20.54	*1.26	*1.26	0.113-005	251.1	*931-001	*462-001	*203-101	WAVI	
50.0	371.5	26595.5	21.85	*1.25	*0.28-301	0.112-005	213.5	*111-000	*112-001	*355-001	DART	
75.0	359.1	26610.3	23.34	*1.23	*1.64-002	0.112-005	185.2	*168-000	*128-002	*617-001	RARI	
100.0	346.7	26625.4	24.21	*1.03	*310-002	0.112-005	162.6	*269-000	*192-002	*189-001	DART	
125.0	334.5	26637.6	25.23	*1.63	*614-002	0.112-005	158.2	*390-000	*209-002	*199-000	DART	
150.0	322.5	26656.1	26.16	*1.54	*124-003	0.112-005	155.4	*576-000	*433-002	*378-000	DART	
175.0	310.5	26656.9	26.67	*1.63	*51-003	0.112-005	171.5	*86-0-000	*66-0-000	*731-000	DART	
200.0	294.5	26658.5	27.62	*1.26	*208-003	0.1-0-003	168.5	*129-001	*971-002	*108-001	DART	
225.0	281.1	26657.6	27.89	*1.89	*980-003	0.179-003	174.5	*193-001	*145-003	*280-001	DART	
250.0	277.9	26731.7	27.86	*1.52	*17-004	0.112-005	1180.6	*271-001	*213-003	*471-001	DART	
275.0	268.3	26595.5	27.82	39.53	*98-004	0.253-003	1178.0	*269-001	*260-003	*775-001	AIR	
300.0	263.4	26516.6	27.00	*1.26	*44-0-004	0.301-003	1111-005	124.6-5	*319-001	*360-003	*113-002	AIR
325.0	255.0	26618.9	26.45	*1.93	*511-004	0.345-003	1104-005	134.2	*363-001	*451-001	*151-002	AIR
350.0	251.6	26627.6	26.07	*1.50	*586-004	0.371-003	1092-005	1327.8	*387-001	*588-003	*178-002	AIR
375.0	249.4	26623.1	25.76	*1.16	*685-004	0.384-003	1088-005	1341.2	*399-001	*687-003	*194-002	AIR
400.0	247.4	23956.1	45.46	*1.16	*695-004	0.397-003	1086-005	1344.3	*402-001	*748-003	*206-002	AIR
425.0	245.5	23786.4	45.21	39.66	*726-004	*848-003	1348.5	*402-001	*848-003	*496-002	AIR	
450.0	244.6	23124.5	24.96	*764-004	*911-003	*1C3-005	1348.6	*402-001	*949-003	*212-002	AIR	
475.0	233.8	23428.4	29.70	39.89	*795-004	*819-003	101-005	1346.0	*403-001	*135-004	*222-002	AIR
500.0	230.3	23424.0	29.49	39.60	*827-004	*846-003	*997-004	1346.5	*403-001	*115-004	*231-002	AIR
515.0	228.6	23329.6	24.29	39.84	*857-004	*849-003	*981-004	1345.3	*402-001	*121-004	*235-002	AIR
540.0	226.1	23318.6	28.14	30.27	*867-004	*852-003	*977-004	1344.0	*401-001	*126-004	*238-002	AIR
562.0	220.0	22977.4	23.98	39.11	*887-004	*836-003	*967-004	1342.6	*400-001	*132-004	*241-002	AIR

**Table 5.1 Example Of LAMIN Generated Environment For Body Point 12**

► Headers are written for LAMIN printed output but are omitted in file generated for EXITS input.

QUANTITY	ENGLISH	METRIC
TIME	SECONDS	SECONDS
FILM COEFFICIENT	$\frac{\text{LBM.}}{\text{FT - SEC}}$	$\frac{\text{Kg.}}{\text{M - SEC.}}$
ADIABATIC WALL ENTHALPY	$\frac{\text{BTU}}{\text{LBM.}}$	$\frac{\text{JOULES}}{\text{Kg.}}$
PRESSURE	$\frac{\text{LBF.}}{\text{FT.}}$	$\frac{\text{NEWTONS}}{\text{M}^2}$

Table 5.2 LANMIN Generated Environment Units

### 5.2 MATERIAL PROPERTIES FILE

The thermophysical properties file is read by subroutine DATA1 which finds the specified material identification number, Table 5.3, and then reads the four or six property tables for a non-ablator or ablator respectively. The present set of property data are contained in the file INP1.FT and read by Unit 8 and are shown in Table 5.4. Property data are usually a function of temperature only, however, the option exists for density, specific heat, thermal conductivity, and emissivity to be a function of both temperature and pressure. Two additional properties, sublimation temperature, and heat of ablation are added for the ablation materials. Both of these properties are input as a function of pressure.

Referring to Table 5.4, we see that the present property file contains twenty eight materials used in thermal protection system design. Material identification numbers are given as the first entry of the header card for the density table. The header card to each table is read by the following statement

```

READ(8,701)KD,JD,TEST1,TEST2,TNPMXA
701 FORMAT(1S,2X,1S,4X,A10,1X,A13,E10.0)

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MATERIAL LIST

- |     |                              |
|-----|------------------------------|
| 1.  | Aluminum 7075-T6             |
| 2.  | Cork                         |
| 3.  | LRSI Coating                 |
| 4.  | HRSI Coating                 |
| 5.  | LI-900 (Bivariate)           |
| 6.  | FRSI Coating                 |
| 7.  | FRSI Nomex Felt (Bivariate)  |
| 8.  | SIP RTV 560                  |
| 9.  | Titanium                     |
| 10. | Coated Columbium             |
| 11. | Copper                       |
| 12. | Beryllium                    |
| 13. | Zirconia                     |
| 14. | Molybdenum                   |
| 15. | RENE 41                      |
| 16. | Micro Quartz Felt Insulation |
| 17. | INCONEL 617                  |
| 18. | RCC                          |
| 19. | Q-Felt 108 (Bivariate)       |
| 20. | Tantalum                     |
| 21. | Tungsten                     |
| 22. | INCONEL X 750                |
| 23. | L 605 Cobalt                 |
| 24. | HAYNES 25                    |
| 25. | MIN-K 1301                   |
| 26. | LI 2200 (Bivariate)          |
| 27. | Nylon Phenolic (ABLATOR)     |
| 28. | B-Stage Cork (ABLATOR)       |
| 29. | MSA-1 (ABLATOR)              |

Table 5.3 Material Identifier Numbers

1	2	AL.7075-T6 DENSITY	660.0
	0.0	175.	
10000.	175.		
7	AL.7075-T6 SPECIFIC HEAT		
	0.0	.170	
310.0	.170		
460.0	.199		
660.0	.210		
1320.0	.279		
1460.0	.279		
10000.	.279		
6	AL.7075-T6 CONDUCTIVITY		
	0.0	1.400E-2	
260.0	1.400E-2		
460.0	2.000E-2		
760.0	2.900E-2		
860.0	2.700E-2		
960.0	2.900E-2		
2	AL.7075-T6 EMISSIVITY		
	0.0	.12	
10000.	.12		
2	2 CORK DENSITY	860.0	
	0.0	10.	
10000.	10.		
2	CORK SPECIFIC HEAT		
	0.0	.04	
10000.	.04		
2	CORK CONDUCTIVITY		
	0.0	6.90E-6	
10000.	6.90E-6		
2	CORK EMISSIVITY		
	0.0	.8	
10000.	.8		
3	2 LRSI COAT DENSITY	1660.0	
	0.0	104.0	
10000.	104.0		
9	LRSI COAT SPECIFIC HEAT		
	0.0	.19	
210.0	.19		
310.0	.17		
460.0	.19		
710.0	.219		
960.0	.240		
1460.0	.285		
2460.0	.345		
3460.0	.390		
9	LRSI COAT CONDUCTIVITY		
	0.0	1.181E-4	
210.0	1.181E-4		
310.0	1.250E-4		
460.0	1.353E-4		
710.0	1.528E-4		
960.0	1.678E-4		
1460.0	1.956E-4		
2460.0	2.453E-4		
3460.0	3.278E-4		

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Table 5.4 Thermophysical Material Properties File  
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2	LRSI COAT	EMISSIVITY
0.0	.80	
10000.	.80	
4	2	HRSI COAT DENSITY      2760.0
0.0	104.0	
10000.	104.0	
9	HRSI COAT	SPECIFIC HEAT
0.0	.15	
210.0	.15	
310.0	.17	
460.0	.19	
710.0	.215	
960.0	.240	
1460.0	.285	
2460.0	.345	
3460.0	.390	
9	HRSI COAT	CONDUCTIVITY
0.0	1.181E-4	
210.0	1.181E-4	
310.0	1.290E-4	
460.0	1.353E-4	
710.0	1.528E-4	
960.0	1.678E-4	
1460.0	1.956E-4	
2460.0	2.433E-4	
3460.0	3.278E-4	
2	HRSI COAT	EMISSIVITY
0.0	.85	
10000.	.85	
5	2	LI-900 DENSITY      2760.0
0.0	9.0	
10000.	9.0	
10	LI-900	SPECIFIC HEAT
0.0	.070	
210.0	.070	
310.0	.105	
460.0	.150	
710.0	.210	
960.0	.252	
1460.0	.288	
1960.0	.300	
2210.0	.303	
3460.0	.303	

Table 5.4 (Continued)

15	LI-900	CONDUCTIVITY				
-6.0	0.0	.21	2.12	21.16	211.6	2116.0
0.0	1.389E-6	1.389E-6	2.083E-6	4.166E-6	6.060E-6	6.472E-6
210.0	1.389E-6	1.389E-6	2.083E-6	4.166E-6	6.060E-6	6.472E-6
460.0	2.083E-6	2.083E-6	2.777E-6	5.083E-6	6.944E-6	7.638E-6
710.0	2.553E-6	2.553E-6	3.472E-6	6.280E-6	8.777E-6	9.472E-6
960.0	3.472E-6	3.472E-6	4.638E-6	7.666E-6	1.111E-5	1.202E-5
1210.0	4.861E-6	4.861E-6	6.000E-6	9.027E-6	1.366E-5	1.483E-5
1460.0	6.472E-6	6.472E-6	7.639E-6	1.089E-5	1.667E-5	1.827E-5
1710.0	8.555E-6	8.555E-6	9.722E-6	1.366E-5	2.014E-5	2.172E-5
1960.0	1.155E-5	1.155E-5	1.275E-5	1.714E-5	2.430E-5	2.616E-5
2210.0	1.575E-5	1.575E-5	1.694E-5	2.130E-5	2.944E-5	3.138E-5
2460.0	2.039E-5	2.039E-5	2.172E-5	2.616E-5	3.927E-5	3.777E-5
2760.0	2.683E-5	2.683E-5	2.833E-5	3.222E-5	4.305E-5	4.638E-5
2960.0	3.222E-5	3.222E-5	3.416E-5	3.861E-5	4.972E-5	5.388E-5
3260.0	4.277E-5	4.277E-5	4.900E-5	5.000E-5	6.111E-5	6.722E-5
3460.0	5.277E-5	5.277E-5	5.444E-5	6.080E-5	7.277E-5	8.059E-5
2	LI-900	EMISSIVITY				
0.0	1.0					
10000.	1.0					
6	2	FRSI COAT DENSITY	1160.0			
0.0	97.0					
10000.	97.0					
2	FRSI COAT SPECIFIC HEAT					
0.0	.39					
10000.	.39					
2	FRSI COAT CONDUCTIVITY					
0.0	5.000E-5					
10000.	5.000E-5					
2	FRSI COAT EMISSIVITY					
0.0	.80					
10000.	.80					
7	2	FRSI NOMEX DENSITY	1160.0			
0.0	5.4					
10000.	5.4					
8	FRSI NOMEX SPECIFIC HEAT					
0.0	.300					
210.0	.300					
460.0	.312					
660.0	.320					
760.0	.335					
1060.0	.345					
1260.0	.360					
1460.0	.380					

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Table 5.4 (Continued)

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## 10 FR81 NOMEK CONDUCTIVITY

-7.0	0.0	.091	.212	2.11E-6	21.16	211.6	2116.0
0.0	1.805E-6	1.805E-6	1.944E-6	2.222E-6	2.555E-6	2.833E-6	3.099E-6
210.0	1.805E-6	1.805E-6	1.944E-6	2.222E-6	2.555E-6	2.833E-6	3.099E-6
460.0	2.222E-6	2.222E-6	2.916E-6	3.888E-6	4.790E-6	6.590E-6	9.722E-6
560.0	2.388E-6	2.388E-6	3.333E-6	4.611E-6	5.694E-6	6.611E-6	6.944E-6
660.0	2.639E-6	2.639E-6	3.633E-6	5.388E-6	6.664E-6	7.639E-6	8.095E-6
760.0	2.833E-6	2.833E-6	4.305E-6	6.111E-6	7.639E-6	8.944E-6	9.861E-6
860.0	3.095E-6	3.095E-6	4.722E-6	6.944E-6	8.777E-6	1.027E-5	1.061E-5
1060.0	3.611E-6	3.611E-6	5.750E-6	8.750E-6	1.130E-5	1.319E-5	1.358E-5
1260.0	4.166E-6	4.166E-6	6.944E-6	1.055E-5	1.388E-5	1.688E-5	1.722E-5
1460.0	4.861E-6	4.861E-6	8.333E-6	1.283E-5	1.708E-5	2.192E-5	2.208E-5

## 2 FR81 NOMEK EMISSIVITY

0.0	1.0
10000.	1.0

8 2 SIP-RTV360 DENSITY 960.0

0.0	98.0
10000.	98.0

## 9 SIP-RTV360 SPECIFIC HEAT

0.0	.373
320.0	.273

360.0	.270
410.0	.260

460.0	.263
560.0	.285

660.0	.300
860.0	.340

1000.0	.340
7	SIP-RTV360 CONDUCTIVITY

0.0	6.472E-5
260.0	6.472E-5

360.0	6.944E-5
460.0	6.805E-5

660.0	5.935E-5
860.0	4.528E-5

960.0	4.055E-5
2	SIP-RTV360 EMISSIVITY

0.0	1.0
10000.	1.0

9 2 TITANIUM DENSITY 1260.0

0.0	912.0
10000.	912.0

Table 5.4 (Continued)

	6	TITANIUM	SPECIFIC HEAT	
	0.0	.096		
	260.0	.096		
	460.0	.129		
	660.0	.146		
	1060.0	.160		
	10000.	.160		
	5	TITANIUM	CONDUCTIVITY	
	0.0	1.200E-3		
	530.0	1.200E-3		
	960.0	1.500E-3		
	1460.0	2.800E-3		
	10000.	2.800E-3		
	2	TITANIUM	EMISSIVITY	
	0.0	.12		
	10000.	.12		
10	2	CTD.COLUMB	DENSITY	2960.0
	0.0	562.0		
	10000.	562.0		
	6	CTD.COLUMB	SPECIFIC HEAT	
	0.0	.059		
	460.0	.059		
	660.0	.061		
	1060.0	.065		
	1460.0	.065		
	10000.	.065		
	6	CTD.COLUMB	CONDUCTIVITY	
	0.0	4.40E-3		
	530.0	4.40E-3		
	960.0	6.10E-3		
	1460.0	7.30E-3		
	2460.0	8.00E-3		
	10000.	8.00E-3		
	4	CTD.COLUMB	EMISSIVITY	
	0.0	.19		
	3160.0	.19		
	4060.0	.24		
	10000.	.24		
11	2	COPPER	DENSITY	1960.0
	0.0	555.0		
	10000.	555.0		
	8	COPPER	SPECIFIC HEAT	
	0.0	.0001		
	60.0	.0400		
	460.0	.088		
	960.0	.100		
	1460.0	.110		
	1960.0	.120		
	2460.0	.130		
	10000.	.130		

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Table 5.4 (Continued)

	8	COPPER	CONDUCTIVITY		ORIGINAL, DRAFT, ETC. OF POOR QUALITY
	0.0	.070			
	400.0	.070			
	460.0	.068			
	960.0	.061			
	1460.0	.058			
	1960.0	.057			
	2460.0	.051			
	10000.	.051			
	2	COPPER	EMISSIVITY		
	0.0	.78			
	10000.	.78			
12	2	BERYLLIUM	DENSITY	1660.0	
	0.0	116.0			
	10000.	116.0			
	10	BERYLLIUM	SPECIFIC HEAT		
	0.0	.000			
	160.0	.001			
	360.0	.20			
	460.0	.39			
	960.0	.58			
	1460.0	.68			
	1960.0	.75			
	2460.0	.84			
	2960.0	.86			
	3460.0	.86			
	9	BERYLLIUM	CONDUCTIVITY		
	0.0	38.88E-3			
	400.0	38.88E-3			
	460.0	35.00E-3			
	960.0	22.22E-3			
	1460.0	17.00E-3			
	1960.0	14.00E-3			
	2460.0	12.30E-3			
	2960.0	12.00E-3			
	3460.0	12.00E-3			
	2	BERYLLIUM	EMISSIVITY		
	0.0	.15			
	10000.	.15			
13	2	ZIRCONIA	DENSITY	3360.0	
	0.0	349.4			
	10000.	349.4			
	6	ZIRCONIA	SPECIFIC HEAT		
	0.0	.119			
	460.0	.115			
	960.0	.140			
	1460.0	.148			
	2460.0	.153			
	3960.0	.155			
	6	ZIRCONIA	CONDUCTIVITY		
	0.0	2.36E-4			
	460.0	2.36E-4			
	960.0	2.56E-4			
	1460.0	2.79E-4			
	2460.0	3.47E-4			
	3960.0	3.61E-4			

Table 5.4 (Continued)

	2	ZIRCONIA EMISSIVITY	
	0.0	.20	
	10000.	.20	
14	2	MOLYBDENUM DENSITY	2660.0
	0.0	640.0	
	10000.	640.0	
	7	MOLYBDENUM SPECIFIC HEAT	
	0.0	.052	
	260.0	.052	
	460.0	.060	
	1460.0	.067	
	2460.0	.080	
	4960.0	.110	
	7460.0	.110	
	7	MOLYBDENUM CONDUCTIVITY	
	0.0	.0244	
	260.0	.0244	
	460.0	.0218	
	1460.0	.0200	
	2460.0	.0160	
	4960.0	.0160	
	7460.0	.0119	
	2	MOLYBDENUM EMISSIVITY	
	0.0	.20	
	10000.	.20	
15	2	RENE 41 DENSITY	2060.0
	0.0	512.0	
	10000.	512.0	
	5	RENE 41 SPECIFIC HEAT	
	0.0	.059	
	530.0	.059	
	930.0	.059	
	2660.0	.230	
	10000.	.230	
	5	RENE 41 CONDUCTIVITY	
	0.0	.0014	
	530.0	.0014	
	930.0	.0022	
	1460.0	.0029	
	2460.0	.0042	
	2	RENE 41 EMISSIVITY	
	0.0	.20	
	10000.	.20	
16	2	MICRO GRTZ DENSITY	1460.0
	0.0	3.5	
	10000.	3.5	
	9	MICRO GRTZ SPECIFIC HEAT	
	0.0	.186	
	535.0	.186	
	760.0	.2173	
	960.0	.2312	
	990.0	.2340	
	1260.0	.26136	
	1380.0	.270	
	1460.0	.27227	
	2260.0	.2950	

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Table 5.4 (Continued)

	9	MICRO GRTZ CONDUCTIVITY	
	0.0	7.220E-6	
	535.0	7.220E-6	
	760.0	7.639E-6	
	860.0	9.020E-6	
	880.0	9.352E-6	
	1260.0	1.951E-5	
	1380.0	1.773E-5	
	1460.0	1.921E-5	
	2260.0	3.970E-5	
	2	MICRO GRTZ EMISSIVITY	
	0.0	.89	
	10000.	.89	
17	2	INCOL 617 DENSITY	2260.0
	0.0	921.86	
	10000.	921.86	
	12	INCOL 617 SPECIFIC HEAT	
	0.0	.100	
	535.0	.100	
	660.0	.104	
	860.0	.111	
	1060.0	.117	
	1460.0	.131	
	1660.0	.137	
	1860.0	.144	
	2060.0	.150	
	2260.0	.157	
	2460.0	.163	
	10000.	.163	
	12	INCOL 617 CONDUCTIVITY	
	0.0	2.176E-3	
	535.0	2.176E-3	
	660.0	2.338E-3	
	860.0	2.616E-3	
	1060.0	2.894E-3	
	1460.0	3.449E-3	
	1660.0	3.727E-3	
	1860.0	4.005E-3	
	2060.0	4.282E-3	
	2260.0	4.560E-3	
	2460.0	4.838E-3	
	10000.	4.838E-3	
	2	INCOL 617 EMISSIVITY	
	0.0	.15	
	10000.	.15	
18	2	RCC DENSITY	3000.0
	0.0	103.7	
	10000.	103.7	

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Table 5.4 (Continued)

11	RCC	SPECIFIC HEAT	
0.0	.080		
160.0	.080		
320.0	.150		
480.0	.170		
960.0	.242		
1440.0	.295		
1920.0	.320		
2400.0	.360		
2960.0	.390		
3440.0	.420		
10000.	.420		
11	RCC	CONDUCTIVITY	
0.0	1.83E-4		
160.0	1.83E-4		
480.0	5.324E-4		
660.0	7.176E-4		
880.0	8.102E-4		
1060.0	8.796E-4		
1440.0	9.239E-4		
2160.0	9.722E-4		
2880.0	9.722E-4		
3360.0	9.491E-4		
10000.	9.491E-4		
8	RCC	EMISSIVITY	
0.0	.8		
880.0	.8		
1280.0	.86		
1760.0	.88		
2240.0	.885		
2880.0	.88		
3440.0	.84		
10000.	.84		
19	2	G-FELT 108 DENSITY	1160.0
		0.0	6.0
		10000.	6.0
	10	G-FELT 108 SPECIFIC HEAT	
		0.0	.20
		680.0	.20
		760.0	.21
		960.0	.24
		1160.0	.26
		1440.0	.27
		1660.0	.28
		1880.0	.29
		2080.0	.30
		10000.	.30

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Table 5.4 (Continued)

			ORIGINAL PAGE "M" OF POOR QUALITY
10	0-FELT 108 CONDUCTIVITY		
-2.0	21.16	2116.0	
0.0	3.700E-6	6.018E-6	
660.0	3.700E-6	6.018E-6	
760.0	4.600E-6	6.944E-6	
960.0	6.700E-6	9.028E-6	
1160.0	9.020E-6	1.157E-5	
1460.0	1.203E-5	1.574E-5	
1660.0	1.435E-5	1.852E-5	
1860.0	1.713E-5	2.129E-5	
2060.0	1.713E-5	2.407E-5	
10000.	1.713E-5	2.407E-5	
6	0-FELT 108 EMISSIVITY		
0.0	.88		
660.0	.88		
1160.0	.70		
1460.0	.63		
1660.0	.60		
10000.	.60		
20	2 TANTALUM DENSITY	4460.0	
0.0	1036.8		
10000.	1036.8		
6	TANTALUM SPECIFIC HEAT		
0.0	.0326		
260.0	.0326		
460.0	.0331		
1460.0	.0356		
2960.0	.0396		
3460.0	.0410		
10000.	.0410		
10	TANTALUM CONDUCTIVITY		
0.0	8.750E-3		
260.0	8.750E-3		
1060.0	1.027E-2		
1440.0	1.067E-2		
2160.0	1.170E-2		
2520.0	1.210E-2		
2820.0	1.230E-2		
3180.0	1.270E-2		
3600.0	1.295E-2		
10000.	1.295E-2		
2	TANTALUM EMISSIVITY		
0.0	.20		
10000.	.20		
21	2 TUNGSTEN DENSITY	4460.0	
0.0	1204.4		
10000.	1204.4		
9	TUNGSTEN SPECIFIC HEAT		
0.0	.0325		
460.0	.0325		
9460.0	.0440		
7460.0	.0470		
10000.	.0470		

Table 5.4 (Continued)

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	7	TUNGSTEN CONDUCTIVITY	
	0.0	1.60E-2	
	460.0	1.60E-2	
	2460.0	1.23E-2	
	3460.0	1.10E-2	
	4460.0	9.50E-3	
	4960.0	9.40E-3	
	10000.	9.40E-3	
	2	TUNGSTEN EMISSIVITY	
	0.0	.066	
	10000.	.066	
22	2	INCONIX750 DENSITY	2260.0
	0.0	531.3	
	10000.	531.3	
	6	INCONIX750 SPECIFIC HEAT	
	0.0	.080	
	260.0	.080	
	460.0	.093	
	1460.0	.136	
	2460.0	.170	
	10000.	.170	
	6	INCONIX750 CONDUCTIVITY	
	0.0	1.900E-3	
	530.0	1.900E-3	
	960.0	2.600E-3	
	1460.0	3.400E-3	
	2460.0	4.800E-3	
	10000.	4.800E-3	
	4	INCONIX750 EMISSIVITY	
	0.0	.60	
	1010.0	.60	
	2035.0	.75	
	10000.	.75	
23	2	L605 COBLT DENSITY	2260.0
	0.0	569.0	
	10000.	569.0	
	4	L605 COBLT SPECIFIC HEAT	
	0.0	.0965	
	160.0	.0965	
	2960.0	.1640	
	10000.	.1640	
	6	L605 COBLT CONDUCTIVITY	
	0.0	1.50E-3	
	530.0	1.50E-3	
	960.0	2.30E-3	
	1460.0	3.10E-3	
	2460.0	4.90E-3	
	10000.	4.90E-3	
	2	L605 COBLT EMISSIVITY	
	0.0	.20	
	10000.	.20	
24	2	HAYNES 23 DENSITY	2460.0
	0.0	570.0	
	10000.	570.0	

Table 5.4 (Continued)

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	2	HAYNES 23	SPECIFIC HEAT				
	0.0	.12					
10000.	.12						
	9	HAYNES 23	CONDUCTIVITY				
	0.0	1.38E-2					
160.0	1.39E-2						
1460.0	3.75E-2						
2260.0	5.55E-2						
10000.	5.95E-2						
	2	HAYNES 23	EMISSIVITY				
	0.0	.78					
10000.	.78						
25	2	MIN-K 1301	DENSITY	1260.0			
	0.0	20.0					
10000.	20.0						
	2	MIN-K 1301	SPECIFIC HEAT				
	0.0	.212					
10000.	.212						
	7	MIN-K 1301	CONDUCTIVITY				
	0.0	4.638E-6					
160.0	4.638E-6						
860.0	4.861E-6						
1060.0	5.333E-6						
1260.0	5.777E-6						
1460.0	6.250E-6						
10000.	6.290E-6						
	2	MIN-K 1301	EMISSIVITY				
	0.0	.93					
10000.	.93						
26	2	LI-2200	DENSITY	2760.0			
	0.0	22.0					
10000.	22.0						
	2	LI-2200	SPECIFIC HEAT				
	0.0	.19					
10000.	.30						
	15	LI-2200	CONDUCTIVITY				
-6.0	0.0	.21	2.12	21.16	211.6	2116.	
0.0	3.88E-6	3.88E-6	4.44E-6	7.22E-6	8.33E-6	8.88E-6	
210.0	3.88E-6	3.88E-6	4.44E-6	7.22E-6	8.33E-6	8.88E-6	
460.0	4.72E-6	4.72E-6	6.11E-6	8.61E-6	9.16E-6	1.19E-5	
710.0	6.11E-6	6.11E-6	7.22E-6	1.09E-5	1.30E-5	1.50E-5	
960.0	6.66E-6	6.66E-6	8.88E-6	1.11E-5	1.55E-5	1.80E-5	
1210.0	8.09E-6	8.09E-6	1.02E-5	1.23E-5	1.77E-5	2.11E-5	
1460.0	9.44E-6	9.44E-6	1.19E-5	1.44E-5	2.08E-5	2.44E-5	
1710.0	1.13E-5	1.13E-5	1.38E-5	1.64E-5	2.35E-5	2.88E-5	
1960.0	1.38E-5	1.38E-5	1.63E-5	1.97E-5	2.94E-5	3.30E-5	
2210.0	1.66E-5	1.66E-5	1.94E-5	2.30E-5	3.41E-5	3.88E-5	
2460.0	1.94E-5	1.94E-5	2.30E-5	2.77E-5	3.94E-5	4.44E-5	
2760.0	2.34E-5	2.34E-5	2.77E-5	3.33E-5	4.63E-5	5.25E-5	
2960.0	2.66E-5	2.66E-5	3.13E-5	3.75E-5	5.16E-5	5.88E-5	
3260.0	3.22E-5	3.22E-5	3.75E-5	4.55E-5	6.08E-5	6.94E-5	
3460.0	3.63E-5	3.63E-5	4.23E-5	5.23E-5	6.80E-5	7.77E-5	

Table 5.4 (Continued)

2	L1-2200	EMISSIVITY	
0.0	.80		
10000.	.90		
27	2	NYLON PHEN DENSITY	6210.0
	0.0	94.0	
	10000.	94.0	
	5	NYLON PHEN SPECIFIC HEAT	
	0.0	.20	
	560.0	.21	
	660.0	.25	
	960.0	.275	
	10000.	.275	
	9	NYLON PHEN CONDUCTIVITY	
	0.0	1.39E-5	
	460.0	1.39E-5	
	660.0	1.94E-5	
	910.0	2.90E-5	
	10000.	2.90E-5	
	2	NYLON PHEN EMISSIVITY	
	0.0	.89	
	10000.	.89	
27	5	NYLON PHEN SUBLIM. TEMP	
	0.0	5670.0	
	21.16	5670.0	
	211.6	5880.0	
	2116.	6210.0	
	21160.0	6210.0	
27	5	NYLON PHEN HEAT-ABLATION	
	0.0	11000.0	
	21.16	11000.0	
	211.6	10600.0	
	2116.	9200.0	
	21160.	9200.0	
28	2	B-STG.CORK DENSITY	1220.0
	0.0	31.0	
	10000.	31.	
	2	B-STG.CORK SPECIFIC HT.	
	0.0	.46	
	10000.	.46	
	4	B-STG.CORK CONDUCTIVITY	
	0.0	1.11 E-5	
	860.0	1.11 E-5	
	1310.0	3.33 E-6	
	10000.0	3.33 E-6	
	2	B-STG.CORK EMISSIVITY	
	0.0	.8	
	10000.	.8	
28	2	B-STG.CORK SUB.TEMP	

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Table 5.4 (Continued)

0.0	1220.		
5000.	1220.		
28 2	B-STD.CORN HEAT ABL.		
0.0	7000.		
5000.	7000.		
24 2	MSA-1	DENSITY	1000.0
0.0	16.0		
10000.	16.		
4	MSA-1	SPECIFIC WT.	
0.0	.28		
500.0	.28		
1000.	.56		
10000.	.56		
4	MSA-1	CONDUCTIVITY	
0.0	8.30 E-6		
510.	8.30 E-6		
1100.	1.39 E-6		
10000.	1.39 E-6		
2	MSA-1	EMISSIVITY	
0.0	.9		
10000.	.9		
29 2	MSA-1	SUB.TEMP	
0.0	1000.		
5000.	1000.		
29 2	MSA-1	HEAT-ABL.	
0.0	3000.		
5000.	3000.		
-1	END OF FILE		

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Table 5.4 (Continued)

where

KD = Material Identifier Number  
JD = Number of entries in Table (down the page)  
TEST1 = Material Name  
TEST2 = Property (Density, specific heat, etc.)  
TMAXA = Maximum allowable temperature for material.  
(Read on density header card)

The next JD records are then read to load the property into arrays using the following read statement and format.

```
READ(8,702) (ARD(MR),MR=1,8)
702 FORMAT(5X,8E10.0).
```

For a monovariate table, the independent variable is ARD(1) and the dependent variable is ARD(2). For a bivariate table, ARD(1) is the negative of the number of pressure entries, going across the page. The pressure values are stored in ARD(2) through ARD(8).

The next JD records are read by the same read statement. ARD(1) will then be the temperature while ARD(2) through ARD(8) are the properties i. e. conductivity or specific heat.

All property tables must be arranged in a particular order. The first property must be density followed by specific heat, conductivity and emissivity. For an ablator-sublimer material, sublimation temperature, and heat of ablation are added as the fifth and sixth properties. To flag a material as being an ablator sublimer, the material identification number is included on the header

card for the sublimation temperature table. Units for the various properties are given in Table 5.5 and are always used regardless of the units set used in the input.

PROPERTY	UNITS
DENSITY	LBM./FT. <sup>3</sup>
MAXIMUM ALLOWABLE TEMPERATURE	DEGREES-R
SPECIFIC HEAT	BTU/LBM.-R
THERMAL CONDUCTIVITY	BTU/Ft-SEC-R
EMISSIVITY	DIMENSIONLESS
SUBLIMATION TEMPERATURE	DEGREES-R
HEIGHT OF ABLATION	BTU/LBM.

TABLE 5.5 Material Property Units

### 5.9 STRUCTURES FILE

The EXITS program will create a file which saves the geometric and material definition of the thermal protection system being analyzed. The user assigns this definition a structure number which is used to identify the structure for later use. By doing this, the user can reevaluate the same thermal protection system under different environment conditions with a much reduced interactive input. The name of the structure file is input during the interactive portion of the input. Structures are added to the file at the bottom or below any structure which already exist in the file. If any of the existing structures have the structure number the user is using to identify the new structure, a message will appear during the interactive input asking for a new identification number. Therefore, each structure in the file will have a distinct structure identification number.

An example of a structure definition is given in Table 5.6 for the demon-

tration cases presented in Section 5.4 and 5.6. No format specifications need to be discussed here since EXITS creates and reads this file exclusively.

The first record gives the identification number, the number of layers, the number of materials per layer and the number of dimensions per layer to define the geometry. The next two lines are a description typed in during the interactive input. The next four lines describe each layer. The first entry gives the layer type. Material types are given in the next three locations. Finally, the next six floating point values define the layer geometry dimensions in feet.

1 4 3 6

TEST CASE STRUCTURE FOR Langley CENTER

ABLATOR SUBLIMER - RADIATION GAP - THIN SKIN - 2 STANDOFF

7	28	0	0	0.8333334E-01	0.000000E+00	0.000000E+00	0.000000E+00	0.000000E+00	0.000000E+00
2	10	1	0	0.1041667E-01	0.8333334E-02	0.000000E+00	0.1250000E+00	0.000000E+00	0.000000E+00
6	1	0	0	0.0000000E+00	0.0000000E+00	0.0000000E+00	0.2500000E-01	0.0000000E+00	0.0000000E+00
3	17	10	1	0.1166667E-01	0.1500000E-01	0.7916667E-02	0.1666667E+00	0.6666667E+00	0.6250000E-01

2 4 3 6

TEST CASE STRUCTURE FOR Langley CENTER

SLAB - SLAB - HONEY COMB - CORRUGATED

1	4	0	0	0.8333334E-02	0.0000000E+00	0.0000000E+00	0.0000000E+00	0.0000000E+00	0.0000000E+00
1	5	0	0	0.4166667E-01	0.0000000E+00	0.0000000E+00	0.0000000E+00	0.0000000E+00	0.0000000E+00
3	1	1	1	0.1000000E-01	0.9166666E-02	0.7916667E-02	0.6250000E-01	0.0000000E+00	0.2500000E-01
4	17	17	9	0.6666666E-02	0.6666666E-02	0.1000000E-01	0.8333334E-01	0.6666667E-01	0.0000000E+00

3 1 3 6

ABLATOR SUBLIMER

HALF INCH OF CORK

7	27	0	0	0.4166667E-01	0.0000000E+00	0.0000000E+00	0.0000000E+00	0.0000000E+00	0.0000000E+00
---	----	---	---	---------------	---------------	---------------	---------------	---------------	---------------

-9999

TABLE 5.6 Structure File For Sample Case Given  
In Section 5.4 And 5.6

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5.4 EXAMPLE CASE ONE (ABLATOR, RADIATION GAP, THIN SKIN, Z-STANDOFF)

This first example case configuration is not representative of an actual thermal protection system structure but serves to illustrate the interactive input requirements for four of the structural types. The configuration for this case is shown in Figure 5.1.

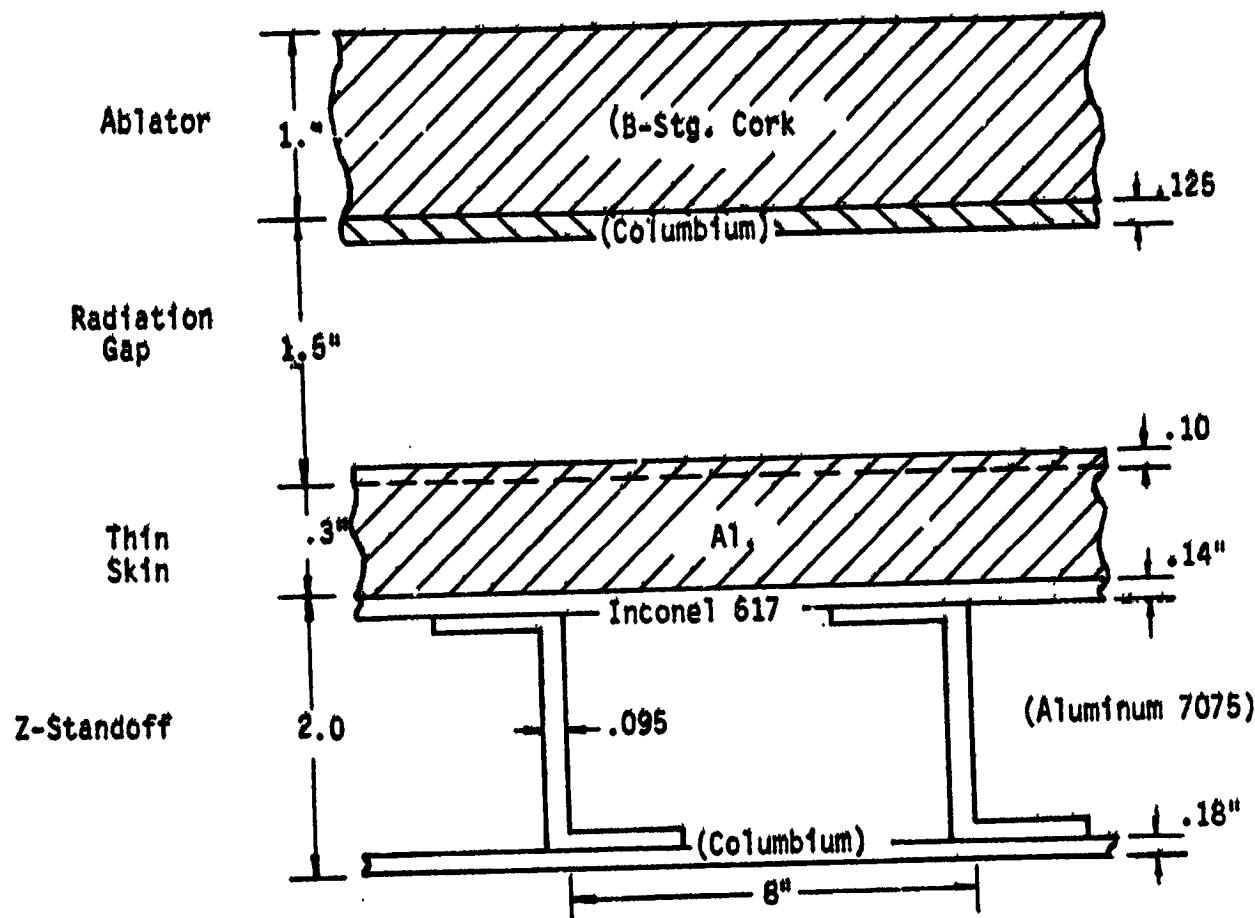


Fig. 5.1 Configuration for Test Case Number One

As shown in Figure 5.1, an ablator 1.0 inch thick is placed on top of a columbium sheet .125 inches in thickness. Next we have an aluminum plate .4 inches thick placed over an Inconel 617 sheet .14 inches thick. An aluminum Z-stanoff structure separates the Inconel 617 sheet from a columbium backface sheet .18 inches thick. We see that the .4 inch aluminum plate is divided at a depth of .1 inches from the top surface. This is required to define the radiation gap model since a lower surface plate is needed. The thermal resistance in the .1 inch aluminum is included in the radiation gap model and does not effect the time step. The aerothermodynamic environment is located on the file MINIVER.DAT and identified by body point number 3. The structure was saved on the structure file STRUCTURE.FIL which is shown in Table 5.6. Also included in this example is the input required to rerun this case using the structural definition saved on STRUCTURE.FIL. The interactive input is shown in Table 5.7. For the case where this example is rerun using the saved structure, the interactive input is shown in Table 5.8.

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**OPTIONAL FORM D**  
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LINE No.	LINR. No.	EXPLANATION
	1	RUN EXISTS
	1	WHAT IS THE MINIVER INPUT DATA FILE NAME ?
	2	MINIVER.DAT
	2	WHAT IS THE STRUCTURE FILE NAME ?
	STRUCTURE.FIL	
	3	WHAT IS THE NAME OF THE OUTPUT FILE ?
	OUTPUT.DAT	
	4	WHAT IS THE INITIAL TIME(SEC) ?
	0.0	
	5	WHAT IS THE FINAL TIME(SEC) ?
	1410.0	
	6	WHAT IS THE TIME(SEC) BETWEEN PRINTOUTS ?
	100.0	
	7	DO YOU WANT TO RESET CONTROL PARAMETERS ?
	N	
	8	WHAT IS THE TOTAL NUMBER OF BODY POINTS ?
	1	
	9	WHAT IS THE BODY POINT NUMBER ?
	1	
	10	DO YOU WANT TO RESET THE TIME OR CONTROL PARAMETERS ?
	T	
	11	WHAT IS THE INITIAL TIME(SEC) ?
	0.0	
	12	WHAT IS THE FINAL TIME(SEC) ?
	1410.0	
	13	WHAT IS THE TIME(SEC) BETWEEN PRINTOUTS ?
	100.0	
	14	DO YOU WANT TO RESET CONTROL PARAMETERS ?
	Y	
	15	RESOLUTION: DEFAULT = 10.0 NEW VALUE =
	15	STABILITY: DEFAULT = 2.0 NEW VALUE =
	17	ITERATION TOLERANCE: DEFAULT = .001 NEW VALUE =
	18	RELAXATION FACTOR: DEFAULT = 0.5 NEW VALUE =
	19	NUMBER OF STEPS BETWEEN PARAMETER CALC.: DEFAULT = 20 NEW VALUE =
	20	MAXIMUM NUMBER OF ITERATIONS: DEFAULT = 5000 NEW VALUE =
		*If no response, return key will advance code to next question and default value will be used.

Table 5.7 Interactive Input for Configuration Sheet in Figure 5.1

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LINE NO.	ENGLISH(SHIFTDEFAULT) METRIC	METRIC	LINE NO.	EXPLANATION
21	ARE THE UNITS OF MINIVER.DAT	IN ENGLISH OR METRIC ?	21.	Units of thermocouplement E = English, N = Metric (See Table 5-2).
E	DEG F	DEG K	22.	E = English, N = Metric.
22	DO YOU WANT OUTPUT DATA IN ENGLISH OR METRIC ?	IN	23.	E = English, N = Metric.
E	BTU	JOULES	24.	N = No, Y = Yes. If yes the values of the capacitors and conductors will be printed.
23	DO YOU WANT INPUT DATA IN ENGLISH OR METRIC ?	KGM	25-27	At this point, questions 8-10 are repeated and quantities can be changed.
E	LBM		28.	Initial temperature of structure is degrees F if answer to question 23 is E or degrees K if answer to 23 is N.
24	DO YOU WANT ADDITIONAL PRINTOUT ?		29.	Temperature of sink which structure radiates to in degrees F if answer to question 23 is E or degrees K if answer is N.
N			30.	View factor between surface of structure and sink.
25	WHAT IS THE TOTAL NUMBER OF BODY POINTS ?		31.	Y = Yes, N = No. If the answer is Y = Yes, then questions defining the structure are omitted. See Table 5-7 for input previously defined using structures file.
I			32.	Y = Yes, N = No. If the answer is Yes, then structure definition will be added to structures file.
26	WHAT IS THE BODY POINT NUMBER ?		33.	Number of layers of structure. For this case we have four, ablator, radiation gap, thin skin, and Z-standoff.
I				
27	DO YOU WANT TO RESET THE TIME OR CONTROL PARAMETERS ?			
P				
28	WHAT IS THE INITIAL TEMPERATURE OF BODY PT.	3 ?		
P	100.0			
29	WHAT IS THE SINK TEMPERATURE OF BODY PT.	3 ?		
P	0.0			
30	WHAT IS THE VIEW FACTOR FOR BODY PT.	3 ?		
P	1.0			
31	DOES THE STRUCTURE FOR BODY PT.	3 EXIST IN THE STRUCTURE FILE ?		
P				
32	DO YOU WANT TO ADD THE STRUCTURE FOR BODY PT.	3 TO THE STRUCTURE FILE ?		
P				
33	HOW MANY LAYERS AT BODY PT.	3 ?		
P				

Table 5.7 (Continued)

STRUCTURE TYPE ---	NUMBER
SLAB	1
RADIATION GAP	2
HONEYCOMB	3
CORRUGATED	4
Z STANDOFF	5
THIN SKIN	6
ABLATOR SUBLINER	7

- LINE NO. QUESTION ANSWER EXPLANATION
34. WHAT IS THE STRUCTURE TYPE NUMBER FOR LAYER 1 OF BODY PT. 3 ? Number of structure type for first layer. Answer 1 through 7 according to table.
35. WHAT IS THE MAT. IDENTIFIER AND THE MAT. THICKNESS FOR LAYER 1 OF BODY PT. 3 ? Material identifier (Table 5.3) and thickness of ablator.
36. ARE THERE ANY CORRECTIONS FOR LAYER 1 OF BODY POINT 3 ? Y = Yes, N = No. If answer is yes, will return to line 34.
37. WHAT IS THE STRUCTURE TYPE NUMBER FOR LAYER 2 OF BODY PT. 3 ? Number of structure type for second layer. Answer 1 through 7 according to table.
38. MATERIAL IDENTIFIER (Table 5.3) AND THICKNESS FOR THE TOP MATERIAL OF THE RADIATION GAP. FOR THIS EXAMPLE CERMELIM (10) .125 INCHES THICK.
39. MATERIAL IDENTIFIER (Table 5.3) AND THICKNESS OF BOTTOM MATERIAL OF THE RADIATION GAP.
40. OVERALL HEIGHT OF RADIATION GAP.
41. Y = Yes, N = No. If answer is yes, will return to line 37.
37. WHAT IS THE STRUCTURE TYPE NUMBER FOR LAYER 2 OF BODY PT. 3 ?
38. WHAT IS THE MAT. IDENTIFIER AND THICKNESS OF MAT. 1 FOR LAYER 2 OF BODY PT. 3 ? 10 .125
39. WHAT IS THE MAT. IDENTIFIER AND THICKNESS OF MAT. 2 FOR LAYER 2 OF BODY PT. 3 ? 1 .10
40. WHAT IS THE STRUCTURE HEIGHT FOR LAYER 2 OF BODY PT. 3 ? 1.5
41. ARE THERE ANY CORRECTIONS FOR LAYER 2 OF BODY POINT 3 ? N

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Table 5.7 (Continued)

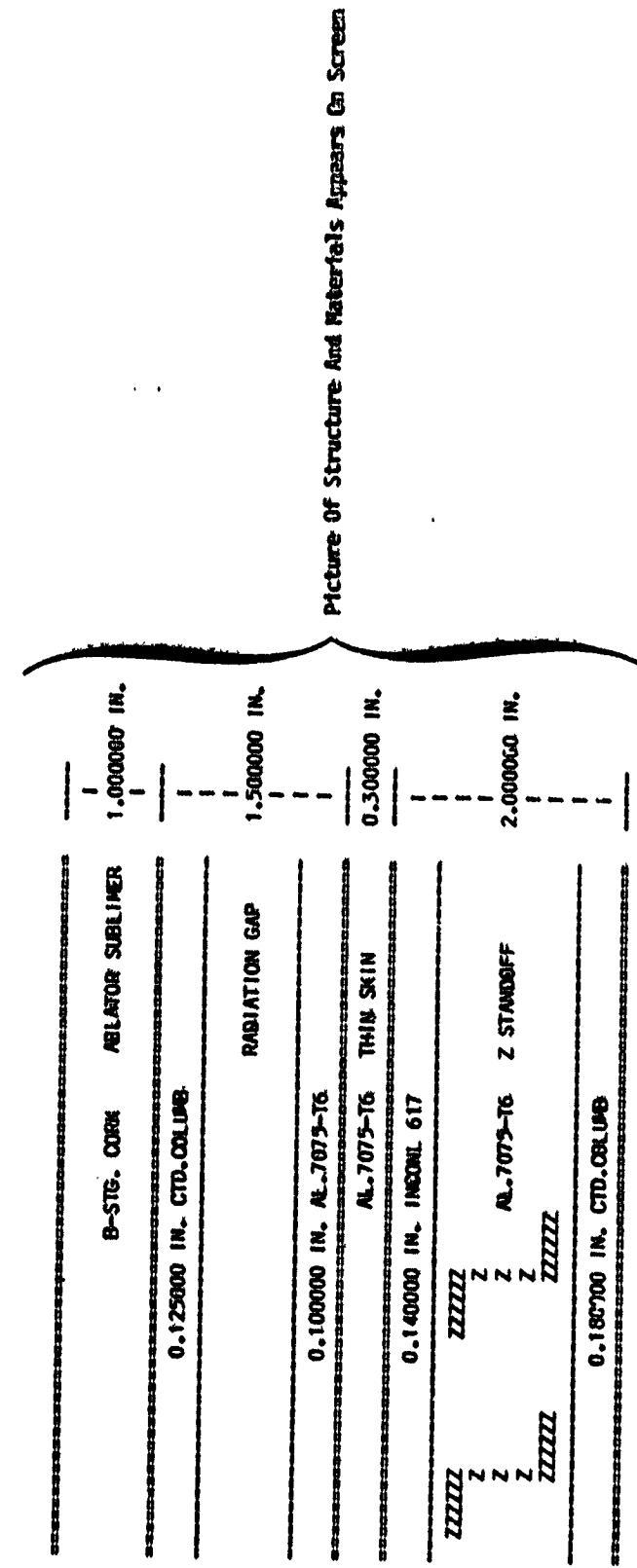
STRUCTURE TYPE	---	NUMBER
SLAB		1
RADIATION GAP		2
HONEYCOMB		3
CORRUGATED		4
Z STANDOFF		5
THIN SKIN		6
ABLATOR SUBLIMER		7

- LINE NO.
- 42 WHAT IS THE STRUCTURE TYPE NUMBER FOR LAYER 3 OF BODY PT. 3 ?
- 43 WHAT IS THE MAT. IDENTIFIER AND THE INT. THICKNESS FOR LAYER 3 OF BODY PT. 3 ?  
1 .3
- 44 ARE THERE ANY CORRECTIONS FOR LAYER 3 OF BODY POINT N
- 45 WHAT IS THE STRUCTURE TYPE NUMBER FOR LAYER 4 OF BODY PT. 3 ?
- 46 WHAT IS THE MAT. IDENTIFIER AND THICKNESS OF MAT. 1 FOR LAYER 4 OF BODY PT. 3 ?  
17 .14
- 47 WHAT IS THE MAT. IDENTIFIER AND THICKNESS OF MAT. 2 FOR LAYER 4 OF BODY PT. 3 ?  
10 .18
- 48 WHAT IS THE MAT. IDENTIFIER AND THICKNESS OF MAT. 3 FOR LAYER 4 OF BODY PT. 3 ?  
1 .005
- 49 WHAT IS THE STRUCTURE HEIGHT, PITCH, AND FLANGE WIDTH FOR LAYER 4 OF BODY PT. 3 ?  
2.0 8.0 .75
- 50 ARE THERE ANY CORRECTIONS FOR LAYER 4 OF BODY POINT N
- EXPLANATION
42. Number of structure type for third layer. Answer 1 through 7 according to table.
43. Material identifier (Table 5.3) and thickness. (.3 for this example).
44. Y = Yes, N = No. If yes, will return to line 42.
45. Number of structure type for fourth layer. Answer 1 through 7 according to table.
46. Material identifier (Table 5.3) and thickness of top of Z-standoff structure.
47. Material identifier (Table 5.3) and thickness for bottom of Z-standoff structure.
48. Material identifier (Table 5.3) and thickness for middle or Z part of structure.
49. Overall height of Z-standoff structure (2.0 inches), pitch or distance between Z structures (8.0 inches), and flange width (.75 inches) of Z-standoff.
50. Y = Yes, N = No. If yes then will return to line 45.

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Table 5.7 (Continued)

THIS IS THE CONFIGURATION FOR BODY PT. 3



LINE  
NO.

51. ARE THERE ANY CORRECTIONS FOR BODY PT. 3 ?

N

52. WHAT IS THE STRUCTURE NUMBER FOR BODY PT. 3

1

53. GIVE A TWO LINE DESCRIPTION OF THE STRUCTURE FOR BODY PT. 3  
TEST CASE STRUCTURE FOR LANGLEY CENTER  
ABLATOR SUBLINER - RADIATION GAP - THIN SKIN - Z STANDOFF

51. Y = Yes, N = No. If answer is yes will return to line number 10.
52. Answer assigns a structure identification number to the thermal protection system definition for storage in the structures file.
53. Reply is two line description of structure which is included in the structures file.

MODEL COMPLETE - - - - - GONE TO EXECUTE

- - - EXECUTION COMPLETE - - -

OUTPUT FILENAME = OUTPUT.DAT

Table 5.7 (Concluded)

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## 5.5 EXAMPLE CASE ONE RERUN FROM STRUCTURES FILE

Here we have rerun the case previously described using the data stored on the structures file which defines the thermal protection system. Table 5.8 shows the input for this case. As can be seen, the interactive input has been greatly simplified. The description of lines one to thirteen has been previously given. At line 14, the response is Y (yes) since we have already described the structure. Line 15 asks for the structure identifier number which was assigned previously. If the answer at line 16 is Y (yes) then control is returned to line 7, if N (no) we have completed the interactive input and EXITS goes into execution.

THIS IS THE CONFIGURATION FOR BODY FT. 3

LINE NO.	RUN EXITS	B-SIG. ODRK	ABLATOR SUBLINER	1.00000 IN.
1	WHAT IS THE MIVER INPUT DATA FILE NAME ?			
2	MIVVER.DAT			
3	STRUCTURE.FIL			
3	WHAT IS THE NAME OF THE OUTPUT FILE ?			
4	OUTPUT.DAT			
4	WHAT IS THE INITIAL TIME(SEC) ?			
5	0.0			
5	WHAT IS THE FINAL TIME(SEC) ?			
6	1410.0			
6	WHAT IS THE TIME(SEC) BETWEEN PRINTOUTS ?			
7	100.0			
7	DO YOU WANT TO RESET CONTROL PARAMETERS ?			
N				
8	WHAT IS THE TOTAL NUMBER OF BODY POINTS ?			
1				
9	WHAT IS THE BODY POINT NUMBER ?			
3				
10	DO YOU WANT TO RESET THE TIME OR CONTROL PARAMETERS ?			
N				
11	WHAT IS THE INITIAL TEMPERATURE OF BODY PT. 3 ?			
100.0				
12	WHAT IS THE SINK TEMPERATURE OF BODY PT. 3 ?			
0.0				
13	WHAT IS THE VIEW FACTOR FOR BODY PT. 3 ?			
1.0				
14	DOES THE STRUCTURE FOR BODY PT. 3 EXIST IN THE STRUCTURE FILE ?			
Y				
15	WHAT IS THE STRUCTURE NUMBER FOR BODY PT. 3 ?			
1				
	STRUCTURE NUMBER = 1			
	TEST CASE STRUCTURE FOR LANGLEY CENTER			
	ABLATOR SUBLINER - RADIATION GAP - THIN SKIN - Z STANDOFF			
	LINEx			
	NO.			
	0.125000 IN. CTD.OCURS			
	RADIATION GAP			
	1.500000 IN.			
	0.100000 IN. AL.7075-T6			
	AL.7075-T6 THIN SKIN			
	0.140000 IN. INCOR.617			
	ZZZZZZ			
	Z Z			
	Z Z			
	Z Z			
	0.100000 IN. CTD.OCURS			
	2.000000 IN.			
	ZZZZZZ			
	Z Z			
	Z Z			
	Z Z			
	LINE NO.			
	16 ARE THERE ANY CORRECTIONS FOR BODY FT. 3 ?			
	N			

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MODEL COMPLETE ----- Close to execute

----- EXECUTION COMPLETE -----

OUTPUT FILENAME = OUTPUT.DAT

>

Table 5.8 Example of Sample Case One Rerun From Structures File

### 5.6 EXAMPLE CASE TWO (SLAB, SLAB, HONEYCOMB AND CORRUGATED)

In this example the input requirements for the remaining three structures (Slab, Honeycomb, Corrugated) are demonstrated. Again, this case is not representative of a thermal protection system but serves to illustrate the input requirements for the remaining three structures. The configuration for this example case is shown in Figure 5.2.

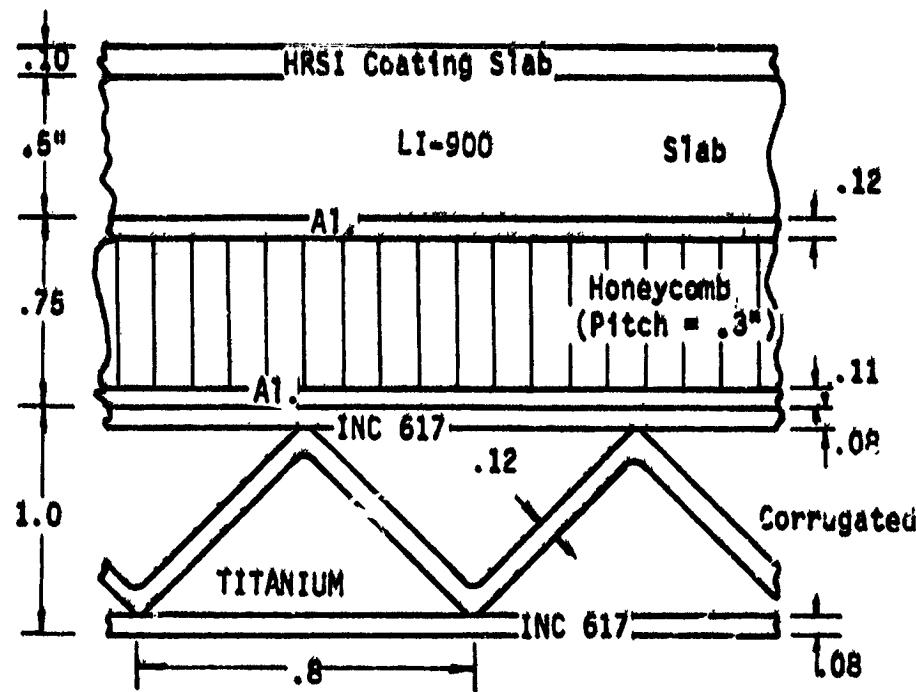


Fig. 5.2 Configuration for Test Case Number Two

As shown in Figure 5.2 a LI-900 insulation .5 inch thick with a .10 inch coating of HRSI coating material is backed up by a honeycomb structure .75 inches thick and a corrugated structure. Different materials are used in the honeycomb and corrugated layers to illustrate their input. The aerothermodynamic environment is defined on the file MINIVER.DAT and identified by body point 9. The structure was saved on the structure file STRUCTURE.FIL and is included in the example shown in Table 5.6. The interactive input for this case is presented in Table 5.9.

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LINE NO.	EXPLANATION	LINE NO.
1	RUN EXISTS	1-16 Same as shown in Example Case One, Table 5.7.
2	MINIVER.DAT	
2	WHAT IS THE STRUCTURE FILE NAME ?	
2	STRUCTURE.FIL	
3	WHAT IS THE NAME OF THE OUTPUT FILE ?	
3	OUTPUT.FIL	
4	WHAT IS THE INITIAL TIME(SEC) ?	
4	0.0	
5	WHAT IS THE FINAL TIME(SEC) ?	
5	1210.0	
6	WHAT IS THE TIME(SEC) BETWEEN PRINTOUTS ?	
6	100.0	
7	DO YOU WANT TO RESET CONTROL PARAMETERS ?	
7	N	
8	WHAT IS THE TOTAL NUMBER OF BODY POINTS ?	
8	1	
9	WHAT IS THE BODY POINT NUMBER ?	
9	3	
10	DO YOU WANT TO RESET THE TIME OR CONTROL PARAMETERS ?	
10	N	
11	WHAT IS THE INITIAL TEMPERATURE OF BODY PT.	3 ?
11	100.0	
12	WHAT IS THE SINK TEMPERATURE OF EASY PT.	3 ?
12	0.0	
13	WHAT IS THE VIEW FACTOR FOR BODY PT.	3 ?
13	1.0	
14	DOES THE STRUCTURE FOR BODY PT. 3 EXIST IN THE STRUCTURE FILE ?	
14	N	
15	DO YOU WANT TO ADD THE STRUCTURE FOR BODY PT.	3 TO THE STRUCTURE FILE ?
15	Y	
16	HOW MANY LAYERS AT BODY PT.	3 ?
16	4	

Table 5.9 Interactive Input for Configuration Shown in Figure 5.2

STRUCTURE TYPE - - -	NUMBER
SLAB	1
RADIATION GAP	2
HONEYCOMB	3
CORRUGATED	4
Z STANDOFF	5
THIN SKIN	6
RELATOR SUBLIMER	7

LINE  
NO.

17. WHAT IS THE STRUCTURE TYPE NUMBER FOR LAYER 1 OF BODY PT. 3 ?  
 ;  
 18. WHAT IS THE MAT. IDENTIFIER AND THE MAT. THICKNESS  
     FOR LAYER 1 OF BODY PT. 3 ?  
     4 .10
19. ARE THERE ANY CORRECTIONS FOR LAYER 1 OF BODY POINT 3 ?  
 N

STRUCTURE TYPE - - -	NUMBER
SLAB	1
RADIATION GAP	2
HONEYCOMB	3
CORRUGATED	4
Z STANDOFF	5
THIN SKIN	6
RELATOR SUBLIMER	7

EXPLANATION

- LINE  
NO.
17. Number of structure type for first layer.  
 18. Material identifier (Table 5.3) and thickness of slab.  
 19. Y = Yes, N = No. If answer is yes, return to line 17.  
 20. Number of structure type for second layer.  
 21. Material identifier (Table 5.3) and thickness of second slab.  
 22. Y = Yes, N = No. If answer is yes, return to line 20.
20. WHAT IS THE STRUCTURE TYPE NUMBER FOR LAYER 2 OF BODY PT. 3 ?  
 1  
 21. WHAT IS THE MAT. IDENTIFIER AND THE MAT. THICKNESS  
     FOR LAYER 2 OF BODY PT. 3 ?  
     5 .5
22. ARE THERE ANY CORRECTIONS FOR LAYER 2 OF BODY POINT 3 ?  
 N

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Table 5.9 (Continued)

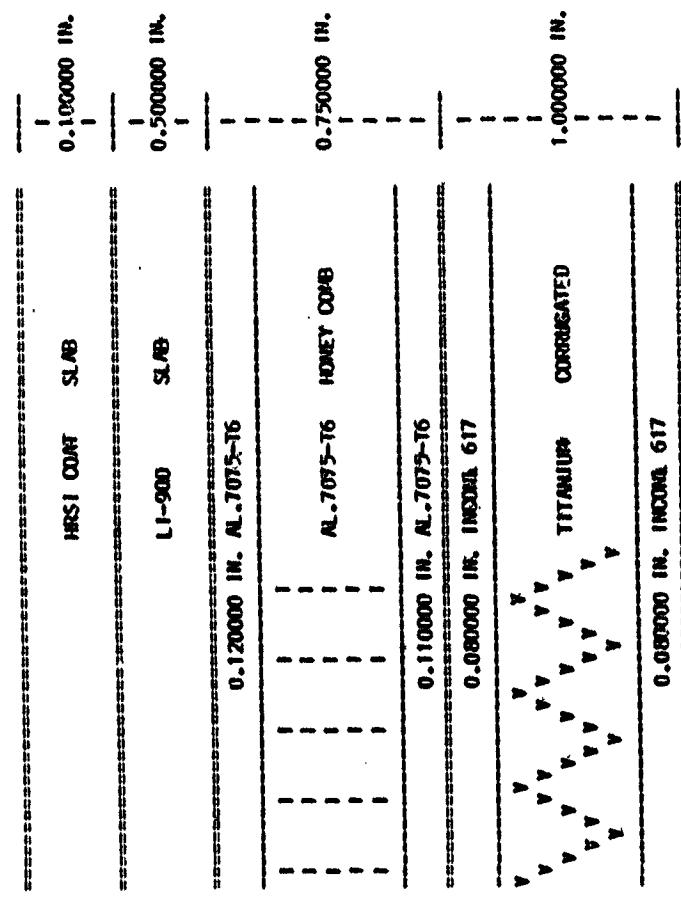
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STRUCTURE TYPE	--	NUMBER
SLAB	--	1
RADIATION GAP	--	2
HONEYCOMB	--	3
CORRUGATED	--	4
Z STANDOFF	--	5
THIN SKIN	--	6
AVIATOR SEALER	--	7

- | LINE NO. | STRUCTURE TYPE  | -- | NUMBER   | LINE NO. | EXPLANATION  |
|----------|---|----|----------|----------|--|
| 23       | WHAT IS THE STRUCTURE TYPE NUMBER FOR LAYER 3 OF BODY PT.                   | -- | 3?       | 23.      | Number of structure type for third layer.  |
| 24       | WHAT IS THE MAT. IDENTIFIER AND THICKNESS OF MAT. 1 FOR LAYER 3 OF BODY PT. | 3? | 1 .12    | 24.      | Material identifier (Table 5.3) and thickness of top of honeycomb.               |
| 25       | WHAT IS THE MAT. IDENTIFIER AND THICKNESS OF MAT. 2 FOR LAYER 3 OF BODY PT. | 3? | 1 .11    | 25.      | Material identifier (Table 5.3) and thickness of bottom of honeycomb.            |
| 26       | WHAT IS THE MAT. IDENTIFIER AND THICKNESS OF MAT. 3 FOR LAYER 3 OF BODY PT. | 3? | 1 .095   | 26.      | Material identifier (Table 5.3) and thickness of honeycomb core.                 |
| 27       | WHAT IS THE STRUCTURE HEIGHT AND CELL DIMENSIONS OF LAYER 3 OF BODY PT.     | 3? | .75 .30  | 27.      | Overall height of honeycomb and honeycomb cell pitch.                            |
| 28       | ARE THERE ANY CORRECTIONS FOR LAYER 3 OF BODY POINT N                       | 3? | N        | 28.      | Y = Yes, N = No. If answer is yes, return to line 23.                            |
| 29       | WHAT IS THE STRUCTURE TYPE NUMBER FOR LAYER 4 OF BODY PT.                   | -- | 3?       | 29.      | Number of structure type of fourth layer.  |
| 30       | WHAT IS THE MAT. IDENTIFIER AND THICKNESS OF MAT. 1 FOR LAYER 4 OF BODY PT. | 3? | 17 .06   | 30.      | Material identifier (Table 5.3) and thickness of top of corrugated structure.    |
| 31       | WHAT IS THE MAT. IDENTIFIER AND THICKNESS OF MAT. 2 FOR LAYER 4 OF BODY PT. | 3? | 17 .08   | 31.      | Material identifier (Table 5.3) and thickness of bottom of corrugated structure. |
| 32       | WHAT IS THE MAT. IDENTIFIER AND THICKNESS OF MAT. 3 FOR LAYER 4 OF BODY PT. | 3? | 9 .12    | 32.      | Overall height of corrugated structure and pitch of corrugations.                |
| 33       | WHAT IS THE STRUCTURE HEIGHT AND PITCH FOR LAYER 4 OF BODY PT.              | 3? | 15 .0 .8 | 33.      | Y = Yes, N = No. If answer is yes, return to line 29.                            |
| 34       | ARE THERE ANY CORRECTIONS FOR LAYER 4 OF BODY POINT N                       | 3? | N        | 34.      |  |

Table 5.9 (Continued)

THIS IS THE CONFIGURATION FOR BODY PT. 3



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- LINE NO. EXPLANATION  
35 N Same as Example Case One, Table 5.7.  
36 2  
37 3

MODEL COMPLETE - - - - - GONE TO EXECUTE

- - - EXECUTION COMPLETE - - -

OUTPUT FILENAME = OUTPUT.FIL

Table 5.9 (Concluded)

## Section 6.0

### OUTPUT

This section presents the results of the two sample cases used as examples of the input requirements in Section 5.0. Input for the first case is presented in Section 5.4 while Section 5.6 contains input requirements for the second case. Output for these examples is shown here for a typical Shuttle reentry trajectory. These structures shown in Figs. 5.1 and 5.2 are not examples of a TPS design but are only presented here to exhibit the EXITS code capabilities.

The first case is the ablator-radiation gap-thin skin-Z standoff structure shown in Fig. 5.1. Results are presented in Table 6.1 for this case. The output is divided into Sections A, B, C . . . H for description purposes.

Section A in Table 6.1 shows the parameters, flags, and time controls for this case. These values are either set during the interactive input or default values are used. A description of these variables is given below:

TSTART	- Initial time
TSTOP	- End time
TIMPT	- Time between printouts
DTIM	- Parameter which controls node spacing for slab or ablator structures
STAB	- Maximum allowable time step is divided by this number to assure stability
TOL	- Convergence criteria for equivalent conductivity calculations
BET	- Relaxation factor for iteration scheme used to compute equivalent conductivity
NBP	- Number of body points
NEXT	- Number of time steps between calculation of new conductor and capacitor values
NSTP	- Maximum number of time steps
IPFLAG	- Flag for printing conductor and capacitor values.

Section B presents the thermophysical property values used in the analysis. Only the properties for the materials used are shown here. Values for density, specific heat, conductivity, emissivity, and for an ablator material, sublima-

tion temperature and heat of ablation are given.

Section C presents the LANMIN generated environment for the body point specified. Values for film coefficient, recovery enthalpy, and pressure are given.

Section D shows the node positions and numbering sequence, structure type, material and conductor number of the network. Initial temperature, sink temperature and the view factor to the sink is also shown.

Section E depicts a graphic representation of the model including the node spacing and materials. A double horizontal dashed line separates the structure types. Node locations are represented by an "O" on the left hand side.

Section F presents the temperature and load histories of the structure beginning at initial time. At each output, the total number of time steps and the value of the last time step taken are shown. Next, the integrated heat loads and heat rates are presented along with the net heat into and out of the structure. Surface recession and recession rates are shown. The temperature at each node within the structure and the node location,  $XX = 0.0$  being the initial surface, is given.

Section G is presented each time a node is dropped from the network as the surface recedes. The same information is contained here as in Section E.

Section H gives the unit mass of the TPS and a message if a temperature has exceeded a material limit as specified in the material property tables.

Output for the second example is presented in Table 6.2. A description of this case is not necessary due to its similarity to the first example.

TSTART =	0.000	TSTOP =	1410.000	TINPT =	100.000	IPFLAG = 1
DTIM =	10.000	STAB =	2.000	TOL =	0.001	
NRP =	1	NEXT =	20	NSTR =	3000	

(A)

## TABLES

(B)

B-BTO.CORK - MAT NO. 28

MAXIMUM TEMPERATURE . . . 760.40 DEG F

TEMP. (DEG F)	DENSITY (LBM/CU.FT)
------------------	------------------------

-0.4596E+03	0.3100E+02
0.9540E+04	0.3100E+02

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TEMP. (DEG F)	SPECIFIC HT. (BTU/LBM-DEG F)
------------------	---------------------------------

-0.4596E+03	0.4600E+00
0.9540E+04	0.4600E+00

TEMP. (DEG F)	CONDUCTIVITY (BTU/FT-S-DEG F)
------------------	----------------------------------

-0.4596E+03	0.1110E-04
0.4004E+03	0.1110E-04
0.8504E+03	0.3330E-05
0.9540E+04	0.3330E-05

TEMP. (DEG F)	EMISSIVITY (DIMENSIONLESS)
------------------	-------------------------------

-0.4596E+03	0.6000E+00
0.9540E+04	0.6000E+00

PRESSURE (LB/SQ.FT)	SUB TEMP (DEG F)
------------------------	---------------------

0.0000E+00	0.7604E+03
0.5000E+04	0.7604E+03

PRESSURE (LB/SQ.FT)	HEAT ABL. (BTU/LBM)
------------------------	------------------------

0.0000E+00	0.7000E+04
0.5000E+04	0.7000E+04

Table 6.1 Output For Example Case One (Table 5.7)

CTD.COLUMB - MAT NO. 10

MAXIMUM TEMPERATURE 2500.40 DEG F

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TEMP. DENSITY  
(DEG F) (LBM/CU.FT)

-0.4596E+03 0.5620E+03  
0.9540E+04 0.5620E+03

TEMP. SPECIFIC HEAT  
(DEG F) (BTU/LBM-DEG F)

-0.4596E+03 0.9900E-01  
0.4000E+00 0.9900E-01  
0.2004E+03 0.6100E-01  
0.6004E+03 0.6500E-01  
0.1000E+04 0.6500E-01  
0.9540E+04 0.6500E-01

TEMP. CONDUCTIVITY  
(DEG F) (BTU/FT-S-DEG F)

-0.4596E+03 0.4400E-02  
0.7040E+02 0.4400E-02  
0.5004E+03 0.6100E-02  
0.1000E+04 0.7300E-02  
0.2000E+04 0.8000E-02  
0.9540E+04 0.8000E-02

TEMP. EMISSIVITY  
(DEG F) (DIMENSIONLESS)

-0.4596E+03 0.1900E+00  
0.2700E+04 0.1900E+00  
0.3600E+04 0.2400E+00  
0.9540E+04 0.2400E+00

AL.7075-T6 - MAT NO. 1

MAXIMUM TEMPERATURE 200.40 DEG F

TEMP. DENSITY  
(DEG F) (LBM/CU.FT)

-0.4596E+03 0.1750E+03  
0.9540E+04 0.1750E+03

Table 6.1 (Continued)

TEMP. (DEG F)	SPECIFIC HEAT (BTU/LBM-DEG F)
-0.4596E+03	0.1700E+00
-0.1496E+03	0.1700E+00
0.4000E+00	0.1950E+00
0.2004E+03	0.2100E+00
0.8604E+03	0.2750E+00
0.1000E+04	0.2750E+00
0.9540E+04	0.2750E+00

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TEMP. (DEG F)	CONDUCTIVITY (BTU/FT-S-DEG F)
-0.4596E+03	0.1400E-01
-0.1496E+03	0.1400E-01
0.4000E+00	0.2000E-01
0.3004E+03	0.2500E-01
0.4004E+03	0.2700E-01
0.5004E+03	0.2900E-01

TEMP. (DEG F)	EMISSIVITY (DIMENSIONLESS)
-0.4596E+03	0.1200E+00
0.9540E+04	0.1200E+00

#### INCONEL 617 - MAT NO. 17

MAXIMUM TEMPERATURE 1800.40 DEG F

TEMP. (DEG F)	DENSITY (LBM/CU.FT)
-0.4596E+03	0.5219E+03
0.9540E+04	0.5219E+03

TEMP. (DEG F)	SPECIFIC HEAT (BTU/LBM-DEG F)
-0.4596E+03	0.1000E+00
0.7840E+02	0.1000E+00
0.2004E+03	0.1040E+00
0.4004E+03	0.1110E+00
0.6004E+03	0.1170E+00
0.1000E+04	0.1310E+00
0.1200E+04	0.1370E+00
0.1400E+04	0.1440E+00
0.1600E+04	0.1500E+00
0.1800E+04	0.1570E+00
0.2000E+04	0.1630E+00
0.9540E+04	0.1630E+00

Table 6.1 (Continued)

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TEMP. CONDUCTIVITY  
(DEG F) (BTU/FT-S-DEG F)

-0.4596E+03	0.2176E-02
0.7840E+02	0.2176E-02
0.2004E+03	0.2339E-02
0.4004E+03	0.2616E-02
0.6004E+03	0.2894E-02
0.1000E+04	0.3449E-02
0.1200E+04	0.3727E-02
0.1400E+04	0.4005E-02
0.1600E+04	0.4282E-02
0.1800E+04	0.4560E-02
0.2000E+04	0.4838E-02
0.9340E+04	0.4838E-02

TEMP. EMISSIVITY  
(DEG F) (DIMENSIONLESS)

-0.4596E+03	0.1500E+00
0.9340E+04	0.1900E+00

Table 6.1 (Continued)

## BODY POINT NUMBER = 3 SAMPLE MINIVER INPUT TABLE

(C)

TIME (SEC)	FILM COEF. (LBM/50.FT-SEC)	REC ENTHALPY (BTU/LBM)	PRESSURE (LBF/50.FT)
0.0000E+00	0.6490E-05	0.1124E+05	0.1252E-01
0.5000E+02	0.1221E-04	0.1124E+05	0.3449E-01
0.1000E+03	0.2502E-04	0.1123E+05	0.1090E+00
0.1250E+03	0.3631E-04	0.1123E+05	0.1992E+00
0.1500E+03	0.5367E-04	0.1122E+05	0.3777E+00
0.1750E+03	0.8009E-04	0.1123E+05	0.7313E+00
0.2000E+03	0.1203E-03	0.1124E+05	0.1440E+01
0.2250E+03	0.1799E-03	0.1127E+05	0.2802E+01
0.2750E+03	0.2529E-03	0.1113E+05	0.7749E+01
0.3000E+03	0.3019E-03	0.1109E+05	0.1124E+02
0.3500E+03	0.3710E-03	0.1092E+05	0.1779E+02
0.4000E+03	0.3974E-03	0.1060E+05	0.2004E+02
0.4500E+03	0.4108E-03	0.1027E+05	0.2121E+02
0.5000E+03	0.4289E-03	0.9966E+04	0.2314E+02
0.5280E+03	0.4325E-03	0.9767E+04	0.2379E+02
0.5560E+03	0.4402E-03	0.9577E+04	0.2470E+02
0.5980E+03	0.4519E-03	0.9270E+04	0.2599E+02
0.6400E+03	0.4661E-03	0.8966E+04	0.2804E+02
0.6540E+03	0.5320E-03	0.8876E+04	0.2915E+02
0.6820E+03	0.6853E-03	0.8680E+04	0.3124E+02
0.7100E+03	0.7182E-03	0.8480E+04	0.3105E+02
0.7380E+03	0.8937E-03	0.8228E+04	0.3303E+02
0.7520E+03	0.1013E-02	0.8106E+04	0.3425E+02
0.7660E+03	0.1145E-02	0.7984E+04	0.3555E+02
0.7800E+03	0.1328E-02	0.7863E+04	0.3739E+02
0.7940E+03	0.1540E-02	0.7738E+04	0.3927E+02
0.8080E+03	0.1784E-02	0.7610E+04	0.4128E+02
0.8220E+03	0.2090E-02	0.7476E+04	0.4368E+02
0.8500E+03	0.2389E-02	0.7149E+04	0.4738E+02
0.8780E+03	0.2615E-02	0.6779E+04	0.5542E+02
0.9060E+03	0.2894E-02	0.6385E+04	0.6202E+02
0.9760E+03	0.3470E-02	0.5284E+04	0.7639E+02
0.1004E+04	0.3900E-02	0.4800E+04	0.8486E+02
0.1032E+04	0.4267E-02	0.4291E+04	0.8984E+02
0.1060E+04	0.4461E-02	0.3797E+04	0.9285E+02
0.1074E+04	0.4484E-02	0.3562E+04	0.9301E+02
0.1102E+04	0.4499E-02	0.3117E+04	0.9302E+02
0.1116E+04	0.4557E-02	0.2903E+04	0.9263E+02
0.1144E+04	0.4844E-02	0.2496E+04	0.9407E+02
0.1172E+04	0.5133E-02	0.2125E+04	0.9569E+02
0.1200E+04	0.5153E-02	0.1813E+04	0.9244E+02
0.1260E+04	0.5462E-02	0.1253E+04	0.8645E+02
0.1290E+04	0.5814E-02	0.1034E+04	0.8521E+02
0.1330E+04	0.6301E-02	0.6930E+03	0.8364E+02
0.1380E+04	0.6169E-02	0.5627E+03	0.8032E+02
0.1410E+04	0.5844E-02	0.4545E+03	0.7643E+02

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Table 6.1 (Continued)

## STRUCTURE DEFINITION

BODY POINT 3  
TINIT = 100.00 DEG F TBLNK = 0.00 DEG F FIJ = 1.000

(D)

NODE NUMBER = 1 DISTANCE FROM SURFACE = 0.00000E+00 IN.  
CONDUCTOR NUMBER = 1

STRUCTURE TYPE = 7 ABLATOR SUBLINER

MATERIAL 1 = B-STG.CORK

NODE NUMBER = 2 DISTANCE FROM SURFACE = 0.416667E-01 IN.

NODE NUMBER = 2 DISTANCE FROM SURFACE = 0.416667E-01 IN.  
CONDUCTOR NUMBER = 2

STRUCTURE TYPE = 7 ABLATOR SUBLINER

MATERIAL 1 = B-STG.CORK

NODE NUMBER = 3 DISTANCE FROM SURFACE = 0.633333E-01 IN.

NODE NUMBER = 3 DISTANCE FROM SURFACE = 0.633333E-01 IN.  
CONDUCTOR NUMBER = 3

STRUCTURE TYPE = 7 ABLATOR SUBLINER

MATERIAL 1 = B-STG.CORK

NODE NUMBER = 4 DISTANCE FROM SURFACE = 0.125000E+00 IN.

NODE NUMBER = 4 DISTANCE FROM SURFACE = 0.125000E+00 IN.  
CONDUCTOR NUMBER = 4

STRUCTURE TYPE = 7 ABLATOR SUBLINER

MATERIAL 1 = B-STG.CORK

NODE NUMBER = 5 DISTANCE FROM SURFACE = 0.166667E+00 IN.

NODE NUMBER = 5 DISTANCE FROM SURFACE = 0.166667E+00 IN.  
CONDUCTOR NUMBER = 5

STRUCTURE TYPE = 7 ABLATOR SUBLINER

MATERIAL 1 = B-STG.CORK

NODE NUMBER = 6 DISTANCE FROM SURFACE = 0.208333E+00 IN.

NODE NUMBER = 6 DISTANCE FROM SURFACE = 0.208333E+00 IN.  
CONDUCTOR NUMBER = 6

STRUCTURE TYPE = 7 ABLATOR SUBLINER

MATERIAL 1 = B-STG.CORK

NODE NUMBER = 7 DISTANCE FROM SURFACE = 0.250000E+00 IN.

NODE NUMBER = 7 DISTANCE FROM SURFACE = 0.250000E+00 IN.  
CONDUCTOR NUMBER = 7

STRUCTURE TYPE = 7 ABLATOR SUBLINER

MATERIAL 1 = B-STG.CORK

NODE NUMBER = 8 DISTANCE FROM SURFACE = 0.291667E+00 IN.

Table 6.1 (Continued)

NODE NUMBER = 8 DISTANCE FROM SURFACE = 0.291667E+00 IN.  
CONDUCTOR NUMBER = 8  
STRUCTURE TYPE = 7 ABLATOR SUBLIMER  
MATERIAL 1 = B-STG.CORK  
NODE NUMBER = 9 DISTANCE FROM SURFACE = 0.333333E+00 IN.

ORIGINAL DRAWING  
OF POOR QUALITY

NODE NUMBER = 9 DISTANCE FROM SURFACE = 0.333333E+00 IN.  
CONDUCTOR NUMBER = 9  
STRUCTURE TYPE = 7 ABLATOR SUBLIMER  
MATERIAL 1 = B-STG.CORK  
NODE NUMBER = 10 DISTANCE FROM SURFACE = 0.375000E+00 IN.

NODE NUMBER = 10 DISTANCE FROM SURFACE = 0.375000E+00 IN.  
CONDUCTOR NUMBER = 10  
STRUCTURE TYPE = 7 ABLATOR SUBLIMER  
MATERIAL 1 = B-STG.CORK  
NODE NUMBER = 11 DISTANCE FROM SURFACE = 0.416667E+00 IN.

NODE NUMBER = 11 DISTANCE FROM SURFACE = 0.416667E+00 IN.  
CONDUCTOR NUMBER = 11  
STRUCTURE TYPE = 7 ABLATOR SUBLIMER  
MATERIAL 1 = B-STG.CORK  
NODE NUMBER = 12 DISTANCE FROM SURFACE = 0.458333E+00 IN.

NODE NUMBER = 12 DISTANCE FROM SURFACE = 0.458333E+00 IN.  
CONDUCTOR NUMBER = 12  
STRUCTURE TYPE = 7 ABLATOR SUBLIMER  
MATERIAL 1 = B-STG.CORK  
NODE NUMBER = 13 DISTANCE FROM SURFACE = 0.500000E+00 IN.

NODE NUMBER = 13 DISTANCE FROM SURFACE = 0.500000E+00 IN.

CONDUCTOR NUMBER = 13  
STRUCTURE TYPE = 7 ABLATOR SUBLIMER  
MATERIAL 1 = B-STG.CORK  
NODE NUMBER = 14 DISTANCE FROM SURFACE = 0.541667E+00 IN.

NODE NUMBER = 14 DISTANCE FROM SURFACE = 0.541667E+00 IN.  
CONDUCTOR NUMBER = 14  
STRUCTURE TYPE = 7 ABLATOR SUBLIMER  
MATERIAL 1 = B-STG.CORK  
NODE NUMBER = 15 DISTANCE FROM SURFACE = 0.583333E+00 IN.

NODE NUMBER = 15 DISTANCE FROM SURFACE = 0.583333E+00 IN.  
CONDUCTOR NUMBER = 15  
STRUCTURE TYPE = 7 ABLATOR SUBLIMER  
MATERIAL 1 = B-STG.CORK  
NODE NUMBER = 16 DISTANCE FROM SURFACE = 0.625000E+00 IN.

Table 6.1 (Continued)

ORIGINAL PAGE IS  
OF POOR QUALITY

NODE NUMBER = 16	DISTANCE FROM SURFACE =	0.625000E+00 IN.
CONDUCTOR NUMBER = 16		
STRUCTURE TYPE = 7	ABLATOR SUBLIMER	
MATERIAL 1 = B-STG.CORK		
NODE NUMBER = 17	DISTANCE FROM SURFACE =	0.666667E+00 IN.
CONDUCTOR NUMBER = 17		
STRUCTURE TYPE = 7	ABLATOR SUBLIMER	
MATERIAL 1 = B-STG.CORK		
NODE NUMBER = 18	DISTANCE FROM SURFACE =	0.708334E+00 IN.
CONDUCTOR NUMBER = 18		
STRUCTURE TYPE = 7	ABLATOR SUBLIMER	
MATERIAL 1 = B-STG.CORK		
NODE NUMBER = 19	DISTANCE FROM SURFACE =	0.750000E+00 IN.
CONDUCTOR NUMBER = 19		
STRUCTURE TYPE = 7	ABLATOR SUBLIMER	
MATERIAL 1 = B-STG.CORK		
NODE NUMBER = 20	DISTANCE FROM SURFACE =	0.791667E+00 IN.
CONDUCTOR NUMBER = 20		
STRUCTURE TYPE = 7	ABLATOR SUBLIMER	
MATERIAL 1 = B-STG.CORK		
NODE NUMBER = 21	DISTANCE FROM SURFACE =	0.833334E+00 IN.
CONDUCTOR NUMBER = 21		
STRUCTURE TYPE = 7	ABLATOR SUBLIMER	
MATERIAL 1 = B-STG.CORK		
NODE NUMBER = 22	DISTANCE FROM SURFACE =	0.875000E+00 IN.
CONDUCTOR NUMBER = 22		
STRUCTURE TYPE = 7	ABLATOR SUBLIMER	
MATERIAL 1 = B-STG.CORK		
NODE NUMBER = 23	DISTANCE FROM SURFACE =	0.916667E+00 IN.
CONDUCTOR NUMBER = 23		
STRUCTURE TYPE = 7	ABLATOR SUBLIMER	
MATERIAL 1 = B-STG.CORK		
NODE NUMBER = 24	DISTANCE FROM SURFACE =	0.958334E+00 IN.

Table 6.1 (Continued)

NODE NUMBER = 24 DISTANCE FROM SURFACE = 0.95834E+00 IN. ORIGINAL PAGE IS  
 CONDUCTOR NUMBER = 24 OF POOR QUALITY  
 STRUCTURE TYPE = 7 ABLATOR SUBLINER  
 MATERIAL 1 = B-STO.CORK  
 NODE NUMBER = 25 DISTANCE FROM SURFACE = 0.100000E+01 IN.

NODE NUMBER = 25 DISTANCE FROM SURFACE = 0.100000E+01 IN.  
 CONDUCTOR NUMBER = 25  
 STRUCTURE TYPE = 2 RADIATION GAP  
 MATERIAL 1 = CTD.COLUMN  
 MATERIAL 2 = AL.7075-T6  
 NODE NUMBER = 26 DISTANCE FROM SURFACE = 0.250000E+01 IN.

NODE NUMBER = 26 DISTANCE FROM SURFACE = 0.250000E+01 IN.  
 CONDUCTOR NUMBER = 26  
 STRUCTURE TYPE = 6 THIN SKIN  
 MATERIAL 1 = AL.7075-T6  
 NODE NUMBER = 27 DISTANCE FROM SURFACE = 0.280000E+01 IN.

NODE NUMBER = 27 DISTANCE FROM SURFACE = 0.280000E+01 IN.  
 CONDUCTOR NUMBER = 27  
 STRUCTURE TYPE = 3 Z STANDOFF  
 MATERIAL 1 = INCONI 617  
 MATERIAL 2 = CTD.COLUMN  
 MATERIAL 3 = AL.7075-T6  
 NODE NUMBER = 28 DISTANCE FROM SURFACE = 0.480000E+01 IN.

Table 6.1 (Continued)

THIS IS THE CONFIGURATION FOR BODY PT. 3

ORIGINAL PAGE IS  
OF POOR QUALITY.

(E)

--- 1. 0		
2. 0	I	
3. 0	I	
4. 0	I	
5. 0	I	
6. 0	I	
7. 0	I	
8. 0	I	
9. 0	I	
10. 0	I	
11. 0	I	
12. 0	I	
13. 0	B-STG.CORK ABLATOR SUBLIMER	1.00000 IN.
14. 0	I	
15. 0	I	
16. 0	I	
17. 0	I	
18. 0	I	
19. 0	I	
20. 0	I	
21. 0	I	
22. 0	I	
23. 0	I	
24. 0	I	
--- 25. 0	0.125000 IN. CTD.COLUMN	
	RADIATION GAP	1.50000 IN.
	0.100000 IN. AL.7075-T6	
--- 26. 0	AL.7075-T6 THIN SKIN	0.30000 IN.
--- 27. 0	0.140000 IN. INCONL 617	
222222	222222	
2	2	
2	2 AL.7075-T6 2 STANDOFF	2.00000 IN.
2	2	
222222	222222	
	0.180000 IN. CTD.COLUMN	
--- 28. 0		

Table 6.1 (Continued)

TIME = 0.00000 TIME STEP = 0.00000 NO. OF STEPS = 0

(F)

INTEGRATED HEAT  
BTU/SQ.FT

HEAT RATES  
BTU/SQ.FT-SEC

CONVECTED	0.0	CONVECTED	0.0
RADIATED	0.0	RADIATED	0.0
NET LOAD		NET LOAD	
STORED	0.0	STORED	0.0
SUBLINED	0.0	SUBLINED	0.0
ADVECTED	0.0	ADVECTED	0.0
TPS NET	0.0	TPS NET	0.0

ORIGINAL DATA IS  
OF POOR QUALITY

#### SURFACE RECEDSION

DISTANCE 0.00000 IN. TEMPERATURE DEG F

RECEDSION RATE 0.00000 IN/SEC

T( 1)= 100.000	T( 2)= 100.000	T( 3)= 100.000	T( 4)= 100.000	T( 5)= 100.000
T( 6)= 100.000	T( 7)= 100.000	T( 8)= 100.000	T( 9)= 100.000	T( 10)= 100.000
T( 11)= 100.000	T( 12)= 100.000	T( 13)= 100.000	T( 14)= 100.000	T( 15)= 100.000
T( 16)= 100.000	T( 17)= 100.000	T( 18)= 100.000	T( 19)= 100.000	T( 20)= 100.000
T( 21)= 100.000	T( 22)= 100.000	T( 23)= 100.000	T( 24)= 100.000	T( 25)= 100.000
T( 26)= 100.000	T( 27)= 100.000	T( 28)= 100.000		

#### NODE POSITION INCHES

XX( 1)= 0.000	XX( 2)= 0.042	XX( 3)= 0.083	XX( 4)= 0.125	XX( 5)= 0.167
XX( 6)= 0.208	XX( 7)= 0.250	XX( 8)= 0.292	XX( 9)= 0.333	XX( 10)= 0.375
XX( 11)= 0.417	XX( 12)= 0.458	XX( 13)= 0.500	XX( 14)= 0.542	XX( 15)= 0.583
XX( 16)= 0.625	XX( 17)= 0.667	XX( 18)= 0.708	XX( 19)= 0.750	XX( 20)= 0.792
XX( 21)= 0.833	XX( 22)= 0.875	XX( 23)= 0.917	XX( 24)= 0.958	XX( 25)= 1.000
XX( 26)= 2.900	XX( 27)= 2.900	XX( 28)= 4.800		

TIME = 100.00000 TIME STEP = 2.04209 NO. OF STEPS = 28

INTEGRATED HEAT  
BTU/SQ.FT

HEAT RATES  
BTU/SQ.FT-SEC

CONVECTED	19.1	CONVECTED	0.3
RADIATED	3.8	RADIATED	0.1
NET LOAD		NET LOAD	
STORED	11.3	STORED	0.2
SUBLINED	0.0	SUBLINED	0.0
ADVECTED	0.0	ADVECTED	0.0
TPS NET	11.3	TPS NET	0.2

#### SURFACE RECEDSION

DISTANCE 0.00000 IN. TEMPERATURE DEG F

RECEDSION RATE 0.00000 IN/SEC

T( 1)= 226.226	T( 2)= 174.707	T( 3)= 142.714	T( 4)= 123.583	T( 5)= 112.544
T( 6)= 106.400	T( 7)= 103.114	T( 8)= 101.435	T( 9)= 100.623	T( 10)= 100.233
T( 11)= 100.096	T( 12)= 100.034	T( 13)= 100.011	T( 14)= 100.003	T( 15)= 100.001
T( 16)= 100.000	T( 17)= 100.000	T( 18)= 100.000	T( 19)= 100.000	T( 20)= 100.000
T( 21)= 100.000	T( 22)= 100.000	T( 23)= 100.000	T( 24)= 100.000	T( 25)= 100.000
T( 26)= 100.000	T( 27)= 100.000	T( 28)= 100.000		

Table 6.1 (Continued)

NODE POSITION INCHES

XX( 1)= 0.000	XX( 2)= 0.042	XX( 3)= 0.083	XX( 4)= 0.125	XX( 5)= 0.167
XX( 6)= 0.208	XX( 7)= 0.250	XX( 8)= 0.292	XX( 9)= 0.333	XX( 10)= 0.375
XX( 11)= 0.417	XX( 12)= 0.458	XX( 13)= 0.500	XX( 14)= 0.542	XX( 15)= 0.583
XX( 16)= 0.625	XX( 17)= 0.667	XX( 18)= 0.708	XX( 19)= 0.750	XX( 20)= 0.792
XX( 21)= 0.833	XX( 22)= 0.875	XX( 23)= 0.917	XX( 24)= 0.958	XX( 25)= 1.000
XX( 26)= 2.900	XX( 27)= 2.800	XX( 28)= 4.800		

TIME = 200.00000 TIME STEP = 0.47696 NO. OF STEPS = 98

INTEGRATED HEAT  
BTU/SQ.FT

HEAT RATES  
BTU/SQ.FT-SEC

CONVECTED	80.2	CONVECTED	1.3
RADIATED	24.0	RADIATED	0.6
NET LOAD	56.2	NET LOAD	0.7
STORED	56.2	STORED	0.7
SUBLINED	0.0	SUBLINED	0.0
ADVECTED	0.0	ADVECTED	0.0
TPS NET	56.2	TPS NET	0.7

ORIGINAL PAGE IS  
OF POOR QUALITY

#### SURFACE RECESSION

DISTANCE 0.00000 IN.  
TEMPERATURE 62.3 F

RECESSION RATE 0.00000 IN/SEC

T( 1)= 695.963	T( 2)= 428.034	T( 3)= 307.611	T( 4)= 229.010	T( 5)= 178.921
T( 6)= 147.377	T( 7)= 128.266	T( 8)= 116.533	T( 9)= 109.504	T( 10)= 103.258
T( 11)= 102.956	T( 12)= 101.393	T( 13)= 100.837	T( 14)= 100.428	T( 15)= 100.212
T( 16)= 100.102	T( 17)= 100.048	T( 18)= 100.022	T( 19)= 100.009	T( 20)= 100.004
T( 21)= 100.002	T( 22)= 100.001	T( 23)= 100.000	T( 24)= 100.000	T( 25)= 100.000
T( 26)= 100.000	T( 27)= 100.000	T( 28)= 100.000		

#### NODE POSITION INCHES

XX( 1)= 0.000	XX( 2)= 0.042	XX( 3)= 0.083	XX( 4)= 0.125	XX( 5)= 0.167
XX( 6)= 0.208	XX( 7)= 0.250	XX( 8)= 0.292	XX( 9)= 0.333	XX( 10)= 0.375
XX( 11)= 0.417	XX( 12)= 0.458	XX( 13)= 0.500	XX( 14)= 0.542	XX( 15)= 0.583
XX( 16)= 0.625	XX( 17)= 0.667	XX( 18)= 0.708	XX( 19)= 0.750	XX( 20)= 0.792
XX( 21)= 0.833	XX( 22)= 0.875	XX( 23)= 0.917	XX( 24)= 0.958	XX( 25)= 1.000
XX( 26)= 2.900	XX( 27)= 2.800	XX( 28)= 4.800		

TIME = 300.00000 TIME STEP = 1.47916 NO. OF STEPS = 85

INTEGRATED HEAT  
BTU/SQ.FT

HEAT RATES  
BTU/SQ.FT-SEC

CONVECTED	310.4	CONVECTED	3.2
RADIATED	104.4	RADIATED	0.8
NET LOAD	206.0	NET LOAD	2.4
STORED	103.8	STORED	0.4
SUBLINED	101.3	SUBLINED	2.0
ADVECTED	0.9	ADVECTED	0.0
TPS NET	206.0	TPS NET	2.4

Table 6.1 (Continued)

SURFACE RECESSION

DISTANCE 0.00564 IN.  
TEMPERATURE DEG F

T( 1)= 760.400	T( 1)= 587.613	T( 3)= 468.664	T( 4)= 378.480	T( 5)= 303.101
T( 6)= 246.091	T( 7)= 200.789	T( 8)= 167.511	T( 9)= 144.023	T( 10)= 126.021
T( 11)= 117.454	T( 12)= 110.662	T( 13)= 106.399	T( 14)= 103.777	T( 15)= 102.194
T( 16)= 101.254	T( 17)= 100.709	T( 18)= 100.390	T( 19)= 100.212	T( 20)= 100.113
T( 21)= 100.039	T( 22)= 100.030	T( 23)= 100.018	T( 24)= 100.006	T( 25)= 100.001
T( 26)= 100.000	T( 27)= 100.000	T( 28)= 100.000		

NODE POSITION INCHES

XX( 1)= 0.006	XX( 2)= 0.044	XX( 3)= 0.083	XX( 4)= 0.125	XX( 5)= 0.167
XX( 6)= 0.208	XX( 7)= 0.250	XX( 8)= 0.292	XX( 9)= 0.333	XX( 10)= 0.375
XX( 11)= 0.417	XX( 12)= 0.458	XX( 13)= 0.500	XX( 14)= 0.542	XX( 15)= 0.583
XX( 16)= 0.625	XX( 17)= 0.667	XX( 18)= 0.708	XX( 19)= 0.750	XX( 20)= 0.792
XX( 21)= 0.833	XX( 22)= 0.875	XX( 23)= 0.917	XX( 24)= 0.958	XX( 25)= 1.000
XX( 26)= 2.500	XX( 27)= 2.600	XX( 28)= 4.800		

TIME = 400.00000 TIME STEP = 1.03931 NO. OF STEPS = 120

INTEGRATED HEAT

BTU/SQ.FT

CONNECTED	690.2	CONNECTED	4.1
RADIATED	187.1	RADIATED	0.8
NET LOAD	503.1	NET LOAD	3.3
STORED	135.7	STORED	0.3
SUBLINED	364.8	SUBLINED	3.0
ADVECTED	2.6	ADVECTED	0.0
TPS NET	503.1	TPS NET	3.3

SURFACE RECESSION

DISTANCE 0.02025 IN.  
TEMPERATURE DEG F

T( 1)= 760.400	T( 2)= 642.650	T( 3)= 539.294	T( 4)= 448.191	T( 5)= 378.192
T( 6)= 318.522	T( 7)= 268.019	T( 8)= 226.433	T( 9)= 193.107	T( 10)= 167.105
T( 11)= 147.345	T( 12)= 132.713	T( 13)= 122.146	T( 14)= 114.700	T( 15)= 109.573
T( 16)= 106.122	T( 17)= 103.847	T( 18)= 102.377	T( 19)= 101.448	T( 20)= 100.864
T( 21)= 100.508	T( 22)= 100.291	T( 23)= 100.160	T( 24)= 100.078	T( 25)= 100.020
T( 26)= 100.000	T( 27)= 100.000	T( 28)= 100.000		

NODE POSITION INCHES

XX( 1)= 0.006	XX( 2)= 0.046	XX( 3)= 0.083	XX( 4)= 0.125	XX( 5)= 0.167
XX( 6)= 0.208	XX( 7)= 0.250	XX( 8)= 0.292	XX( 9)= 0.333	XX( 10)= 0.375
XX( 11)= 0.417	XX( 12)= 0.458	XX( 13)= 0.500	XX( 14)= 0.542	XX( 15)= 0.583
XX( 16)= 0.625	XX( 17)= 0.667	XX( 18)= 0.708	XX( 19)= 0.750	XX( 20)= 0.792
XX( 21)= 0.833	XX( 22)= 0.875	XX( 23)= 0.917	XX( 24)= 0.958	XX( 25)= 1.000
XX( 26)= 2.500	XX( 27)= 2.600	XX( 28)= 4.800		

Table 6.1 (Continued)

TIME = 500.00000 TIME STEP = 0.73596 NO. OF STEPS = 105

ORIGINAL PAGE IS  
OF POOR QUALITY

INTEGRATED HEAT  
BTU/SQ.FT

HEAT RATES  
BTU/SQ.FT-SEC

CONNECTED	1100.6	CONNECTED	4.1
RADIATED	269.8	RADIATED	0.8
NET LOAD	830.8	NET LOAD	3.3
STORED	163.9	STORED	0.3
SUBLINED	663.1	SUBLINED	3.0
ADVECTED	3.8	ADVECTED	0.0
TPS NET	830.8	TPS NET	3.3

SURFACE RECESSION

DISTANCE 0.03672 IN.

RECESSION RATE 0.00016 IN/SEC

TEMPERATURE DEG F

T( 1)= 760.400	T( 2)= 689.840	T( 3)= 598.048	T( 4)= 502.372	T( 5)= 429.213
T( 6)= 370.571	T( 7)= 319.121	T( 8)= 274.774	T( 9)= 237.242	T( 10)= 206.067
T( 11)= 180.663	T( 12)= 160.333	T( 13)= 144.426	T( 14)= 132.173	T( 15)= 122.924
T( 16)= 116.072	T( 17)= 111.089	T( 18)= 107.530	T( 19)= 105.032	T( 20)= 103.307
T( 21)= 102.131	T( 22)= 101.338	T( 23)= 100.800	T( 24)= 100.426	T( 25)= 100.144
T( 26)= 100.000	T( 27)= 100.000	T( 28)= 100.000		

NODE POSITION INCHES

XX( 1)= 0.037	XX( 2)= 0.054	XX( 3)= 0.083	XX( 4)= 0.129	XX( 5)= 0.167
XX( 6)= 0.208	XX( 7)= 0.250	XX( 8)= 0.292	XX( 9)= 0.333	XX( 10)= 0.375
XX( 11)= 0.417	XX( 12)= 0.458	XX( 13)= 0.500	XX( 14)= 0.542	XX( 15)= 0.583
XX( 16)= 0.625	XX( 17)= 0.667	XX( 18)= 0.708	XX( 19)= 0.750	XX( 20)= 0.792
XX( 21)= 0.833	XX( 22)= 0.875	XX( 23)= 0.917	XX( 24)= 0.958	XX( 25)= 1.000
XX( 26)= 2.500	XX( 27)= 2.600	XX( 28)= 4.800		

TIME = 600.00000 TIME STEP = 0.14777 NO. OF STEPS = 421

INTEGRATED HEAT  
BTU/SQ.FT

HEAT RATES  
BTU/SQ.FT-SEC

CONNECTED	1504.3	CONNECTED	4.1
RADIATED	352.4	RADIATED	0.8
NET LOAD	1156.9	NET LOAD	3.2
STORED	190.2	STORED	0.3
SUBLINED	962.3	SUBLINED	3.0
ADVECTED	4.4	ADVECTED	0.0
TPS NET	1156.9	TPS NET	3.2

SURFACE RECESSION

DISTANCE 0.03322 IN.

RECESSION RATE 0.00017 IN/SEC

TEMPERATURE DEG F

T( 1)= 760.400	T( 2)= 734.636	T( 3)= 692.931	T( 4)= 550.040	T( 5)= 473.121
T( 6)= 411.593	T( 7)= 360.579	T( 8)= 315.170	T( 9)= 275.544	T( 10)= 241.459
T( 11)= 212.566	T( 12)= 189.438	T( 13)= 166.590	T( 14)= 152.507	T( 15)= 139.670
T( 16)= 129.577	T( 17)= 121.759	T( 18)= 119.788	T( 19)= 111.293	T( 20)= 107.931
T( 21)= 105.492	T( 22)= 103.691	T( 23)= 102.366	T( 24)= 101.368	T( 25)= 100.573
T( 26)= 100.000	T( 27)= 100.000	T( 28)= 100.000		

Table 6.1 (Continued)

## NODE POSITION INCHES

XX( 1)=	0.093	XX( 2)=	0.099	XX( 3)=	0.083	XX( 4)=	0.129	XX( 5)=	0.167
XX( 6)=	0.208	XX( 7)=	0.200	XX( 8)=	0.292	XX( 9)=	0.333	XX( 10)=	0.375
XX( 11)=	0.417	XX( 12)=	0.488	XX( 13)=	0.500	XX( 14)=	0.542	XX( 15)=	0.583
XX( 16)=	0.625	XX( 17)=	0.667	XX( 18)=	0.708	XX( 19)=	0.750	XX( 20)=	0.792
XX( 21)=	0.833	XX( 22)=	0.875	XX( 23)=	0.917	XX( 24)=	0.958	XX( 25)=	1.000
XX( 26)=	2.500	XX( 27)=	2.600	XX( 28)=	4.800				

NODE DROPPED FROM SUBLIMER-ABLATOR MODEL

ORIGINAL PATH, 1%  
OF POOR QUALITY

THIS IS THE CONFIGURATION FOR BODY PT. 3

---1. 0									
2. 0									
3. 0									
4. 0									
5. 0									
6. 0									
7. 0									
8. 0									
9. 0									
10. 0									
11. 0									
12. 0									
13. 0		6 STG.CORK ABLATOR SUBLIMER		0.943296 IN.					
14. 0									
15. 0									
16. 0									
17. 0									
18. 0									
19. 0									
20. 0									
21. 0									
22. 0									
23. 0									
---24. 0		0.125000 IN. CTD.COLUMB							
		RADIATION GAP		1.500000 IN.					
		0.100000 IN. AL.7075-T6							
---25. 0									
		AL.7075-T6 THIN SKIN		0.300000 IN.					
---26. 0									
		0.140000 IN. INCONI 617							
222222	222222								
2	2								
2	2	AL.7075-T6 Z STANDOFF		2.000000 IN.					
2	2								
222222	222222								
		0.160000 IN. CTD.COLUMB							
---27. 0									

Table 6.1 (Continued)  
114

ORIGINAL PAGE IS  
OF POOR QUALITY

TIME = 700.00000 TIME STEP = 0.46820 NO. OF STEPS = 943

INTEGRATED HEAT  
BTU/80.FT

HEAT RATES  
BTU/80.FT-SEC

CONVECTED	1978.8		CONVECTED	5.0
RADIATED	433.1		RADIATED	0.8
NET LOAD		1943.7	NET LOAD	5.0
STORED	214.5		STORED	0.2
SUBLINED	1323.4		SUBLINED	4.7
ADVECTED	9.8		ADVECTED	0.0
TPS NET		1943.7	TPS NET	5.0

#### SURFACE RECESSION

DISTANCE 0.07319 IN. RECESSION RATE 0.00020 IN/SEC

TEMPERATURE DEG F

T( 1)= 760.400	T( 2)= 698.392	T( 3)= 598.511	T( 4)= 514.310	T( 5)= 449.099
T( 6)= 396.113	T( 7)= 350.169	T( 8)= 309.324	T( 9)= 273.413	T( 10)= 242.208
T( 11)= 215.415	T( 12)= 192.687	T( 13)= 173.642	T( 14)= 157.877	T( 15)= 144.984
T( 16)= 134.567	T( 17)= 126.247	T( 18)= 119.674	T( 19)= 114.532	T( 20)= 110.537
T( 21)= 107.441	T( 22)= 105.029	T( 23)= 103.118	T( 24)= 101.547	T( 25)= 100.000
T( 26)= 100.000	T( 27)= 100.000			

NODE POSITION INCHES

XX( 1)= 0.073	XX( 2)= 0.089	XX( 3)= 0.129	XX( 4)= 0.167	XX( 5)= 0.208
XX( 6)= 0.290	XX( 7)= 0.292	XX( 8)= 0.333	XX( 9)= 0.375	XX( 10)= 0.417
XX( 11)= 0.458	XX( 12)= 0.500	XX( 13)= 0.542	XX( 14)= 0.583	XX( 15)= 0.625
XX( 16)= 0.667	XX( 17)= 0.708	XX( 18)= 0.730	XX( 19)= 0.792	XX( 20)= 0.833
XX( 21)= 0.879	XX( 22)= 0.917	XX( 23)= 0.938	XX( 24)= 1.000	XX( 25)= 2.500
XX( 26)= 2.600	XX( 27)= 4.800			

NODE DROPPED FROM SUBLIMER-ABLATOR MODEL

Table 6.1 (Continued)

THIS IS THE CONFIGURATION FOR BODY PT. 3

--- 1. 0			
2. 0			
3. 0			
4. 0			
5. 0			
6. 0			
7. 0			
8. 0			
9. 0			
10. 0			
11. 0			
12. 0	B-STO.CORK ABLATOR SUBLIMER	0.910197 IN.	
13. 0			
14. 0			
15. 0			
16. 0			
17. 0			
18. 0			
19. 0			
20. 0			
21. 0			
22. 0			
--- 23. 0	0.125000 IN. CTD.COLUMB		
--- 24. 0	0.100000 IN. AL.7075-T6	RADIATION GAP 1.500000 IN.	
--- 25. 0	AL.7075-T6 THIN SKIN	0.300000 IN.	
--- 26. 0	0.140000 IN. INCONL 617		
	222222 222222		
	2 2		
	2 2 AL.7075-T6 Z STANDOFF	2.000000 IN.	
	2 2		
	222222 222222		
--- 26. 0	0.150000 IN. CTD.COLUMB		

ORIGINAL PAGE IS  
OF POOR QUALITY

Table 6.1 (Continued)

TIME = 800.00000 TIME STEP = 0.97308 NO. OF STEPS = 711

ORIGINAL PARCELS  
OF POOR QUALITY

INTEGRATED HEAT  
BTU/BQ.FT

		HEAT RATES BTU/BQ.FT-SEC	
CONVECTED	2789.9	CONVECTED	12.0
RADIATED	517.8	RADIATED	0.8
NET LOAD	2272.1	NET LOAD	11.2
STORED	240.3	STORED	0.3
SUBLINED	2024.9	SUBLINED	10.9
TPS NET	8.7	ADVECTED	0.0
	2272.1	TPS NET	11.2

SURFACE RECESSION

DISTANCE 0.11191 IN.

RECESSION RATE 0.00039 IN/SEC

TEMPERATURE DEG F

T( 1)= 760.400	T( 2)= 673.709	T( 3)= 574.758	T( 4)= 493.329	T( 5)= 432.302
T( 6)= 383.468	T( 7)= 340.771	T( 8)= 303.091	T( 9)= 269.926	T( 10)= 241.045
T( 11)= 216.076	T( 12)= 194.683	T( 13)= 176.528	T( 14)= 161.269	T( 15)= 148.569
T( 16)= 138.093	T( 17)= 129.532	T( 18)= 122.561	T( 19)= 116.968	T( 20)= 112.442
T( 21)= 108.777	T( 22)= 109.772	T( 23)= 103.249	T( 24)= 100.002	T( 25)= 100.002

T( 26)= 100.000

NODE POSITION INCHES

XX( 1)= 0.112	XX( 2)= 0.132	XX( 3)= 0.167	XX( 4)= 0.208	XX( 5)= 0.230
XX( 6)= 0.292	XX( 7)= 0.333	XX( 8)= 0.375	XX( 9)= 0.417	XX( 10)= 0.458
XX( 11)= 0.500	XX( 12)= 0.542	XX( 13)= 0.583	XX( 14)= 0.625	XX( 15)= 0.667
XX( 16)= 0.708	XX( 17)= 0.750	XX( 18)= 0.792	XX( 19)= 0.833	XX( 20)= 0.875
XX( 21)= 0.917	XX( 22)= 0.958	XX( 23)= 1.000	XX( 24)= 2.300	XX( 25)= 2.600

XX( 26)= 4.800

Table 6.1 (Continued)

MODE DROPPED FROM SUBLIMER-ABLATOR MODEL

DISCONTINUED DUE TO  
OF POOR QUALITY

THIS IS THE CONFIGURATION FOR BODY PT. 3

--- 1.	0		
2.	0		
3.	0		
4.	0		
5.	0		
6.	0		
7.	0		
8.	0		
9.	0		
10.	0		
11.	0		
12.	0	B-STG.CORK ABLATOR SUBLIMER	0.065148 IN.
13.	0		
14.	0		
15.	0		
16.	0		
17.	0		
18.	0		
19.	0		
20.	0		
21.	0		
--- 22.	0	0.125000 IN. CTD.COLUMB	
		RADIATION GAP	1.900000 IN.
		0.100000 IN. AL.7075-T6	
--- 23.	0	AL.7075-T6 THIN SKIN	
--- 24.	0	0.140000 IN. INCONL 617	
222222	222222		
2	2		
2	2	AL.7075-T6 2 STANDOFF	2.000000 IN.
2	2		
222222	222222		
		0.160000 IN. CTD.COLUMB	
--- 25.	0		

MODE DROPPED FROM SUBLIMER-ABLATOR MODEL

Table 6.1 (Continued)

THIS IS THE CONFIGURATION FOR BODY PT. 3

1. 0			
2. 0			
3. 0			
4. 0			
5. 0			
6. 0			
7. 0			
8. 0			
9. 0			
10. 0			
11. 0	B-STD.CORK ABLATOR SUBLIMER	0.023189 IN.	
12. 0			
13. 0			
14. 0			
15. 0			
16. 0			
17. 0			
18. 0			
19. 0			
20. 0			
21. 0	0.125000 IN. CTD.COLUMN		
		RADIATION GAP	1.500000 IN.
	0.100000 IN. AL.7075-T6		
22. 0			
	AL.7075-T6 THIN SKIN		0.300000 IN.
23. 0			
	0.140000 IN. INCONEL 617		
222222	222222		
2	2		
2	2	AL.7075-T6 2 STANDOFF	2.000000 IN.
2	2		
222222	222222		
	0.180000 IN. CTD.COLUMN		
24. 0			

ORIGINAL PAPER IS  
OF POOR QUALITY

Table 6.1 (Continued)

TIME = 900.00000 TIME STEP = 0.00100 NO. OF STEPS = 877      ORIGINAL MODELS  
 OF POOR QUALITY

INTEGRATED HEAT		HEAT RATES	
BTU/BD.FT		BTU/BD.FT-SEC	
CONNECTED	4369.1	CONNECTED	17.3
RADIATED	600.9	RADIATED	0.8
NET LOAD	3768.6	NET LOAD	16.5
STORED	275.7	STORED	0.4
SUBLINED	3476.9	SUBLINED	16.0
ADVECTED	15.9	ADVECTED	0.1
TPS NET	3768.6	TPS NET	16.5

SURFACE RECEDITION  
 DISTANCE 0.19232 IN.  
 TEMPERATURE DEG F

		RECEDITION RATE	0.00060 IN/SEC	
T( 1)= 760.400	T( 2)= 639.981	T( 3)= 522.308	T( 4)= 440.134	T( 5)= 383.468
T( 6)= 337.984	T( 7)= 299.871	T( 8)= 267.453	T( 9)= 239.678	T( 10)= 215.637
T( 11)= 199.419	T( 12)= 177.999	T( 13)= 163.230	T( 14)= 150.784	T( 15)= 140.399
T( 16)= 131.673	T( 17)= 124.461	T( 18)= 118.479	T( 19)= 113.504	T( 20)= 109.332
T( 21)= 105.780	T( 22)= 100.007	T( 23)= 100.007	T( 24)= 100.000	

NODE POSITION INCHES

XX( 1)= 0.192	XX( 2)= 0.214	XX( 3)= 0.250	XX( 4)= 0.292	XX( 5)= 0.333
XX( 6)= 0.375	XX( 7)= 0.417	XX( 8)= 0.458	XX( 9)= 0.500	XX( 10)= 0.542
XX( 11)= 0.583	XX( 12)= 0.625	XX( 13)= 0.667	XX( 14)= 0.708	XX( 15)= 0.750
XX( 16)= 0.792	XX( 17)= 0.833	XX( 18)= 0.875	XX( 19)= 0.917	XX( 20)= 0.950
XX( 21)= 1.000	XX( 22)= 2.500	XX( 23)= 2.800	XX( 24)= 4.800	

NODE DROPPED FROM SUBLINER-ABLATOR MODEL

Table 6.1 (Continued)

THIS IS THE CONFIGURATION FOR BODY PT. 3

ORIGINAL PAGE IS  
OF POOR QUALITY

--- 1. 0		
2. 0		
3. 0		
4. 0		
5. 0		
6. 0		
7. 0		
8. 0		
9. 0		
10. 0		
11. 0	B-STG.CORK ABLATOR SUBLIMER	0.782658 IN.
12. 0		
13. 0		
14. 0		
15. 0		
16. 0		
17. 0		
18. 0		
19. 0		
20. 0	0.129000 IN. CTD.COLUMB	
	RADIATION GAP	1.500000 IN.
	0.100000 IN. AL.7075-T6	
--- 21. 0	AL.7075-T6 THIN SKIN	0.300000 IN.
--- 22. 0	0.140000 IN. INCONL 617	
222222	222222	
2	2	
2	2	AL.7075-T6 Z STANDOFF
2	2	2.000000 IN.
222222	222222	
	0.180000 IN. CTD.COLUMB	
--- 23. 0		

NOTE DROPPED FROM SUBLIMER-ABLATOR MODEL

Table 6.1 (Continued)

THIS IS THE CONFIGURATION FOR BODY PT. 3

ORIGINAL PAGE IS  
OF POOR QUALITY

-- 1. 0  
2. 0  
3. 0  
4. 0  
5. 0  
6. 0  
7. 0  
8. 0  
9. 0  
10. 0      B-STG.CORK ABLATOR SUBLIMER      0.741394 IN.  
11. 0  
12. 0  
13. 0  
14. 0  
15. 0  
16. 0  
17. 0  
18. 0  
-- 19. 0      0.125000 IN. CTD.COLUMB  
-----  
RADIATION GAP      1.500000 IN.  
-----  
0.100000 IN. AL.7075-T6  
-- 20. 0  
AL.7075-T6 THIN SKIN      0.300000 IN.  
-- 21. 0  
0.140000 IN. INCONL 617  
-----  
Z Z Z Z Z Z      Z Z Z Z Z Z  
Z Z Z Z Z Z      AL.7075-T6 Z STANDOFF      2.000000 IN.  
Z Z Z Z Z Z      Z Z Z Z Z Z  
-----  
0.180000 IN. CTD.COLUMB  
-- 22. 0

TIME = 1000.00000 TIME STEP = 0.06152 NO. OF STEPS = 1043

INTEGRATED HEAT  
BTU/SQ.FT

HEAT RATES  
BTU/SQ.FT-SEC

CONVECTED      6115.4

CONVECTED      17.6

RADIATED      683.2

RADIATED      0.8

NET LOAD

5432.2

NET LOAD

16.7

STORED      319.2

STORED      0.5

SUBLIMED      5087.5

SUBLIMED      16.2

ADVECTED      25.4

ADVECTED      0.1

TPS NET

5432.1

TPS NET

16.7

Table 6.1 (Continued)

ORIGINAL PAGE IS  
OF POOR QUALITY

SURFACE RECESSION

DISTANCE 0.28137 IN.

TEMPERATURE DEG F

T( 1)= 760.400      T( 2)= 644.619      T( 3)= 513.528      T( 4)= 420.801      T( 5)= 358.421  
T( 6)= 310.831      T( 7)= 273.095      T( 8)= 242.609      T( 9)= 217.590      T( 10)= 196.701  
T( 11)= 179.223      T( 12)= 164.911      T( 13)= 152.110      T( 14)= 141.685      T( 15)= 132.842  
T( 16)= 125.406      T( 17)= 119.116      T( 18)= 113.785      T( 19)= 109.169      T( 20)= 100.016  
T( 21)= 100.016      T( 22)= 100.000

NODE POSITION INCHES

XX( 1)= 0.281      XX( 2)= 0.299      XX( 3)= 0.333      XX( 4)= 0.375      XX( 5)= 0.417  
XX( 6)= 0.498      XX( 7)= 0.500      XX( 8)= 0.542      XX( 9)= 0.583      XX( 10)= 0.625  
XX( 11)= 0.667      XX( 12)= 0.708      XX( 13)= 0.750      XX( 14)= 0.792      XX( 15)= 0.833  
XX( 16)= 0.879      XX( 17)= 0.917      XX( 18)= 0.958      XX( 19)= 1.000      XX( 20)= 2.900  
XX( 21)= 2.600      XX( 22)= 4.800

NODE DROPPED FROM SUBLIMER-ABLATOR MODEL

Table 6.1 (Continued)

1 THIS IS THE CONFIGURATION FOR BODY PT. 3

ORIGINAL PAGE IS  
OF POOR QUALITY

== 1. 0		
2. 0	I	
3. 0	I	
4. 0	I	
5. 0	I	
6. 0	I	
7. 0	I	
8. 0	I	
9. 0	I	
10. 0	B-STG.CORK ABLATOR SUBLINER	0.699940 IN.
11. 0		
12. 0	I	
13. 0	I	
14. 0	I	
15. 0	I	
16. 0	I	
17. 0	I	
==18. 0	0.125000 IN. CTD.COLUMN	
	RADIATION GAP	1.500000 IN.
	0.100000 IN. AL.7075-T6	
==19. 0		
	AL.7075-T6 THIN SKIN	0.300000 IN.
==20. 0		
	0.140000 IN. INCONL 617	
222222	222222	
2	2	
2	2 AL.7075-T6 2 STANDOFF	2.000000 IN.
2	2	
222222	222222	
	0.180000 IN. CTD.COLUMN	
==21. 0		

NOTE DROPPED FROM SUBLINER-ABLATOR MODEL

Table 6.1 (Continued)

THIS IS THE CONFIGURATION FOR BODY PT. 3

— 1. 0		
2. 0		
3. 0		
4. 0		
5. 0		
6. 0		
7. 0		
8. 0		
9. 0	B-STB.CORK ABLATOR SUBLIMER	0.158035 IN.
10. 0		
11. 0		
12. 0		
13. 0		
14. 0		
15. 0		
16. 0		
— 17. 0	0.125000 IN. CTD.COLUMB	

ORIGINAL PAGE IS  
OF POOR QUALITY.

	RADIATION GAP	1.500000 IN.
	0.100000 IN. AL.7075-T6	
— 18. 0	AL.7075-T6 THIN SKIN	0.300000 IN.
— 19. 0	0.140000 IN. INCON 617	
222222	222222	
2	2	
2	2 AL.7075-T6 2 STANDOFF	2.000000 IN.
2	2	
222222	222222	
	0.180000 IN. CTD.COLUMB	
— 20. 0		

TIME = 1100.00000 TIME STEP = 0.34009 NO. OF STEPS = 1220

INTEGRATED HEAT		HEAT RATES	
BTU/SEC.FT		BTU/SEC.FT-SEC	
CONVECTED	7701.9	CONVECTED	12.9
RADIATED	765.8	RADIATED	0.8
NET LOAD	6936.0	NET LOAD	12.0
STORED	369.7	STORED	0.4
SUBLINED	6539.7	SUBLINED	11.9
ADVECTED	34.5	ADVECTED	0.1
TPS NET	6936.0	TPS NET	12.0

Table 6.1 (Continued)

## SURFACE RECESSION

DISTANCE 0.36147 IN.

TEMPERATURE DEG F

RECESSION RATE 0.00080 IN/SEC

T( 1)= 760.400	T( 2)= 637.213	T( 3)= 509.053	T( 4)= 410.413	T( 5)= 344.900
T( 6)= 294.890	T( 7)= 256.280	T( 8)= 226.039	T( 9)= 201.990	T( 10)= 182.585
T( 11)= 166.729	T( 12)= 153.619	T( 13)= 142.694	T( 14)= 133.516	T( 15)= 125.750
T( 16)= 119.127	T( 17)= 113.429	T( 18)= 100.032	T( 19)= 100.032	T( 20)= 100.000

NODE POSITION INCHES

XX( 1)= 0.361	XX( 2)= 0.382	XX( 3)= 0.417	XX( 4)= 0.458	XX( 5)= 0.500
XX( 6)= 0.542	XX( 7)= 0.583	XX( 8)= 0.625	XX( 9)= 0.667	XX( 10)= 0.708
XX( 11)= 0.750	XX( 12)= 0.792	XX( 13)= 0.833	XX( 14)= 0.875	XX( 15)= 0.917
XX( 16)= 0.958	XX( 17)= 1.000	XX( 18)= 2.500	XX( 19)= 2.800	XX( 20)= 4.800

NODE DROPPED FROM SUBLIMER-ABLATOR MODEL

THIS IS THE CONFIGURATION FOR BODY PT. 3

--- 1. 0			1
2. 0			1
3. 0			1
4. 0			1
5. 0			1
6. 0			1
7. 0			1
8. 0			1
9. 0	B-STG.CORK ABLATOR SUBLIMER	0.616264 IN.	
10. 0			1
11. 0			1
12. 0			1
13. 0			1
14. 0			1
15. 0			1
---16. 0			1
	0.125000 IN. CTD.COLUMB		1
			1
			1
	RADIATION GAP	1.900000 IN.	
			1
			1
---17. 0	0.100000 IN. AL.7075-T6		1
---18. 0	AL.7075-T6 THIN SKIN	0.300000 IN.	
			1
	0.140000 IN. INCONEL 617		1
			1
222222	222222		1
2	2		1
2	2	AL.7075-T6 Z STANDEFF	2.000000 IN.
2	2		1
222222	222222		1
			1
---19. 0	0.180000 IN. CTD.COLUMB		1

ORIGINAL PAGE IS  
OF POOR QUALITY

Table 6.1 (Continued)

ORIGINAL PAGE IS  
OF POOR QUALITY

TIME = 1200.00000 . TIME STEP = 0.16211 NO. OF STEPS = 1384

INTEGRATED HEAT  
BTU/BG.FT

HEAT RATES  
BTU/BG.FT-SEC

CONVECTED	6741.6	CONVECTED	7.0
RADIATED	848.5	RADIATED	0.8
NET LOAD		NET LOAD	7.0
STORED	406.7	STORED	0.4
SUBLINED	7446.8	SUBLINED	6.6
ADVECTED	39.5	ADVECTED	0.0
TPS NET	7893.0	TPS NET	7.0

SURFACE RECEDSION

DISTANCE 0.41182 IN.

RECEDSION RATE 0.00090 IN/SEC

TEMPERATURE DEG F

T( 1)= 760.400	T( 2)= 680.492	T( 3)= 555.839	T( 4)= 492.119	T( 5)= 380.072
T( 6)= 323.148	T( 7)= 277.637	T( 8)= 241.483	T( 9)= 212.780	T( 10)= 189.906
T( 11)= 171.952	T( 12)= 156.687	T( 13)= 144.518	T( 14)= 134.433	T( 15)= 125.957
T( 16)= 118.715	T( 17)= 100.054	T( 18)= 100.054	T( 19)= 100.002	

NODE POSITION INCHES

XX( 1)= 0.412	XX( 2)= 0.428	XX( 3)= 0.498	XX( 4)= 0.800	XX( 5)= 0.942
XX( 6)= 0.583	XX( 7)= 0.629	XX( 8)= 0.667	XX( 9)= 0.708	XX( 10)= 0.750
XX( 11)= 0.792	XX( 12)= 0.833	XX( 13)= 0.875	XX( 14)= 0.917	XX( 15)= 0.958
XX( 16)= 1.000	XX( 17)= 2.500	XX( 18)= 2.800	XX( 19)= 4.800	

NODE DROPPED FROM SUBLINER-ABLATOR MODEL

Table 6.1 (Continued)

THIS IS THE CONFIGURATION FOR BODY PT. 3

1. 0				ORIGINAL PAGE IS OF POOR QUALITY.
2. 0				
3. 0				
4. 0				
5. 0				
6. 0				
7. 0				
8. 0	B-STG.CORK	ABLATOR SUBLIMER	0.574676 IN.	
9. 0				
10. 0				
11. 0				
12. 0				
13. 0				
14. 0				
15. 0	0.123000 IN. CTD.COLUMN			
		RADIATION GAP	1.900000 IN.	
		0.100000 IN. AL.7075-T6		
16. 0		AL.7075-T6 THIN SKIN		
17. 0		0.140000 IN. INCONEL 617	0.300000 IN.	
222222	222222			
2	2			
2	2	AL.7075-T6 2 STANDOFF	2.000000 IN.	
2	2			
222222	222222			
		0.180000 IN. CTD.COLUMN		
18. 0				

TIME = 1300.00000 TIME STEP = 0.71448 NO. OF STEPS = 1369

INTEGRATED HEAT  
BTU/SQ.FT

HEAT RATES  
BTU/SQ.FT-SEC

CONNECTED	9323.0	CONNECTED	4.1
RADIATED	931.2	RADIATED	0.8
NET LOAD	8391.8	NET LOAD	3.2
STORED	440.1	STORED	0.3
SUBLINED	7909.9	SUBLINED	2.9
ADVECTED	41.8	ADVECTED	0.0
TPS NET	8391.7	TPS NET	3.2

Table 6.1 (Continued)

ORIGINAL PAGE IS  
OF POOR QUALITY

SURFACE RECEDITION  
DISTANCE 0.43746 IN.

TEMPERATURE DEG F

T( 1)= 760.400	T( 2)= 659.889	T( 3)= 549.233	T( 4)= 457.768	T( 5)= 391.872
T( 6)= 337.706	T( 7)= 292.214	T( 8)= 234.516	T( 9)= 223.871	T( 10)= 198.209
T( 11)= 177.716	T( 12)= 160.885	T( 13)= 147.027	T( 14)= 135.474	T( 15)= 125.657
T( 16)= 100.086	T( 17)= 100.086	T( 18)= 100.004		

NODE POSITION INCHES

XX( 1)= 0.437	XX( 2)= 0.462	XX( 3)= 0.500	XX( 4)= 0.542	XX( 5)= 0.583
XX( 6)= 0.625	XX( 7)= 0.667	XX( 8)= 0.708	XX( 9)= 0.750	XX( 10)= 0.792
XX( 11)= 0.833	XX( 12)= 0.875	XX( 13)= 0.917	XX( 14)= 0.958	XX( 15)= 1.000
XX( 16)= 2.900	XX( 17)= 2.800	XX( 18)= 4.800		

TIME = 1400.00000 TIME STEP = 0.92102 NO. OF STEPS = 1676

INTEGRATED HEAT  
BTU/SQ.FT

CONNECTED	9580.4	CONNECTED	1.2
RADIATED	1013.9	RADIATED	0.8
NET LOAD	8566.6	NET LOAD	0.4
STORED	466.2	STORED	0.2
SUBLINED	8057.9	SUBLINED	0.1
ADVECTED	42.4	ADVECTED	0.0
TPS NET	8566.5	TPS NET	0.4

SURFACE RECEDITION  
DISTANCE 0.44543 IN.

TEMPERATURE DEG F

T( 1)= 760.400	T( 2)= 651.868	T( 3)= 596.917	T( 4)= 509.654	T( 5)= 441.324
T( 6)= 386.024	T( 7)= 337.964	T( 8)= 296.053	T( 9)= 260.032	T( 10)= 229.423
T( 11)= 203.613	T( 12)= 181.932	T( 13)= 163.737	T( 14)= 148.292	T( 15)= 135.091
T( 16)= 100.131	T( 17)= 100.131	T( 18)= 100.010		

NODE POSITION INCHES

XX( 1)= 0.446	XX( 2)= 0.465	XX( 3)= 0.500	XX( 4)= 0.542	XX( 5)= 0.583
XX( 6)= 0.625	XX( 7)= 0.667	XX( 8)= 0.708	XX( 9)= 0.750	XX( 10)= 0.792
XX( 11)= 0.833	XX( 12)= 0.875	XX( 13)= 0.917	XX( 14)= 0.958	XX( 15)= 1.000
XX( 16)= 2.900	XX( 17)= 2.800	XX( 18)= 4.800		

INITIAL MASS = 28.83250 (LBM/SQ.FT.)

(H)

Table 6.1 (Concluded)

TSTART = 0.000    TTSTOP = 1410.000    TIMPT = 100.000  
 DTIM = 10.000    STAB = 2.000    TOL = 0.001  
 NRP = 1    NEXT = 20    NSTP = 3000    BET = 0.500  
 IPFLAG = 1

(A)

TABLES

(B)

HRSI COAT - MAT NO. 4

MAXIMUM TEMPERATURE 2300.40 DEG F

TEMP. (DEG F)	DENSITY (LBM/CU.FT)
-0.459E+03	0.1040E+03
0.9540E+04	0.1040E+03

ORIGINAL PAGE IS  
OF POOR QUALITY

TEMP. (DEG F)	SPECIFIC HEAT (BTU/LBM-DEG F)
-0.459E+03	0.1500E+00
-0.249E+03	0.1500E+00
-0.149E+03	0.1700E+00
0.400E+00	0.1900E+00
0.2504E+03	0.2150E+00
0.5004E+03	0.2400E+00
0.1000E+04	0.2850E+00
0.2000E+04	0.3450E+00
0.3000E+04	0.3900E+00

TEMP. (DEG F)	CONDUCTIVITY (BTU/FT-S-DEG F)
-0.459E+03	0.1181E-03
-0.249E+03	0.1181E-03
-0.149E+03	0.1250E-03
0.400E+00	0.1353E-03
0.2504E+03	0.1528E-03
0.5004E+03	0.1678E-03
0.1000E+04	0.1938E-03
0.2000E+04	0.2453E-03
0.3000E+04	0.3278E-03

TEMP. (DEG F)	EMISSIVITY (DIMENSIONLESS)
-0.459E+03	0.6500E+00
0.9540E+04	0.8900E+00

L1-900 - MAT NO. 5

Table 6.2 Output For Example Case Two (Table 5.9)

MAXIMUM TEMPERATURE 2300.40 DEG F

ORIGINAL DATA IS  
OF POOR QUALITY

TEMP. (DEG F)	DENSITY (LBM/CU.FT)
-0.4596E+03	0.9000E+01
0.9540E+04	0.9000E+01

TEMP. (DEG F)	SPECIFIC HEAT (BTU/LBM-DEG F)
-0.4596E+03	0.7000E-01
-0.2496E+03	0.7000E-01
-0.1496E+03	0.1050E+00
0.4000E+00	0.1500E+00
0.2504E+03	0.2100E+00
0.5004E+03	0.2320E+00
0.1000E+04	0.2680E+00
0.1500E+04	0.3000E+00
0.1750E+04	0.3030E+00
0.3000E+04	0.3030E+00

CONDUCTIVITY  
(BTU/FT-S-DEG F)

TEMP. PRESSURE (LB/SQ.FT)	0.00	0.21	2.12	21.16	211.60	2116.00
-0.4596E+03	0.1389E-05	0.1389E-05	0.2083E-05	0.4166E-05	0.6060E-05	0.6472E-05
-0.2496E+03	0.1389E-05	0.1389E-05	0.2083E-05	0.4166E-05	0.6060E-05	0.6472E-05
0.4000E+00	0.2083E-05	0.2083E-05	0.2777E-05	0.5033E-05	0.6944E-05	0.7638E-05
0.2504E+03	0.2555E-05	0.2555E-05	0.3472E-05	0.6250E-05	0.8777E-05	0.9472E-05
0.5004E+03	0.3472E-05	0.3472E-05	0.4633E-05	0.7666E-05	0.1111E-04	0.1202E-04
0.7504E+03	0.4861E-05	0.4861E-05	0.6000E-05	0.9027E-05	0.1364E-04	0.1483E-04
0.1000E+04	0.6472E-05	0.6472E-05	0.7639E-05	0.1088E-04	0.1667E-04	0.1827E-04
0.1250E+04	0.8533E-05	0.8533E-05	0.9722E-05	0.1366E-04	0.2014E-04	0.2172E-04
0.1500E+04	0.1155E-04	0.1155E-04	0.1275E-04	0.1714E-04	0.2430E-04	0.2616E-04
0.1750E+04	0.1575E-04	0.1575E-04	0.1694E-04	0.2130E-04	0.2944E-04	0.3138E-04
0.2000E+04	0.2039E-04	0.2039E-04	0.2172E-04	0.2616E-04	0.3527E-04	0.3777E-04
0.2300E+04	0.2683E-04	0.2683E-04	0.2833E-04	0.3223E-04	0.4305E-04	0.4638E-04
0.2500E+04	0.3222E-04	0.3222E-04	0.3416E-04	0.3861E-04	0.4972E-04	0.5388E-04
0.2800E+04	0.4277E-04	0.4277E-04	0.4500E-04	0.5000E-04	0.6111E-04	0.6722E-04
0.3000E+04	0.5277E-04	0.5277E-04	0.5444E-04	0.6080E-04	0.7277E-04	0.8055E-04

TEMP. (DEG F)	EMISSIVITY (DIMENSIONLESS)
-0.4596E+03	0.1000E+01
0.9540E+04	0.1000E+01

AL.7079-T6 - NAT NO. 1

TABLE 6.2 (Continued)

MAXIMUM TEMPERATURE . . . . . 200.40 DEG F

ORIGINAL PAGE IS  
OF POOR QUALITY

TEMP. (DEG F)	DENSITY (LBM/CU.FT)
-0.4596E+03	0.1750E+03
0.9940E+04	0.1750E+03

TEMP. (DEG F)	SPECIFIC HEAT (BTU/LBM-DEG F)
-0.4596E+03	0.1700E+00
-0.1496E+03	0.1700E+00
0.4000E+00	0.1950E+00
0.8004E+03	0.2100E+00
0.8604E+03	0.2750E+00
0.1000E+04	0.2750E+00
0.9940E+04	0.2750E+00

TEMP. (DEG F)	CONDUCTIVITY (BTU/FT-S-DEG F)
-0.4596E+03	0.1400E-01
-0.1496E+03	0.1400E-01
0.4000E+00	0.2000E-01
0.3004E+03	0.2500E-01
0.4004E+03	0.2700E-01
0.5004E+03	0.2900E-01

TEMP. (DEG F)	EMISSIVITY (DIMENSIONLESS)
-0.4596E+03	0.1200E+00
0.9940E+04	0.1200E+00

INCONEL 617 - MAT NO. 17

MAXIMUM TEMPERATURE . . . . . 1800.40 DEG F

TEMP. (DEG F)	DENSITY (LBM/CU.FT)
-0.4596E+03	0.5219E+03
0.9940E+04	0.5219E+03

TEMP. (DEG F)	SPECIFIC HEAT (BTU/LBM-DEG F)
-0.4596E+03	0.1000E+00

Table 6.2 (Continued)

0.7840E+02	0.1000E+00
0.2004E+03	0.1040E+00
0.4004E+03	0.1110E+00
0.6004E+03	0.1170E+00
0.1000E+04	0.1310E+00
0.1200E+04	0.1370E+00
0.1400E+04	0.1440E+00
0.1600E+04	0.1500E+00
0.1800E+04	0.1570E+00
0.2000E+04	0.1630E+00
0.9540E+04	0.1630E+00

ORIGINAL PAGE IS  
OF POOR QUALITY

TEMP. CONDUCTIVITY  
(DEG F) (BTU/FT-S-DEG F)

-0.4596E+03	0.2176E-02
0.7840E+02	0.2176E-02
0.2004E+03	0.2328E-02
0.4004E+03	0.2616E-02
0.6004E+03	0.2894E-02
0.1000E+04	0.3449E-02
0.1200E+04	0.3727E-02
0.1400E+04	0.4005E-02
0.1600E+04	0.4282E-02
0.1800E+04	0.4560E-02
0.2000E+04	0.4838E-02
0.9540E+04	0.4838E-02

TEMP. EMISSIVITY  
(DEG F) (DIMENSIONLESS)

-0.4596E+03	0.1500E+00
0.9540E+04	0.1500E+00

TITANIUM - MAT NO. 9

MAXIMUM TEMPERATURE 600.40 DEG F

TEMP. DENSITY  
(DEG F) (LBIN/CU.FT)

-0.4596E+03	0.5120E+03
0.9540E+04	0.5120E+03

TEMP. SPECIFIC HEAT  
(DEG F) (BTU/LBH-DEG F)

-0.4596E+03	0.9600E-01
-0.1996E+03	0.9600E-01
0.4000E+00	0.1250E+00
0.4004E+03	0.1460E+00
0.1200E+04	0.1600E+00
0.9540E+04	0.1600E+00

Table 6.2 (Continued)

TEMP. CONDUCTIVITY  
(DEG F) (BTU/FT-B-DEG F)

-0.459E+03 0.1200E-02  
0.704E+02 0.1200E-02  
0.500E+03 0.1500E-02  
0.100E+04 0.2800E-02  
0.954E+04 0.2800E-02

ORIGINAL PAGE IS  
OF POOR QUALITY

TEMP. EMISSIVITY  
(DEG F) (DIMENSIONLESS)

-0.459E+03 0.1200E+00  
0.954E+04 0.1200E+00

Table 6.2 (Continued)

## BODY POINT NUMBER = 3 SAMPLE MINIVER INPUT TABLE

TIME (SEC)	FILM COEF. (LBIN/SQ.FT-SEC)	REC ENTHALPY (BTU/LBM)	PRESSURE (LBF/SQ.FT)
0.0000E+00	0.6490E-05	0.1126E+03	0.1252E+01
0.5000E+02	0.1221E-04	0.1124E+03	0.3449E+01
0.1000E+03	0.2502E-04	0.1123E+03	0.1090E+00
0.1290E+03	0.3831E-04	0.1123E+03	0.1992E+00
0.1500E+03	0.5367E-04	0.1122E+03	0.3777E+00
0.1750E+03	0.8009E-04	0.1123E+03	0.7313E+00
0.2000E+03	0.1203E-03	0.1124E+03	0.1440E+01
0.2250E+03	0.1795E-03	0.1127E+03	0.2802E+01
0.2750E+03	0.2328E-03	0.1113E+03	0.7749E+01
0.3000E+03	0.3015E-03	0.1109E+03	0.1126E+02
0.3500E+03	0.3710E-03	0.1072E+03	0.1779E+02
0.4000E+03	0.3974E-03	0.1060E+03	0.2004E+02
0.4500E+03	0.4108E-03	0.1027E+03	0.2121E+02
0.5000E+03	0.4259E-03	0.9966E+04	0.2314E+02
0.5280E+03	0.4385E-03	0.9769E+04	0.2379E+02
0.5560E+03	0.4402E-03	0.9577E+04	0.2470E+02
0.5980E+03	0.4515E-03	0.9270E+04	0.2599E+02
0.6400E+03	0.4661E-03	0.8966E+04	0.2806E+02
0.6840E+03	0.5320E-03	0.8676E+04	0.2915E+02
0.6890E+03	0.6833E-03	0.8680E+04	0.3124E+02
0.7100E+03	0.7182E-03	0.8450E+04	0.3105E+02
0.7330E+03	0.8937E-03	0.8228E+04	0.3303E+02
0.7520E+03	0.1013E-02	0.8106E+04	0.3425E+02
0.7660E+03	0.1148E-02	0.7984E+04	0.3565E+02
0.7800E+03	0.1328E-02	0.7863E+04	0.3733E+02
0.7940E+03	0.1340E-02	0.7739E+04	0.3927E+02
0.8080E+03	0.1784E-02	0.7610E+04	0.4128E+02
0.8220E+03	0.2090E-02	0.7476E+04	0.4368E+02
0.8360E+03	0.2389E-02	0.7149E+04	0.4938E+02
0.8780E+03	0.2615E-02	0.6777E+04	0.5542E+02
0.9060E+03	0.2856E-02	0.6385E+04	0.6202E+02
0.9760E+03	0.3470E-02	0.5284E+04	0.7639E+02
0.1004E+04	0.3900E-02	0.4800E+04	0.8486E+02
0.1032E+04	0.4267E-02	0.4291E+04	0.8986E+02
0.1064E+04	0.4461E-02	0.3797E+04	0.9285E+02
0.1074E+04	0.4484E-02	0.3562E+04	0.9301E+02
0.1102E+04	0.4499E-02	0.3117E+04	0.9302E+02
0.1116E+04	0.4597E-02	0.2903E+04	0.9263E+02
0.1144E+04	0.4844E-02	0.2496E+04	0.9407E+02
0.1172E+04	0.5133E-02	0.2125E+04	0.9565E+02
0.1204E+04	0.5153E-02	0.1813E+04	0.9244E+02
0.1260E+04	0.5462E-02	0.1253E+04	0.8645E+02
0.1290E+04	0.5814E-02	0.1034E+04	0.8921E+02
0.1350E+04	0.6301E-02	0.6930E+03	0.8369E+02
0.1380E+04	0.6169E-02	0.3627E+03	0.8032E+02
0.1410E+04	0.5844E-02	0.4545E+03	0.7643E+02

ORIGINAL PAGE IS  
OF POOR QUALITY

Table 6.2 (Continued)

## STRUCTURE DEFINITION

BODY POINT 3

TINIT = 100.00 DEG F TBINR = 0.00 DEG F FIJ = 1.000

(D)

NODE NUMBER = 1 DISTANCE FROM SURFACE = 0.00000E+00 IN.  
 CONDUCTOR NUMBER = 1  
 STRUCTURE TYPE = 1 SLAB  
 MATERIAL 1 = MRSI COAT  
 NODE NUMBER = 2 DISTANCE FROM SURFACE = 0.10000E+00 IN.

NODE NUMBER = 2 DISTANCE FROM SURFACE = 0.10000E+00 IN.  
 CONDUCTOR NUMBER = 2  
 STRUCTURE TYPE = 1 SLAB  
 MATERIAL 1 = LI-900  
 NODE NUMBER = 3 DISTANCE FROM SURFACE = 0.16250E+00 IN.

NODE NUMBER = 3 DISTANCE FROM SURFACE = 0.16250E+00 IN.  
 CONDUCTOR NUMBER = 3  
 STRUCTURE TYPE = 1 SLAB  
 MATERIAL 1 = LI-900  
 NODE NUMBER = 4 DISTANCE FROM SURFACE = 0.22500E+00 IN.

NODE NUMBER = 4 DISTANCE FROM SURFACE = 0.22500E+00 IN.  
 CONDUCTOR NUMBER = 4  
 STRUCTURE TYPE = 1 SLAB  
 MATERIAL 1 = LI-900  
 NODE NUMBER = 5 DISTANCE FROM SURFACE = 0.28750E+00 IN.

NODE NUMBER = 5 DISTANCE FROM SURFACE = 0.28750E+00 IN.  
 CONDUCTOR NUMBER = 5  
 STRUCTURE TYPE = 1 SLAB  
 MATERIAL 1 = LI-900  
 NODE NUMBER = 6 DISTANCE FROM SURFACE = 0.35000E+00 IN.

NODE NUMBER = 6 DISTANCE FROM SURFACE = 0.35000E+00 IN.  
 CONDUCTOR NUMBER = 6  
 STRUCTURE TYPE = 1 SLAB  
 MATERIAL 1 = LI-900  
 NODE NUMBER = 7 DISTANCE FROM SURFACE = 0.41250E+00 IN.

NODE NUMBER = 7 DISTANCE FROM SURFACE = 0.41250E+00 IN.  
 CONDUCTOR NUMBER = 7  
 STRUCTURE TYPE = 1 SLAB  
 MATERIAL 1 = LI-900  
 NODE NUMBER = 8 DISTANCE FROM SURFACE = 0.47500E+00 IN.

ORIGINAL PAGE IS  
OF POOR QUALITY

Table 6.2 (Continued)

ORIGINAL PAGE IS  
OF POOR QUALITY

NODE NUMBER = 8 DISTANCE FROM SURFACE = 0.47500E+00 IN.  
CONDUCTOR NUMBER = 8  
STRUCTURE TYPE = 1 SLAB  
MATERIAL 1 = LI-900  
NODE NUMBER = 9 DISTANCE FROM SURFACE = 0.53750E+00 IN.

NODE NUMBER = 9 DISTANCE FROM SURFACE = 0.53750E+00 IN.  
CONDUCTOR NUMBER = 9  
STRUCTURE TYPE = 1 SLAB  
MATERIAL 1 = LI-900  
NODE NUMBER = 10 DISTANCE FROM SURFACE = 0.60000E+00 IN.

NODE NUMBER = 10 DISTANCE FROM SURFACE = 0.60000E+00 IN.  
CONDUCTOR NUMBER = 10  
STRUCTURE TYPE = 3 HONEY COMB  
MATERIAL 1 = AL.7075-T6  
MATERIAL 2 = AL.7075-T6  
MATERIAL 3 = AL.7075-T6  
NODE NUMBER = 11 DISTANCE FROM SURFACE = 0.13500E+01 IN.

NODE NUMBER = 11 DISTANCE FROM SURFACE = 0.13500E+01 IN.  
CONDUCTOR NUMBER = 11  
STRUCTURE TYPE = 4 CORRUGATED  
MATERIAL 1 = INCONI 617  
MATERIAL 2 = INCONI 617  
MATERIAL 3 = TITANIUM  
NODE NUMBER = 12 DISTANCE FROM SURFACE = 0.23500E+01 IN.

Table 6.2 (Continued)

THIS IS THE CONFIGURATION FOR BODY PT. 3

ORIGINAL PAGE IS  
OF POOR QUALITY

--- 1. 0

	MRSI COAT	SLAB	0.10000 IN.
---	2. 0		
	3. 0		
	4. 0		
	5. 0		
	6. 0	LI-900	SLAB
	7. 0		0.80000 IN.
	8. 0		
	9. 0		
---	10. 0		
		0.12000 IN. AL.7075-T6	

(E)

I	I	I	I	I		
I	I	I	I	I		
I	I	I	I	I	AL.7075-T6	HONEY COMB
I	I	I	I	I		0.75000 IN.
I	I	I	I	I		
					0.11000 IN. AL.7075-T6	
---	11. 0					
					0.08000 IN. INCONL 617	
V	V	V	V	V		
V	V	V	V	V		
V	V	V	V	V	TITANIUM	CORRUGATED
V	V	V	V	V		1.00000 IN.
V	V	V	V	V		
					0.08000 IN. INCONL 617	
---	12. 0					

(F)

TIME = 0.00000 TIME STEP = 0.00000 NO. OF STEPS = 0

INTEGRATED HEAT  
BTU/SQ.FT

HEAT RATES  
BTU/SQ.FT-SEC

CONVECTED	0.0	CONVECTED	0.0
RADIATED	0.0	RADIATED	0.0
NET LOAD	0.0	NET LOAD	0.0
STORED	0.0	STORED	0.0
SUBLINED	0.0	SUBLINED	0.0
ADVECTED	0.0	ADVECTED	0.0
TPS NET	0.0	TPS NET	0.0

Table 6.2 (Continued)

ORIGINAL PAGE IS  
OF POOR QUALITY

SURFACE RECEDITION

DISTANCE 0.00000 IN.

RECEDITION RATE 0.00000 IN/SEC

TEMPERATURE DEG F

T( 1)= 100.000

T( 2)= 100.000

T( 3)= 100.000

T( 4)= 100.000

T( 5)= 100.000

T( 6)= 100.000

T( 7)= 100.000

T( 8)= 100.000

T( 9)= 100.000

T( 10)= 100.000

T( 11)= 100.000

T( 12)= 100.000

TIME = 100.00000

TIME STEP =

0.96809

NO. OF STEPS =

41

INTEGRATED HEAT

BTU/SQ.FT

HEAT RATES

BTU/SQ.FT-SEC

CONVECTED 15.3

CONVECTED 0.3

RADIATED 3.0

RADIATED 0.0

NET LOAD 12.3

NET LOAD 0.2

STORED 12.3

STORED 0.2

SUBLINED 0.0

SUBLINED 0.0

ADVECTED 0.0

ADVECTED 0.0

TPS NET 12.3

TPS NET 0.2

SURFACE RECEDITION

DISTANCE 0.00000 IN.

RECEDITION RATE 0.00000 IN/SEC

TEMPERATURE DEG F

T( 1)= 169.266

T( 2)= 162.265

T( 3)= 132.831

T( 4)= 116.628

T( 5)= 108.067

T( 6)= 103.731

T( 7)= 101.635

T( 8)= 100.668

T( 9)= 100.232

T( 10)= 100.001

T( 11)= 100.000

T( 12)= 100.000

TIME = 200.00000

TIME STEP =

1.02477

NO. OF STEPS =

62

INTEGRATED HEAT

BTU/SQ.FT

HEAT RATES

BTU/SQ.FT-SEC

CONVECTED 61.0

CONVECTED 1.3

RADIATED 13.8

RADIATED 0.3

NET LOAD 67.2

NET LOAD 1.1

STORED 67.2

STORED 1.1

SUBLINED 0.0

SUBLINED 0.0

ADVECTED 0.0

ADVECTED 0.0

TPS NET 67.2

TPS NET 1.1

SURFACE RECEDITION

DISTANCE 0.00000 IN.

RECEDITION RATE 0.00000 IN/SEC

TEMPERATURE DEG F

T( 1)= 452.735

T( 2)= 423.512

T( 3)= 293.834

T( 4)= 211.620

T( 5)= 163.732

T( 6)= 135.797

T( 7)= 119.360

T( 8)= 110.076

T( 9)= 104.283

T( 10)= 100.026

T( 11)= 100.020

T( 12)= 100.002

Table 6.2 (Continued)

ORIGINAL PAGE IS  
OF POOR QUALITY.

TIME = 300.00000 TIME STEP = 0.19823 NO. OF STEPS = 124

INTEGRATED HEAT  
BTU/50.FT

HEAT RATES  
BTU/50.FT-SEC

CONVECTED	312.4	CONVECTED	3.2
RADIATED	105.4	RADIATED	1.9
NET LOAD	207.0	NET LOAD	1.4
STORED	207.0	STORED	1.4
SUBLIMED	0.0	SUBLIMED	0.0
ADVECTED	0.0	ADVECTED	0.0
TPS NET	207.0	TPS NET	1.4

SURFACE RECESSION

DISTANCE 0.00000 IN. RECESSION RATE 0.00000 IN/SEC

TEMPERATURE DEG F

T( 1)=1012.798	T( 2)= 976.538	T( 3)= 782.617	T( 4)= 602.948	T( 5)= 451.330
T( 6)= 333.322	T( 7)= 246.278	T( 8)= 184.383	T( 9)= 138.668	T( 10)= 100.391
T( 11)= 100.301	T( 12)= 100.029			

TIME = 400.00000 TIME STEP = 0.46954 NO. OF STEPS = 179

INTEGRATED HEAT  
BTU/50.FT

HEAT RATES  
BTU/50.FT-SEC

CONVECTED	689.1	CONVECTED	4.0
RADIATED	385.9	RADIATED	3.5
NET LOAD	303.2	NET LOAD	0.6
STORED	303.2	STORED	0.6
SUBLIMED	0.0	SUBLIMED	0.0
ADVECTED	0.0	ADVECTED	0.0
TPS NET	303.2	TPS NET	0.6

SURFACE RECESSION

DISTANCE 0.00000 IN. RECESSION RATE 0.00000 IN/SEC

TEMPERATURE DEG F

T( 1)=1255.578	T( 2)=1237.569	T( 3)=1103.591	T( 4)= 959.801	T( 5)= 812.666
T( 6)= 666.667	T( 7)= 524.981	T( 8)= 386.589	T( 9)= 247.965	T( 10)= 103.293
T( 11)= 102.816	T( 12)= 100.352			

Table 6.2 (Continued)

TIME = 500.00000 TIME STEP = 0.29317 NO. OF STEPS = 249 ORIGINAL PAGE IS  
 OF POOR QUALITY

INTEGRATED HEAT		HEAT RATES	
BTU/SQ.FT		BTU/SQ.FT-SEC	
CONVECTED	1094.9	CONVECTED	4.1
RADIATED	752.9	RADIATED	3.8
NET LOAD	342.0	NET LOAD	0.3
STORED	342.0	STORED	0.3
SUBLINED	0.0	SUBLINED	0.0
ADVECTED	0.0	ADVECTED	0.0
TPS NET	342.0	TPS NET	0.3

SURFACE RECEDSION  
 DISTANCE 0.00000 IN. RECESSION RATE 0.00000 IN/SEC  
 TEMPERATURE DEG F

T( 1)=1288.586	T( 2)=1277.074	T( 3)=1171.910	T( 4)=1096.579	T( 5)= 929.987
T( 6)= 793.873	T( 7)= 646.440	T( 8)= 487.771	T( 9)= 312.073	T( 10)= 109.496
T( 11)= 108.731	T( 12)= 101.737			

TIME = 600.00000 TIME STEP = 0.54968 NO. OF STEPS = 322

INTEGRATED HEAT		HEAT RATES	
BTU/SQ.FT		BTU/SQ.FT-SEC	
CONVECTED	1497.7	CONVECTED	4.0
RADIATED	1129.4	RADIATED	3.8
NET LOAD	369.3	NET LOAD	0.2
STORED	369.3	STORED	0.2
SUBLINED	0.0	SUBLINED	0.0
ADVECTED	0.0	ADVECTED	0.0
TPS NET	369.3	TPS NET	0.2

SURFACE RECEDSION  
 DISTANCE 0.00000 IN. RECESSION RATE 0.00000 IN/SEC  
 TEMPERATURE DEG F

T( 1)=1287.774	T( 2)=1277.886	T( 3)=1179.176	T( 4)=1071.059	T( 5)= 950.515
T( 6)= 818.517	T( 7)= 672.034	T( 8)= 511.087	T( 9)= 329.583	T( 10)= 116.426
T( 11)= 115.991	T( 12)= 104.472			

Table 6.2 (Continued)

TIME = 700.00000 TIME STEP = 0.98474 NO. OF STEPS = 396

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INTEGRATED HEAT  
BTU/SQ.FT

HEAT RATES  
BTU/SQ.FT-SEC

CONVECTED	1939.9		CONVECTED	9.7
RADIATED	1532.6		RADIATED	4.9
NET LOAD		427.3	NET LOAD	0.0
STORED	427.3		STORED	0.0
SUBLINED	0.0		SUBLINED	0.0
ADVECTED	0.0		ADVECTED	0.0
TPS NET		427.3	TPS NET	0.0

SURFACE RECEDSION

DISTANCE 0.00000 IN. RECESSION RATE 0.00000 IN/SEC

TEMPERATURE DEG F

T( 1)=1407.930	T( 2)=1384.297	T( 3)=1256.196	T( 4)=1123.893	T( 5)= 985.418
T( 6)= 841.158	T( 7)= 687.121	T( 8)= 521.981	T( 9)= 339.205	T(10)= 123.229
T(11)= 122.360	T(12)= 108.277			

TIME = 800.00000 TIME STEP = 0.94961 NO. OF STEPS = 481

INTEGRATED HEAT  
BTU/SQ.FT

HEAT RATES  
BTU/SQ.FT-SEC

CONVECTED	2732.0		CONVECTED	11.7
RADIATED	2189.2		RADIATED	9.6
NET LOAD		563.8	NET LOAD	2.1
STORED	563.8		STORED	2.1
SUBLINED	0.0		SUBLINED	0.0
ADVECTED	0.0		ADVECTED	0.0
TPS NET		563.8	TPS NET	2.1

SURFACE RECEDSION

DISTANCE 0.00000 IN. RECESSION RATE 0.00000 IN/SEC

TEMPERATURE DEG F

T( 1)=1750.231	T( 2)=1700.599	T( 3)=1535.362	T( 4)=1366.276	T( 5)=1191.687
T( 6)=1009.413	T( 7)= 818.077	T( 8)= 614.490	T( 9)= 392.530	T(10)= 130.807
T(11)= 129.737	T(12)= 112.903			

Table 6.2 (Continued)

TIME = 900.00000 TIME STEP = 0.65125 NO. OF STEPS = 600 ORIGINAL PAGE IS  
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INTEGRATED HEAT  
BTU/BG.FT

HEAT RATES  
BTU/BG.FT-SEC

CONVECTED	4264.8	CONVECTED	16.5
RADIATED	3543.0	RADIATED	15.9
NET LOAD	721.8	NET LOAD	0.9
STORED	721.8	STORED	0.9
SUBLINED	0.0	SUBLINED	0.0
ADVECTED	0.0	ADVECTED	0.0
TPS NET	721.8	TPS NET	0.9

SURFACE RECESSION

DISTANCE 0.00000 IN. RECESSION RATE 0.00000 IN/SEC

TEMPERATURE DEG F

T( 1)=2031.192	T( 2)=2003.603	T( 3)=1869.158	T( 4)=1711.928	T( 5)=1541.243
T( 6)=1347.765	T( 7)=1123.810	T( 8)= 938.972	T( 9)= 543.678	T( 10)= 142.943
T( 11)= 141.188	T( 12)= 118.692			

TIME = 1000.00000 TIME STEP = 0.52612 NO. OF STEPS = 741

INTEGRATED HEAT  
BTU/BG.FT

HEAT RATES  
BTU/BG.FT-SEC

CONVECTED	5911.3	CONVECTED	16.4
RADIATED	5110.6	RADIATED	15.7
NET LOAD	792.7	NET LOAD	0.7
STORED	792.7	STORED	0.7
SUBLINED	0.0	SUBLINED	0.0
ADVECTED	0.0	ADVECTED	0.0
TPS NET	792.7	TPS NET	0.7

SURFACE RECESSION

DISTANCE 0.00000 IN. RECESSION RATE 0.00000 IN/SEC

TEMPERATURE DEG F

T( 1)=2036.951	T( 2)=2013.919	T( 3)=1887.971	T( 4)=1748.456	T( 5)=1591.306
T( 6)=1409.575	T( 7)=1195.487	T( 8)= 933.759	T( 9)= 602.838	T( 10)= 160.286
T( 11)= 198.092	T( 12)= 126.829			

TIME = 1100.00000 TIME STEP = 0.05627 NO. OF STEPS = 885

Table 6.2 (Continued)

INTEGRATED HEAT BTU/SQ.FT		HEAT RATES BTU/BQ.FT-SEC		ORIGINAL WARM UP OF POOR QUALITY.
CONVECTED	7370.4	CONVECTED	11.6	
RADIATED	6561.9	RADIATED	12.0	
NET LOAD		NET LOAD	-0.4	
STORED	808.9	STORED	-0.4	
SUBLINED	0.0	SUBLINED	0.0	
ADVECTED	0.0	ADVECTED	0.0	
TPS NET	808.9	TPS NET	-0.4	
SURFACE RECESSION DISTANCE 0.00000 IN.		RECESSION RATE 0.00000 IN/SEC		
TEMPERATURE DEG F				
T( 1)=1874.703	T( 2)=1872.074	T( 3)=1771.484	T( 4)=1693.849	T( 5)=1515.935
T( 6)=1392.060	T( 7)=1153.591	T( 8)= 908.004	T( 9)= 595.531	T( 10)= 177.798
T( 11)= 179.627	T( 12)= 137.479			
TIME = 1200.00000	TIME STEP = 0.07275	NO. OF STEPS = 1010		
INTEGRATED HEAT BTU/SQ.FT		HEAT RATES BTU/BQ.FT-SEC		
CONVECTED	8295.0	CONVECTED	6.8	
RADIATED	7321.9	RADIATED	7.4	
NET LOAD		NET LOAD	-0.6	
STORED	773.1	STORED	-0.6	
SUBLINED	0.0	SUBLINED	0.0	
ADVECTED	0.0	ADVECTED	0.0	
TPS NET	773.1	TPS NET	-0.6	
SURFACE RECESSION DISTANCE 0.00000 IN.		RECESSION RATE 0.00000 IN/SEC		
TEMPERATURE DEG F				
T( 1)=1606.336	T( 2)=1610.530	T( 3)=1528.443	T( 4)=1429.194	T( 5)=1311.007
T( 6)=1168.983	T( 7)= 997.009	T( 8)= 787.067	T( 9)= 527.224	T( 10)= 191.614
T( 11)= 189.900	T( 12)= 149.606			
TIME = 1300.00000	TIME STEP = 0.00439	NO. OF STEPS = 1112		
INTEGRATED HEAT BTU/SQ.FT		HEAT RATES BTU/BQ.FT-SEC		
CONVECTED	8785.6	CONVECTED	3.3	
RADIATED	8071.3	RADIATED	3.9	
NET LOAD		NET LOAD	-0.6	
STORED	714.3	STORED	-0.6	
SUBLINED	0.0	SUBLINED	0.0	
ADVECTED	0.0	ADVECTED	0.0	
TPS NET	714.3	TPS NET	-0.6	

Table 6.2 (Continued)

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SURFACE RECEDITION

DISTANCE 0.00000 IN.

RECEDITION RATE 0.00000 IN/SEC

TEMPERATURE DEG F

T( 1)=1300.991

T( 2)=1308.961

T( 3)=1349.482

T( 4)=1173.469

T( 5)=1079.632

T( 6)= 964.443

T( 7)= 824.781

T( 8)= 659.586

T( 9)= 492.626

T( 11)= 199.629

T( 12)= 161.701

T( 10)= 200.894 \*

TIME = 1400.00000 TIME STEP = 0.10059 NO. OF STEPS = 1194

INTEGRATED HEAT

BTU/SG.FT

HEAT RATES

BTU/SG.FT-SEC

CONNECTED 8986.6

CONNECTED 0.9

RADIATED 6343.8

RADIATED 1.7

NET LOAD

642.8

NET LOAD

-0.8

STORED 642.8

STORED -0.8

SUBLIMED 0.0

SUBLIMED 0.0

ADVECTED 0.0

ADVECTED 0.0

TPS NET

642.8

TPS NET

-0.8

SURFACE RECEDITION

DISTANCE 0.00000 IN.

RECEDITION RATE 0.00000 IN/SEC

TEMPERATURE DEG F

T( 1)= 974.126

T( 2)= 990.248

T( 3)= 961.280

T( 4)= 913.690

T( 5)= 847.822

T( 6)= 763.097

T( 7)= 638.051

T( 8)= 532.607

T( 9)= 383.915

T( 11)= 205.611 \*

T( 12)= 172.647

T( 10)= 206.506 \*

INITIAL MASS = 32.26707 (LBM/SG.FT.)

1

MAXIMUM TEMPERATURE OF AL.7075-T6 EXCEEDED AT NODE 10

MAXIMUM TEMPERATURE OF AL.7075-T6 EXCEEDED AT NODE 10

MAXIMUM TEMPERATURE OF AL.7075-T6 EXCEEDED AT NODE 11

MAXIMUM TEMPERATURE OF AL.7075-T6 EXCEEDED AT NODE 11

Table 6.2 (Concluded)

Section 7.0

INCLUSIONS AND RECOMMENDATIONS

The EXIT5 code is an interactive one dimensional thermal analysis tool which has the capability to model a large variety of aerospace thermostructures with a minimum amount of effort on the part of the analyst. The code is used in conjunction with the LANMIN code which produces the environments and is linked to the EXIT5 code using an output file. The ability to store data describing the structure and the ability to access any number of environment files, allows the user to make parametric studies using various trajectories and TPS structure types, trading thermal performance and weight.

The present program's capabilities allow the analyst to investigate many of the current and envisioned TPS structures. However, limitations do exist as every candidate structure type could not be anticipated. In view of this, an effort was made to allow changes, modifications, and additions to be made with a minimum of reprogramming effort. Additional capability can be added to give the user a more versatile tool by incorporating the following recommendations:

1. Add the capability to include additional types of boundary conditions on the backwall. Presently an adiabatic boundary is assumed. Known temperature and known heat flux should be added.
2. Logic should be added to automatically change a slab structure type to a thin type if the computed time step is too small. Presently, the user must make this change.
3. Add logic that would allow heat flux on the surface to be computed given the temperature history of a thermocouple placed within the structure.
4. Include a TPS sizing routine to automatically optimize the structure given temperature, weight and cost constraints.
5. Add additional routines for computing equivalent thermal conductance, capacitance, and weight for additional structure types e. g. hot section stringer-panel etc.
6. Add logic which will allow all ablation material to be removed. Presently the surface node must remain in the ablation material.

Consequently, a small amount of ablator material must remain on the substructure.

Additional studies are recommended to lend confidence to the accuracy of the effective thermal conductance calculations of the various structure types. Comparison with experimental or test data would be quite useful in determining the dependence of the conductance as a function of temperature level, temperature difference, joint or contact conductance and material properties.

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Section 8.0

REFERENCES

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2. Landau, H. G. "Heat Conduction in a Melting Solid", Quart. Appl. Math., Vol. 8, No. 1, Jan. 1950, pp 81-94.
3. Love T. J. Radiative Heat Transfer, Merrill Publishing Co., Columbus, Ohio, 1968.

**APPENDIX**  
**(EXITS Listing)**

## PROGRAM MAIN

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**EXITS CODE**

## **EXPLICIT INTERACTIVE THERMAL STRUCTURES CODE**

REMTech Inc. 1983

BY J. POND  
C. SCHMITZ  
PH. 205-536-8581

PARAMETER (NMB1=20, NMB2=6, NMB3=3, NMB4=6, NMB5=40, NMB6=10, NMB10=100)  
 NMB1 - MAX NUMBER BODY POINTS  
 NMB2 - MAX NUMBER LAYERS/BODY POINT  
 NMB3 - MAX NUMBER MATERIALS/LAYER  
 NMB4 - MAX NUMBER DIMENSIONS/LAYER  
 NMB5 - MAX NUMBER CONDUCTORS/NODES  
 NMB6 - MAX NUMBER MATERIALS USED  
 NMB7 - LARGEST MATERIAL NUMBER  
 NMB8 - MAX NUMBER MATERIAL PROPERTY TABLES STORED  
       ((4\*NMB6)+1)  
 NMB9 - SIZE OF MONOVARIATE MATERIAL PROPERTY TABLES ARRAY  
       ((MAX TABLE ENTRIES)\*2+1)  
 NMB10 - MAX NUMBER TIMES FOR MINIVÉR ENVIRONMENT TABLE  
 NMB11 - SIZE OF BIVARIATE MATERIAL PROPERTY TABLE ARRAY(2ND DIM)  
       (MAX NUMBER OF TEMPERATURES)  
 NMB12 - SIZE OF BIVARIATE MATERIAL PROPERTY TABLE ARRAY(3RD DIM)  
       ((MAX NUMBER OF PRESSURES)+1)

```

COMMON/ENVIR/TM1(NMB10),HC1(NMB10),HAW1(NMB10),PRES1(NMB10)
COMMON/GAP/T1,T2,TH1,TH2,TH3,TH,P,H,M1,M2,M3,TOL,BET,SIG,XM,
$ CAP1,CAP2,XK
COMMON/INIT/TSTART,TSTOP,TIMPT,DTIM,NBP,NEXT,METRIC,
$ METRIK,METRIX,NSTP,IPFLAG,TINI(NMB1),SINKT(NMB1),XFIJ(NMB1),
$ MBP(NMB1),IN1,IN2
COMMON/TAX/ TK(NMB2),XX(NMB5)
COMMON/TIME/NNDS,CONV,CRAD,STAB,ISBFG,NCDS
COMMON/ARA/T(NMB5),TO(NMB5),C(NMB5),CD(NMB5),ICD(NMB5),L(NMB5,2)
COMMON/LD/LS(NMB1,NMB2),XP(NMB1,NMB2,NMB4),MATS(NMB1,NMB2,NMB3),
$ NS(NMB1)
COMMON /NODES/NN,I,TT,TINIT,TSINK,FIJ,TMPMAX(NMB6)
COMMON/CTMP/TAW,DTSM
COMMON/CAC/NEXFG,NIT,XMAS
COMMON/PICT/NN1S(NMB1,NMB2)
COMMON/SUBLM/TSUB,XL,XLP,EXCHT,NAB,ISTAR,NDIV,IAB,EXCHSV,QADV,
$ QADV5,TMSV,IDROP
COMMON/TITLE/CHAR2,CHAR1,FNAM1,FNAM3
COMMON/PRESS/PRES
COMMON/SAVE/XEND1,XEND2,XTST1,XTST2,XLTS,XMIN
DIMENSION TTT(NMB5),ZXX(NMB5),FLGG(NMB5,NMB3,2)
CHARACTER#13 CHAR1(NMB6)
CHARACTER#20 FNAM1,FNAM3
CHARACTER#10 CHAR2(NMB6)
INTEGER      FLG(NMB5)
IN1=5
IN2=5
CPW=.24

```

SIG=.1714E-8/3600.0  
TT=1000.0

C DETERMINE INITIAL CONDITIONS AND STRUCTURE FOR ALL BODY POINTS

CALL INPGEO  
WRITE(9,720)TSTART,TSTOP,TIMPT  
WRITE(9,722)DTIM,STAB,TOL,BET  
WRITE(9,723)NBP,NEXT,NSTP,IPFLAG  
DO 8000 I=1,NBP  
IAB=0  
NDIV=4  
ISTAR=0  
TINIT=TINI(I)  
TSINK=SINKT(I)  
FIJ=XFIJ(I)

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C FIND PROPERTIES OF MATERIALS

CALL DATA1

C FIND MINIVER ENVIRONMENT FOR BODY POINT

CALL DATA2(MBP(I))  
NN=NS(I)  
XX(I)=0.0  
XMAS=0.0

C DETERMINE NODAL NETWORK

CALL NODE  
MP=1

C DRAW PICTURE (INCLUDING NODES) FOR OUTPUT FILE

CALL PICTUR(9,1)  
ISBFG=0  
DO 436 IK=1,NN  
IF(LS(I,IK).EQ.6)ISBFG=1

436 CONTINUE

TIM=TSTART

DT=0.0

NCTRL=0

NIT=0

NPR=0

NPFG=1

ISV=1

DTSM=0.0

NEXFG=0

DO 140 K=1,NNDS

C(K)=0.0

FLG(K)=(1H )

140 CONTINUE

DO 141 K=1,NCDS

CD(K)=0.0

141 CONTINUE

WRITE(9,749)

QCONV=0.0

QRAD=0.0

QNET=0.0

QSTOR=0.0

QSUB=0.0

EXCHSV=0.0

EXCHT=0.0

QTOT=0.0

QADV=0.0  
QADVS=0.0  
TMSV=0.0  
QCOR=0.0  
QRAR=0.0  
QNER=0.0  
QSTR=0.0  
QSUR=0.0  
QTOR=0.0  
RECR=0.0  
QADR=0.0  
QAD=0.0

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C  
C  
C

START TIME LOOP

1000 CONTINUE  
IF(NPFG.NE.1)GO TO 501  
XXSV=XX(1)  
TMPSV=TIM  
WRITE(9,750)TIM,DTSM,NIT  
WRITE(9,600)  
600 FORMAT(1X,'INTEGRATED HEAT',34X,'HEAT RATES')  
IF(METRIK.EQ.0)WRITE(9,601)  
IF(METRIK.EQ.1)WRITE(9,602)  
601 FORMAT(1X,'BTU/SQ.FT',40X,'BTU/SQ.FT-SEC',/)  
602 FORMAT(1X,'JOULES/SQ.M',38X,'WATTS/SQ.M',/)  
IF(METRIK.EQ.0)WRITE(9,603)QCONV,QCOR,QRAD,QRAR,QNET,QNER,  
\$ QSTOR,QSTR,bsub,qsur,qad,qadr,qtot,qtor  
Z1=QCONV\*11355.9  
Z2=QCOR\*11355.9  
Z3=QRAD\*11355.9  
Z4=QRAR\*11355.9  
Z5=QNET\*11355.9  
Z6=QNER\*11355.9  
Z7=QSTOR\*11355.9  
Z8=QSTR\*11355.9  
Z9=bsub\*11355.9  
Z10=qsur\*11355.9  
Z11=qad\*11355.9  
Z12=qadr\*11355.9  
Z13=qtot\*11355.9  
Z14=qtor\*11355.9  
IF(METRIK.EQ.1)WRITE(9,603)Z1,Z2,Z3,Z4,Z5,Z6,Z7,Z8,Z9,  
\$ Z10,Z11,Z12,Z13,Z14  
603 FORMAT(1X,'CONVECTED',T16,F10.1,T51,'CONVECTED',T61,F10.1,/,  
\$ 'RADIATED',T16,F10.1,T51,'RADIATED',T61,F10.1,/,  
\$ 'NET LOAD',T26,F10.1,T51,'NET LOAD',T71,F10.1,/,  
\$ 'STORED',T16,F10.1,T51,'STORED',T61,F10.1,/,  
\$ 'SUBLIMED',T16,F10.1,T51,'SUBLIMED',T61,F10.1,/,  
\$ 'ADVECTED',T16,F10.1,T51,'ADVECTED',T61,F10.1,/,  
\$ 'TPS NET',T26,F10.1,T51,'TPS NET',T71,F10.1,/)  
Z21=XX(1)\*12.  
ZZ2=RECR\*12.  
IF(METRIK.EQ.0)WRITE(9,604)Z21,ZZ2  
604 FORMAT(1X,'SURFACE RECEDITION',/, 'DISTANCE ',F11.5,

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$ ' IN.',T51,'RECESSION RATE ',F11.5,' IN/SEC')
ZZ1=XX(1)*12.0*2.54
ZZ2=RECR*12.0*2.54
IF(METRIK.EQ.1)WRITE(9,605)ZZ1,ZZ2
605 FORMAT(1X,'SURFACE RECESSION',/, ' DISTANCE ',F11.5,
$ ' CM.',T51,'RECESSION RATE ',F11.5,' CM/SEC')
DO 2000 INC3=1,NCDS
N1=L(INC3,1)
N2=L(INC3,2)
IF(METRIK.EQ.0)TTT(N1)=T(N1)-459.6
IF(METRIK.EQ.0)TTT(N2)=T(N2)-459.6
IF(METRIK.EQ.1)TTT(N1)=T(N1)/1.8
IF(METRIK.EQ.1)TTT(N2)=T(N2)/1.8
IF(LS(1,ICD(INC3)).EQ.7)GO TO 2000
IF(MATS(1,ICD(INC3),1).EQ.0)GO TO 11
C FLAGS FOR MAXIMUM TEMPERATURE OF MATERIALS
IF(T(N1).GT.TMPMAX(MATS(1,ICD(INC3),1)))FLGG(N1,1,1)=1
IF(T(N1).GT.TMPMAX(MATS(1,ICD(INC3),1)))FLG(N1)=1H*
11 IF(MATS(1,ICD(INC3),3).EQ.0)GO TO 12
IF(T(N1).GT.TMPMAX(MATS(1,ICD(INC3),3)))FLGG(N1,3,1)=1
IF(T(N1).GT.TMPMAX(MATS(1,ICD(INC3),3)))FLG(N1)=1H*
12 IF(MATS(1,ICD(INC3),2).EQ.0)GO TO 13
IF(T(N2).GT.TMPMAX(MATS(1,ICD(INC3),2)))FLGG(N2,2,1)=1
IF(T(N2).GT.TMPMAX(MATS(1,ICD(INC3),2)))FLG(N2)=1H*
13 IF(MATS(1,ICD(INC3),3).EQ.0)GO TO 14
IF(T(N2).GT.TMPMAX(MATS(1,ICD(INC3),3)))FLGG(N2,3,2)=1
IF(T(N2).GT.TMPMAX(MATS(1,ICD(INC3),3)))FLG(N2)=1H*
14 CONTINUE
2000 CONTINUE
IF(METRIK.EQ.0)WRITE(9,714)
IF(METRIK.EQ.1)WRITE(9,715)
714 FORMAT(1X,'TEMPERATURE DEG F')
715 FORMAT(1X,'TEMPERATURE DEG K')
WRITE(9,711)(JJ,TTT(JJ),FLG(JJ),JJ=1,NNDS)
IF(LS(1,1).NE.7)GO TO 499
DO 2020 IR1=1,NNDS
IF(METRIK.EQ.0)ZXX(IR1)=XX(IR1)*12.0
IF(METRIK.EQ.1)ZXX(IR1)=XX(IR1)*12.0*2.54
2020 CONTINUE
IF(METRIK.EQ.0)WRITE(9,716)
IF(METRIK.EQ.1)WRITE(9,717)
716 FORMAT(1X,'NODE POSITION INCHES')
717 FORMAT(1X,'NODE POSITION CM')
WRITE(9,726)(JJ,ZXX(JJ),JJ=1,NNDS)
499 CONTINUE
IF(IPFLAG.EQ.1)GO TO 500
DO 2021 IR1=1,NCDS
IF(METRIK.EQ.0)ZXX(IR1)=CD(IR1)
IF(METRIK.EQ.1)ZXX(IR1)=CD(IR1)*1899.0
2021 CONTINUE
IF(METRIK.EQ.0)WRITE(9,718)
IF(METRIK.EQ.1)WRITE(9,719)
718 FORMAT(1X,'CONDUCTORS BTU/SEC-DEG F')
719 FORMAT(1X,'CONDUCTORS WATTS/DEG K')
WRITE(9,713)(JJ,ZXX(JJ),JJ=1,NCDS)
```

DO 2022 IR1=1,NNDS  
 IF(METRIK.EQ.0)ZXX(IR1)=C(IR1)  
 IF(METRIK.EQ.1)ZXX(IR1)=C(IR1)\*1899.0  
**2022** CONTINUE  
 IF(METRIK.EQ.0)WRITE(9,727)  
 IF(METRIK.EQ.1)WRITE(9,728)  
**727** FORMAT(1X,'CAPACITORS BTU/DEG F')  
**728** FORMAT(1X,'CAPACITORS JOULES/DEG K')  
 WRITE(9,712)(JJ,ZXX(JJ),JJ=1,NNDS)  
**500** CONTINUE  
 NPR=NPR+1  
 NPFG=0  
**501** CONTINUE  
 MAT=MATS(1,1,1)  
 CALL HEATN(TIM,HC,HAW,PRES,ISV)  
 CALL PROP(TO(1),PRES,MAT,RO,CP,XK,EP)  
 CRAD=SIG\*EP\*FIJ\*(TO(1)\*\*2+TSINK\*\*2)\*(TO(1)+TSINK)  
 IF(LS(1,1).NE.7)GO TO 502  
**C** IF ABLATOR SUBLIMER THEN DETERMINE TEMPERATURE OF SUBLIMATION AND  
**C** HEAT OF SUBLIMATION  
 CALL SUBPR(PRES,1,TSUB)  
 CALL SUBPR(PRES,2,XL)  
 IF(TIM.LE.0.0)XLP=XL  
**502** CONTINUE  
 TAW=HAW/CPW  
 CONV=CPW\*HC  
 IF(NEXT\*(NIT/NEXT).EQ.NIT)NEXFG=1  
 IF(NEXFG.NE.1)GO TO 466  
**C** FIND CAPACITOR AND CONDUCTOR VALUES  
 CALL COMPOC  
**466** CONTINUE  
**C** DETERMINE TIME STEP  
 CALL TMSTEP(DTSM,1)  
 TMPT1=TIM-TSTART  
 TTTEMP=TMPT1+DTSM  
 PTIM=FLOAT(NPR)\*TIMPT  
 IF(TTEMP.LT.PTIM)GO TO 365  
 DTSM=PTIM-TMPT1  
 NEXFG=1  
 NPFG=1  
**365** CONTINUE  
**C** COMPUTE TEMPERATURES  
 CALL COMTMP  
 JJ=1CD(1)  
 JN=LS(1,JJ)  
 IF(JN.NE.7)GO TO 366  
 TMSV=TIM  
**C** FIND RECESSION DISTANCE  
 CALL ABSUB(1)  
 IF(IDROP.EQ.0)GO TO 3659  
 DO 3658 JIL=1,NMB5  
**3658** FLG(JIL)=0.0  
**3659** CONTINUE  
 NEXFG=1  
**366** CONTINUE

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C -----  
C -----  
C

### INTEGRATE HEAT LOADS

```
QQ=0.0  
QAD=QAD+QADV  
QCONV=QCONV+DTSM*(TAW-TO(1))*CONV  
QRAD=QRAD+DTSM*(TO(1)-TSINK)*CRAD  
QCOR=(TAW-TO(1))*CONV  
QRAR=(TO(1)-TSINK)*CRAD  
QNER=QCOR-QRAR  
DO 492 JJ=1,NNDS  
QQ=QQ+(T(JJ)-TO(JJ))*C(JJ)  
TO(JJ)=T(JJ)
```

492 CONTINUE

```
QNET=QCONV-QRAD  
QSTOR=QSTOR+QQ  
QSUB=QSUB+EXCHT-QADV  
QTOT=QSTOR+QSUB+QAD
```

C -----  
C -----  
C

C TIME STEP  
TIM=TIM+DTSM

C -----  
C -----  
C

### FIND HEAT RATES

```
QADR=QADV/DTSM  
QSTR=QQ/DTSM  
QSUR=EXCHT/DTSM-QADR  
QTOR=QSTR+QSUR+QADR  
RECR=(XX(1)-XXSV)/(TIM-TMPSV)
```

C -----  
C -----  
C

```
EXCHSV=EXCHT  
NIT=NIT+1  
IF(NIT.GE.NSTP)GO TO 8000  
IF(TIM.LE.TSTOP) GO TO 1000  
IF(METRIK.EQ.1)XMAS2=XMAS*4.8824  
IF(METRIK.EQ.0)WRITE(9,752)XMAS  
IF(METRIK.EQ.1)WRITE(9,753)XMAS2  
WRITE(9,760)
```

760 FORMAT('1')  
DO 3000 NDS1=1,NCDS

ND1=L(NDS1,1)

ND2=L(NDS1,2)

IF(MATS(1,ICD(NDS1),1).EQ.0)GO TO 770

770 IF(FLGG(ND1,1,1).EQ.1)WRITE(9,761)CHAR2(MATS(1,ICD(NDS1),1)),ND1

IF(FLGG(ND1,3,1).EQ.1)WRITE(9,761)CHAR2(MATS(1,ICD(NDS1),3)),ND1

771 IF(MATS(1,ICD(NDS1),2).EQ.0)GO TO 772

IF(FLGG(ND2,2,1).EQ.1)WRITE(9,761)CHAR2(MATS(1,ICD(NDS1),2)),ND2

772 IF(MATS(1,ICD(NDS1),3).EQ.0)GO TO 773

IF(FLGG(ND2,3,2).EQ.1)WRITE(9,761)CHAR2(MATS(1,ICD(NDS1),3)),ND2

761 FORMAT(/,1X,'MAXIMUM TEMPERATURE OF ',A10,' EXCEEDED AT NODE ',

\$ 13)

773 CONTINUE

3000 CONTINUE

8000 CONTINUE

711 FORMAT((5(3H T(, 13, 2H)=,F8.3,2X,A1,4X,:)))

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```
726 FORMAT((5(4H XX(,13, 2H)=,F8.3,6X,:)))
712 FORMAT((5(3H C(, 13, 2H)=, E10.3,:, 5X)))
713 FORMAT((5(4H CD(,13, 2H)=, E10.3,:, 4X)))
749 FORMAT(1H1)
750 FORMAT(//,1H ,7HTIME = ,F12.5,5X,12HTIME STEP = ,F12.5,5X,
$      15HNO. OF STEPS = ,110)
752 FORMAT(//,' INITIAL MASS = ',F11.5,3X,'(LBM/SQ.FT.)',//)
753 FORMAT(//,' INITIAL MASS = ',F11.5,3X,'(KGM/SQ.M.)',//)
720 FORMAT(1H1,5X,'TSTART = ',F12.3,5X,'TSTOP = ',F12.3,5X,
$      'TIMPT = ',F12.3)
722 FORMAT(1H ,5X,'DTIM = ',F12.3,5X,'STAB = ',F12.3,5X,
$      'TOL = ',F12.3,5X,'BET = ',F12.3)
723 FORMAT(1H ,5X,'NBP = ',3X,15,9X,'NEXT = ',3X,15,9X,
$      'NSTP = ',18,9X,'IPFLAG = ',18,/)

    WRITE(11IN2,724)
724 FORMAT(//,15X,'--- EXECUTION COMPLETE ---')
    WRITE(11IN2,725)FNAM1
725 FORMAT(/,1X,'OUTPUT FILENAME = ',A20,/)

CLOSE(UNIT=7,STATUS='KEEP')
CLOSE(UNIT=9,STATUS='KEEP')
CALL EXIT
END
```

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SUBROUTINE ABSUB(11)

C SUBROUTINE TO COMPUTE RECESSION RATE OF ABLATOR AND ALSO  
C THE HEAT REQUIRED TO CAUSE THE MELT LINE TO REcede.

PARAMETER (NMB1=20,NMB2=6,NMB3=3,NMB4=6,NMB5=40,NMB6=10)

COMMON/INIT/TSTART,TSTOP,TIMPT,DTIM,NBP,NEXT,METRIC,  
\$ METRIK,METRIX,NSTP,IPFLAG,TINI(NMB1),SINKT(NMB1),XFIJ(NMB1),  
\$ MBP(NMB1),IN,IN2

COMMON/SUBLM/TSUB,XL,XLP,EXCHT,NAB,ISTAR,NDIV,IAB,EXCHSV,QADV,  
\$ QADVS,TMSV,IDROP

COMMON/TIME/NNDS,CONV,CRAD,STAB,ISBFG,NCDS

COMMON/ARA/T(NMB5),TO(NMB5),C(NMB5),CD(NMB5),ICD(NMB5),L(NMB5,2)

COMMON/LD/LS(NMB1,NMB2),XP(NMB1,NMB2,NMB4),MATS(NMB1,NMB2,NMB3),  
\$ NS(NMB1)

COMMON/TAX/TK(NMB2),XX(NMB5)

COMMON/NODES/NN,I,TT,TINIT,TSINK,FIJ,TMPMAX(NMB6)

COMMON/PICT/NNI(NMB1,NMB2)

COMMON/PRESS/PRES

COMMON/SAVE/XEND1,XEND2,XTST1,XTST2,XLTS,XMIN

COMMON/TITLE/CHAR2,CHAR1,FNAM1,FNAM3

CHARACTER\*10 CHAR2(NMB6)

CHARACTER\*13 CHAR1(NMB6)

CHARACTER\*20 FNAM1,FNAM3

DIMENSION ZXX(NMB5)

IF(NAB.EQ.0) GO TO 1000

JJ=ICD(11)

IIP1=11+1

JJP1=ICD(IIP1)

DIV=FLOAT(NDIV)

JN=LS(1,JJ)

N1=L(11,1)

N2=L(11,2)

N3=L(IIP1,2)

MA=MATS(1,JJ,1)

C COMPUTE RECESSION DISTANCE

QADVS=QADV

CALL PROP(TT,PRES,MA,RO,CP,XK,EP)

DS=EXCHT/(XLP\*RO)

EXCHSV=EXCHT

IDROP=0

IF(ISTAR.NE.0)GO TO 555

ISTAR=1

C COMPUTE NODE BOUNDARIES

XEND1=(XX(N2)+XX(N1))/2.0

XEND2=(XX(N3)+XX(N2))/2.0

XMIN=XEND1/DIV

555 CONTINUE

C MOVE NODE LOCATIONS

XX(N1)=XX(N1)+DS

XX(N2)=XX(N2)+DS/3.0

XP(I,JJ,1)=XP(I,JJ,1)-DS

XTST1=XEND1-XX(N1)

XTST2=2.0\*DS/3.0

XLTS=XX(N2)-XX(N1)

IF(XLTS.GT.XMIN)GO TO 560

XTST2=XEND2-XEND1

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1DROP=1  
560 CONTINUE  
XEND1=XEND1+XTST2  
C COMPUTE EFFECTIVE HEAT OF ABLATION  
QTEL=RO\*(XL\*XTST1+XL\*XTST2+CP\*XTST2\*(TSUB-T0(N2)))  
XL=QTEL/(RO\*(XTST1+XTST2))  
C COMPUTE ADVECTED HEAT  
QADV=RO\*CP\*XTST2\*(TSUB-T0(N2))  
IF(1DROP.EQ.1)GO TO 520  
GO TO 1000  
520 CONTINUE  
C RENUMBER NODES IF NODE DROPPED  
NNDS=NNDS-1  
NCDS=NCDS-1  
DO 521 KK=1,NNDS  
IF(KK.EQ.1) GO TO 521  
KKP1=KK+1  
XX(KK)=XX(KKP1)  
C(KK)=C(KKP1)  
T0(KK)=T0(KKP1)  
T(KK)=T(KKP1)  
521 CONTINUE  
XEND2=(XX(N3)+XX(N2))/2.0  
DO 522 KK=1,NCDS  
IF(KK.EQ.1) GO TO 522  
KKP1=KK+1  
CD(KK)=CD(KKP1)  
ICD(KK)=ICD(KKP1)  
522 CONTINUE  
WRITE(9,700)  
NNIS(I,JJ)=NNIS(I,JJ)-1  
C PRINT PICTURE OF NEW CONFIGURATION  
CALL PICTUR(9,1)  
IF(NCDS.EQ.1)GO TO 2000  
IF(JJ.NE.JJP1)GO TO 2000  
1000 CONTINUE  
700 FORMAT(1H ,//,' NODE DROPPED FROM SUBLIMER-ABLATOR MODEL',//)  
GO TO 3000  
2000 CONTINUE  
WRITE(11,N2,2003)FNAM1  
2003 FORMAT(//,1X,'RUN STOPPED DUE TO INSUFFICIENT ABLATIVE MATERIAL',  
\$ ' LEFT',//,1X,'OUTPUT FILE = ',A20)  
STOP  
3000 CONTINUE  
RETURN  
END

SUBROUTINE COMPO

C SUBROUTINE TO COMPUTE VALUES OF THERMAL CAPACITORS AND CONDUCTORS  
 PARAMETER (NMB1=20,NMB2=6,NMB3=3,NMB4=6,NMB5=40,NMB6=10)  
 COMMON/CAC/NEXFG,NIT,XMAS  
 COMMON/GAP/T1,T2,TH1,TH2,TH3,TH,P,H,M1,M2,M3,TOL,BET,SIG,XM,  
 \$ CAP1,CAP2,XK  
 COMMON/TAX/ TK(NMB2),XX(NMB5)  
 COMMON/TIME/NNDS,CONV,CRAD,STAB,ISBFG,NCD\$  
 COMMON/ARA/T(NMB5),TO(NMB5),C(NMB5),CD(NMB5),ICD(NMB5),L(NMB5,2)  
 COMMON/LD/LS(NMB1,NMB2),XP(NMB1,NMB2,NMB4),MATS(NMB1,NMB2,NMB3),  
 \$ NS(NMB1)  
 COMMON /NODES/NN,I,TT,TINIT,TSINK,FIJ,TMPMAX(NMB6)  
 COMMON/PRESS/PRES  
 NEXFG=0  
 DO 176 K=1,NNDS  
 C(K)=0.0

176 CONTINUE  
 DO 177 K=1,NCD\$  
 CD(K)=0.0

177 CONTINUE  
 DO 226 II=1,NCD\$  
 JJ=ICD(II)  
 JN=LS(I,JJ)  
 N1=L(II,1)  
 N2=L(II,2)  
 MA=MATS(I,JJ,1)  
 IF(JN.EQ.6) GO TO 227  
 IF(JN.EQ.1) GO TO 225  
 IF(JN.EQ.7) GO TO 225  
 GO TO 227

225 CONTINUE

C COMPUTE CAPACITANCE AND CONDUCTANCE OF SLAB AND ABLATOR NODES  
 TT=(TO(N1)+TO(N2))/2.0  
 CALL PROP(TT,PRES,MA,R0,CP,XK,EP)  
 DI=XX(N2)-XX(N1)  
 CTM=DI\*R0\*CP/2.0  
 CMAS=DI\*R0  
 IF(NIT.NE.0)CMAS=0.0  
 XMAS=XMAS+CMAS  
 C(N1)=C(N1)+CTM  
 C(N2)=C(N2)+CTM  
 CD(II)=XK/DI  
 GO TO 226

227 CONTINUE

C LOAD GEOMETRY AND MATERIAL NUMBERS INTO COMMON - GAP  
 CALL LOAD(I,JJ,N1,N2)  
 ITST=JN-1

C COMPUTE EQUIVALENT CONDUCTIVITY AND CAPACITANCE OF ALL  
 C OTHER STRUCTURES  
 GO TO (1,2,3,4,5),ITST

1 CONTINUE  
 CALL RGAP  
 GO TO 7

2 CONTINUE  
 CALL HONEY

3 GO TO 7  
CONTINUE  
CALL DORG  
GO TO 7  
4 CONTINUE  
CALL STAND  
GO TO 7  
5 CONTINUE  
CALL THINS  
7 CONTINUE  
 $C(N1)=C(N1)+CAP1$   
 $C(N2)=C(N2)+CAP2$   
IF(NIT.NE.0)XM=0.0  
C SUM MASS OF STRUCTURE  
XMAS=XMAS+XM  
CD(11)=XK  
226 CONTINUE  
RETURN  
END

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SUBROUTINE COMTMP

C THIS SUBROUTINE COMPUTES THE TEMPERATURES  
PARAMETER (NMB1=20,NMB2=6,NMB3=3,NMB4=6,NMB5=40,NMB6=10)  
COMMON/LD/LS(NMB1,NMB2),XP(NMB1,NMB2,NMB4),MATS(NMB1,NMB2,NMB3),  
\$ NS(NMB1)  
COMMON/NODES/NN,I,TT,TINIT,TSINK,FIJ,TMPMAX(NMB6)  
COMMON/CTMP/TAW,DTSM  
COMMON/TIME/NNDS,CONV,CRAD,STAB,ISBFG,NCDS  
COMMON/ARA/T(NMB5),TO(NMB5),C(NMB5),CD(NMB5),ICD(NMB5),L(NMB5,2)  
COMMON/SUBLM/TSUB,XL,XLP,EXCHT,NAB,ISTAR,NDIV,IAB,EXCHSV,QADV,  
\$ QADVS,TMSV,IDROP  
NAB=0  
DO 456 JJ=1,NNDS

C CHECK TO SEE IF THIN SKIN SECTIONS EXIST ANYWHERE IN STRUCTURE  
IF(ISBFG.EQ.0)GO TO 615

C -----

C THIN SKIN LAYER

JM2=JJ-2  
JP2=JJ+2  
JP1=JJ+1  
JM1=JJ-1  
IF(JJ.EQ.NNDS)GO TO 599  
IF(CD(JJ).GT.1.0E9)GO TO 601  
IF(JJ.EQ.1)GO TO 615

599 CONTINUE  
IF(CD(JM1).GT.1.0E9)GO TO 610  
GO TO 615

601 CONTINUE

C NODE ABOVE THIN SECTION  
CC=C(JJ)/(C(JP1)+C(JJ))  
IF(JJ.EQ.1)GO TO 602  
IT=NNDS-1  
A1=TO(JM1)\*CD(JM1)  
A3=TO(JJ)\*CD(JM1)  
IF(JJ.EQ.IT)GO TO 603  
A2=TO(JP2)\*CD(JP1)  
A4=TO(JJ)\*CD(JP1)  
GO TO 461

603 CONTINUE  
A2=0.0  
A4=0.0  
GO TO 461

602 CONTINUE

C THIN SKIN ON SURFACE, NODE ABOVE THIN SECTION  
A1=TAW\*CONV+TSINK\*CRAD  
A3=TO(JJ)\*CONV+TO(JJ)\*CRAD  
IF(NNDS.EQ.2)GO TO 604  
A4=TO(JJ)\*CD(JP1)  
A2=TO(JP2)\*CD(JP1)  
GO TO 461

604 CONTINUE  
A4=0.0  
A2=0.0  
GO TO 461

610 CONTINUE

C NODE BELOW THIN SECTION  
CC=C(JJ)/(C(JM1)+C(JJ))  
IF(JJ.EQ.2)GO TO 612  
A1=TO(JM2)\*CD(JM2)  
A3=TO(JJ)\*CD(JM2)  
IF(JJ.EQ.NNDS)GO TO 613  
A2=TO(JP1)\*CD(JJ)  
A4=TO(JJ)\*CD(JJ)  
GO TO 461

613 CONTINUE

A2=0.0

A4=0.0

GO TO 461

612 CONTINUE

C THIN SKIN ON SURFACE, NODE BELOW THIN SECTION

A1=TAW\*CONV+TSINK\*CRAD  
A3=TO(JJ)\*CONV+TO(JJ)\*CRAD  
IF(NNDS.EQ.2)GO TO 614  
A2=TO(JP1)\*CD(JJ)  
A4=TO(JJ)\*CD(JJ)  
GO TO 461

614 CONTINUE

A4=0.0

A2=0.0

461 CONTINUE

F1=(A1+A2)\*CC

F2=(A3+A4)\*CC

GO TO 460

C -----

615 CONTINUE

C STANDARD HEAT BALANCE

IF(JJ.NE.1)GO TO 457

C SURFACE NODE

F1=TSINK\*CRAD+TAW\*CONV+TO(2)\*CD(1)  
F2=TO(1)\*(CRAD+CONV+CD(1))  
GO TO 460

457 CONTINUE

IF(JJ.NE.NNDS)GO TO 458

C LAST NODE

JM1=JJ-1

F1=TO(JM1)\*CD(JM1)

F2=TO(NNDS)\*CD(JM1)

GO TO 460

458 CONTINUE

C GENERAL NODE

JM1=JJ-1

JP1=JJ+1

F1=TO(JM1)\*CD(JM1)+TO(JP1)\*CD(JJ)

F2=TO(JJ)\*(CD(JM1)+CD(JJ))

460 CONTINUE

C COMPUTE TEMPERATURES

T(JJ)=TO(JJ)+(F1-F2)\*(DTSM/C(JJ))

C CHECK TO SEE IF SUBLIMER TEMPERATURE HAS BEEN EXCEEDED

IF(JJ.NE.1) GO TO 456

IF(LS(1,1).NE.7) GO TO 456

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EXCHT=0.0  
IF(T(JJ).LE.TSUB) GO TO 456  
NAB=1  
EXCHT=(T(JJ)-TSUB)\*C(JJ)  
T(JJ)=TSUB  
456 CONTINUE  
RETURN  
END

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SUBROUTINE CORG  
C SUBROUTINE COMPUTES EFFECTIVE THERMAL CONDUCTIVITY, CAPACITY,  
C AND MASS OF A CORRUGATED STRUCTURE  
COMMON/GAP/T1,T2,TH1,TH2,TH3,TH,P,M,M1,M2,M3,TOL,BET,SIG,  
S XM,CAP1,CAP2,XK  
COMMON/FACT/XX(10,2),YY(10,2)  
COMMON/SF/AR(10),EPP(10),F(10,10),ASF(10,10)  
COMMON/PRESS/PRES  
T30=(T1+T2)/2.0  
T3=T30  
CONK=1.0E8  
AKC=CONK\*2.0\*TH3  
P2=P/2.0  
TT1=T1  
TT2=T2  
TT3=(T1+T2)/2.0  
B1=P2/2.0  
B2=TH/2.0  
B3=SQRT(B1\*\*2+B2\*\*2)  
A1=P2  
A3=2.0\*B3  
VOL=TH1+TH2+(2.0\*B3)\*TH3/P2  
C START ITERATION FOR MIDPOINT TEMPERATURES  
DO 100 I=1,100  
CALL PROP(TT1,PRES,M1,RHO1,CP1,XK1,EPP(1))  
CALL PROP(TT2,PRES,M2,RHO2,CP2,XK2,EPP(2))  
CALL PROP(TT3,PRES,M3,RHO3,CP3,XK3,EPP(3))  
IF(I.NE.1)GO TO 101  
C SET COORDINATES FOR ENDS OF EACH OF THREE SURFACES FOR  
C RADIATION ENCLOSURE  
DO 102 II=1,3  
EPP(2)=1.0  
DO 103 JJ=1,2  
XT=0.0  
YT=TH  
IF(II.EQ.2.AND.JJ.EQ.1)YT=0.0  
IF(II.EQ.3.AND.JJ.EQ.1)YT=0.0  
IF(II.EQ.1.AND.JJ.EQ.2)XT=P2  
IF(II.EQ.3.AND.JJ.EQ.2)XT=P2  
XX(II,JJ)=XT  
YY(II,JJ)=YT  
103 CONTINUE  
102 CONTINUE  
C FIND GEOMETRIC VIEW FACTORS AND RADIANT INTERCHANGE FACTORS  
CALL VFAC(3)  
CALL SRIPF(3)  
A2F23=ASF(1,3)  
A1F13=ASF(1,3)  
101 CONTINUE  
AK1=XK1\*TH1/B1  
AK2=XK2\*TH2/B1  
AK3=XK3\*TH3/B3  
C COMPUTE EQUIVALENT CONDUCTOR  
C1=AK1\*AKC\*AK3/(AK1\*AKC+AK1\*AK3+AKC\*AK3)  
C2=AK2\*AKC\*AK3/(AK2\*AKC+AK2\*AK3+AKC\*AK3)

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C COMPUTE RADIATION CONDUCTOR  
 $C4=A1F13*SIG*(T1**2+T3**2)*(T1+T3)$   
 $C5=A2F23*SIG*(T2**2+T3**2)*(T2+T3)$   
C ITERATE ON T3  
 $T3N=(T1*C1+T2*C2+T1*C4+T2*C5)/(C1+C2+C4+C5)$   
 $T3=BET*T3N+(1.0-BET)*T3$   
 $TEST=ABS(T3-T30)/T3$   
IF (TEST.LT.TOL) GO TO 200  
 $T3=T3$   
 $T30=T3$   
100 CONTINUE  
GO TO 300  
200 CONTINUE  
 $T12=ABS(T1-T2)$   
 $T13=ABS(T1-T3)$   
C COMPUTE TOTAL HEAT TRANSFER  
 $Q=T13*(C1+C4)/P2$   
 $XK=Q/T12$   
C COMPUTE EQUIVALENT CONDUCTIVITY  
 $XM=TH1*RHO1+TH2*RHO2+(2.0*B3)*TH3*RHO3/P2$   
C COMPUTE CAPACITORS  
 $CAP1=(VOL/2.0)*RHO1*CP1$   
 $CAP2=(VOL/2.0)*RHO2*CP2$   
300 CONTINUE  
RETURN  
END

SUBROUTINE DATA1

C SUBROUTINE TO READ AND STORE THERMOPHYSICAL PROPERTY DATA

PARAMETER (NMB1=20,NMB2=6,NMB3=3,NMB4=6)

PARAMETER (NMB8=41,NMB9=41,NMB6=10,NMB11=20,NMB12=8)

COMMON/DTA/CC(NMB8,NMB9),BSV(NMB8,NMB11,NMB12)

COMMON/CSUB/COS(2,NMB9)

COMMON/LD/LS(NMB1,NMB2),XP(NMB1,NMB2,NMB4),MATS(NMB1,NMB2,NMB3),  
\$ NS(NMB1)

COMMON/TITLE/CHAR2,CHAR1,FNAM1,FNAM3

COMMON/NODES/NN,I,TT,TINIT,TSINK,FIJ,TMPMAX(NMB6)

COMMON/INIT/TSTART,TSTOP,TIMPT,DTIM,NBP,NEXT,METRIC,  
\$ METRIK,METRIX,NSTP,IPFLAG,TINI(NMB1),SINKT(NMB1),XFIJ(NMB1),  
\$ MBP(NMB1),LIN,LIN2

CHARACTER\*10 CHAR2(NMB6),TEST1

CHARACTER\*13 CHAR1(NMB6),TEST2

CHARACTER\*20 FNAM1,FNAM3

DIMENSION MTST(NMB6),BB(6),ARD(8),ARDS(8)

OPEN(UNIT=8,NAME='INP1.DAT',TYPE='OLD',RECORDSIZE=132)

WRITE(9,703)

DO 471 JJ=1,NMB6

MTST(JJ)=0

471 CONTINUE

IC=0

NLA=NS(1)

C LOOP 1 TO NUMBER OF LAYERS

DO 400 LT=1,NLA

C LOOP 1 TO NUMBER OF MATERIALS PER LAYER(MAX)

DO 500 IM=1,NMB3

MA=MATS(I,LT,IM)

IF(MA.EQ.0)GO TO 500

100 CONTINUE

READ(8,701)KD

701 FORMAT(13,2X,15,4X,A10,1X,A13,E10.0)

707 FORMAT(5X,15,4X,A10,1X,A13)

IF(KD.LT.0)GO TO 300

C CHECK TO SEE IF MATERIAL NUMBER MATCHES

IF(KD.NE.MA)GO TO 100

BACKSPACE (UNIT=8)

READ(8,701)KD,JD,TEST1,TEST2,TMPMXA

DO 351 K5=1,NMB6

KSV=K5

C CHECK TO SEE IF MATERIAL HAS BEEN USED

IF(MA.EQ.MTST(K5))GO TO 352

351 CONTINUE

IC=IC+1

MTST(IC)=MA

C RENUMBER MATERIAL IDENTIFIERS

MATS(I,LT,IM)=IC

TMPMAX(IC)=TMPMXA

GO TO 353

352 CONTINUE

MATS(I,LT,IM)=KSV

REWIND (UNIT=8)

GO TO 500

353 CONTINUE

C STORE TITLES  
CHAR2(1C)=TEST1  
CHAR1(1C)=TEST2  
DO 250 IB=1,4  
IF(IB.EQ.1)GO TO 251  
C READ TABLE TITLE FOR TABLE 2,3, AND 4  
READ(8,707)JD,TEST1,TEST2  
251 CONTINUE  
C STORE NUMBER OF ENTRIES  
IT=(1C-1)\*4+IB  
CC(IT,1)=FLOAT(JD)  
DO 200 ITT=1,JD  
READ(8,702)(ARD(MR),MR=1,8)  
IF(ARD(1).LT.0)GO TO 600  
IF(ITT.GT.1)GO TO 210  
C STORE MAXIMUM TEMPERATURE FOR PRINTING IN DESIRED UNITS  
IF(METRIK.EQ.0)TEMMAX=TMPMAX(1C)-459.6  
IF(METRIK.EQ.1)TEMMAX=TMPMAX(1C)/1.8  
IF(IB.EQ.1.AND.METRIK.EQ.0)WRITE(9,800)TEST1,KD,TEMMAX,TEST2  
800 FORMAT(//,8X,A10,' - MAT NO. ',12,/, ' MAXIMUM TEMPERATURE',11X,  
\$ F7.2,' DEG F',//,5X,'TEMP.',9X,A13,/,.4X,'(DEG F)',6X,  
\$ '(LBM/CU.FT)',/)  
IF(IB.EQ.1.AND.METRIK.EQ.1)WRITE(9,801)TEST1,KD,TEMMAX,TEST2  
801 FORMAT(//,8X,A10,' - MAT NO. ',12,/, ' MAXIMUM TEMPERATURE',11X,  
\$ F7.2,' DEG K',//,5X,'TEMP.',9X,A13,/,.4X,'(DEG K)',6X,  
\$ '(KGM/CU.M.)',/)  
IF(IB.EQ.2.AND.METRIK.EQ.0)WRITE(9,802)TEST2  
802 FORMAT(//,5X,'TEMP.',7X,A13,/,.4X,'(DEG F)',5X,'(BTU/LBM-DEG F)',/)  
IF(IB.EQ.2.AND.METRIK.EQ.1)WRITE(9,803)TEST2  
803 FORMAT(//,5X,'TEMP.',7X,A13,/,.4X,'(DEG K)',4X,  
\$ '(JOULES/KGM-DEG K)',/)  
IF(IB.EQ.3.AND.METRIK.EQ.0)WRITE(9,804)TEST2  
804 FORMAT(//,5X,'TEMP.',7X,A13,/,.4X,'(DEG F)',4X,  
\$ '(BTU/FT-S-DEG F)',/)  
IF(IB.EQ.3.AND.METRIK.EQ.1)WRITE(9,805)TEST2  
805 FORMAT(//,5X,'TEMP.',7X,A13,/,.4X,'(DEG K)',4X,'(WATTS/M-DEG K)',/)  
IF(IB.EQ.4.AND.METRIK.EQ.0)WRITE(9,806)TEST2  
806 FORMAT(//,5X,'TEMP.',8X,A13,/,.4X,'(DEG F)',4X,'(DIMENSIONLESS)',/)  
IF(IB.EQ.4.AND.METRIK.EQ.1)WRITE(9,807)TEST2  
807 FORMAT(//,5X,'TEMP.',8X,A13,/,.4X,'(DEG K)',4X,'(DIMENSIONLESS)',/)  
210 CONTINUE  
A=ARD(1)  
B=ARD(2)  
IF(METRIK.EQ.0)AAA=A-459.6  
IF(METRIK.EQ.1)AAA=A/1.8  
BBB=B  
IF(METRIK.EQ.1.AND.IB.EQ.1)BBB=B\*16.018067  
IF(METRIK.EQ.1.AND.IB.EQ.2)BBB=B\*4187.6  
IF(METRIK.EQ.1.AND.IB.EQ.3)BBB=B\*6228.343  
WRITE(9,705)AAA,BBB  
705 FORMAT(1X,E12.4,3X,E12.4)  
K1=ITT\*2  
K2=K1+1  
C STORE INDEPENDENT AND DEPENDENT ARRAYS  
CC(IT,K1)=A

```

CC(IT,K2)=B
200 CONTINUE
GO TO 250
600 CONTINUE
C BIVARIAT TABLE
  IARDS=1FIX(-ARD(1))
  IF(METRIK.EQ.1)GO TO 605
  WRITE(9,808)(ARD(MR),MR=2,IARDS+1)
808 FORMAT(//,35X,'CONDUCTIVITY',,33X,'(BTU/FT-S-DEG F)',,
$ //,5X,'TEMP. PRESSURE (LB/SQ.FT)',,4X,'(DEG F)',,
$ 7(4X,F7.2,2X))
  GO TO 610
605 CONTINUE
  DO 608 MR1=2,IARDS+1
606 ARDS(MR1)=ARD(MR1)*47.88
  WRITE(9,809)(ARDS(MR),MR=2,IARDS+1)
809 FORMAT(//,35X,'CONDUCTIVITY',,33X,'(WATTS/M-DEG K)',,
$ //,5X,'TEMP. PRESSURE (N/SQ.M)',,4X,'(DEG K)',,
$ 7(3X,F9.2,1X))
610 CONTINUE
  WRITE(9,810)
810 FORMAT(13X,'')
C STORE NUMBER OF PRESSURES
  CC(IT,2)=ARD(1)
  NARD=-ARD(1)+2
C STORE PRESSURES
  DO 601 IKS=3,NARD
    IM1=IKS-1
    CC(IT,IKS)=ARD(IM1)
601 CONTINUE
  NSTR=NARD+1
  NSS=NARD-2
C READ REST OF BIVARIAT TABLE
  DO 602 I6=1,JD
  READ(8,702)(ARD(MR),MR=1,IARDS+1)
  DO 620 MKR=1,IARDS+1
    IF(METRIK.EQ.0)ARDS(1)=ARD(1)-459.6
    IF(METRIK.EQ.1)ARDS(1)=ARD(1)/1.8
    ARDS(MKR)=ARD(MKR)
    IF(METRIK.EQ.1)ARDS(MKR)=ARD(MKR)*6231.1
620 CONTINUE
  WRITE(9,811)(ARDS(MKR),MKR=1,IARDS+1)
811 FORMAT(8(2X,E11.4))
C SAVE TEMPERATURES
  CC(IT,NSTR)=ARD(1)
  NSTR=NSTR+1
  DO 603 I7=1,NSS
    I71=I7+1
C SAVE DEPENDENT VARIABLE
  BSV(IT,I6,I7)=ARD(I71)
603 CONTINUE
602 CONTINUE
250 CONTINUE
  READ(8,701,END=1000)KDS,JD,TEST1,TEST2,TMPMXA
  IF(KDS.NE.KD)GO TO 1000

```

C STORE TWO ABLATOR PROPERTIES

```
DO 1100 IM2=1,2
CCS(IM2,1)=FLOAT(JD)
IF(IM2.EQ.1.AND.METRIK.EQ.0)WRITE(9,830)TEST2
830 FORMAT(//,4X,'PRESSURE',5X,A13,/,,3X,'(LB/SQ.FT)',5X,
$ '(DEG F)',/)
IF(IM2.EQ.1.AND.METRIK.EQ.1)WRITE(9,831)TEST2
831 FORMAT(//,4X,'PRESSURE',5X,A13,/,,4X,'(N/SQ.M)',7X,
$ '(DEG K)',/)
IF(IM2.EQ.2.AND.METRIK.EQ.0)WRITE(9,832)TEST2
832 FORMAT(//,4X,'PRESSURE',5X,A13,/,,3X,'(LB/SQ.FT)',5X,
$ '(BTU/LBM)',/)
IF(IM2.EQ.2.AND.METRIK.EQ.1)WRITE(9,833)TEST2
833 FORMAT(//,4X,'PRESSURE',5X,A13,/,,4X,'(N/SQ.M)',5X,
$ '(JOULES/KGM)',/)
DO 1101 ITT=1,JD
READ(8,702)A,B
IF(METRIK.EQ.0)AA1=A
IF(METRIK.EQ.1)AA1=A*47.08
IF(IM2.EQ.1.AND.METRIK.EQ.0)BB1=B-459.6
IF(IM2.EQ.1.AND.METRIK.EQ.1)BB1=B/1.8
IF(IM2.EQ.2.AND.METRIK.EQ.0)BB1=B
IF(IM2.EQ.2.AND.METRIK.EQ.1)BB1=B*2326.4
WRITE(9,705)AA1,BB1
K1=ITT*2
K2=K1+1
CCS(IM2,K1)=A
CCS(IM2,K2)=B
1101 CONTINUE
READ(8,701)KD,JD,TEST1,TEST2,TMPMXA
1100 CONTINUE
1000 CONTINUE
REWIND (UNIT=8)
GO TO 500
300 CONTINUE
WRITE(9,708)MA
708 FORMAT(1H , 'MATERIAL NUMBER',3X,15,3X,'CANNOT BE FOUND.')
REWIND (UNIT=8)
GO TO 500
500 CONTINUE
400 CONTINUE
REWIND (UNIT=8)
702 FORMAT(5X,8E10.0)
703 FORMAT(1H1,10X,11HT A B L E S)
CLOSE(UNIT=8,STATUS='KEEP')
RETURN
END
```

```

SUBROUTINE DATA2(LBP)
C SUBROUTINE TO READ AND STORE ENVIRONMENT FROM LANMIN FILE (UNIT 7)
PARAMETER (NMB1=100, NMB1=20)
COMMON/INIT/TSTART,TSTOP,TIMPT,DTIM,NBP,NEXT,METRIC,
$ METRIK,METRIX,NSTP,IPFLAG,TINI(NMB1),SINKT(NMB1),XFIJ(NMB1),
$ MBP(NMB1),IN,IN2
COMMON/ENVIR/TM1(NMB10),HC1(NMB10),HAW1(NMB10),PRES1(NMB10)
CHARACTER#72 DESCRP
WRITE(9,666)
666 FORMAT(1H1)
100 CONTINUE
IC=0
READ(7,700,END=1000)DESCRP,IBP
700 FORMAT(A72,15)
C CHECK FOR CORRECT BODY POINT NUMBER
IF(IBP.EQ.LBP)GO TO 900
50 CONTINUE
READ(7,701)D1,D2,D3,D4
701 FORMAT(2X,F6.1,39X,E10.3,2X,E10.3,36X,E10.3)
IF(D1.LT.0.0)GO TO 100
GO TO 50
900 CONTINUE
IF(METRIC.EQ.0)WRITE(9,750)IBP,DESCRP
750 FORMAT(1H,'BODY POINT NUMBER = ',15,5X,A72,//,10X,'TIME',9X,
$'FILM COEF.',5X,'REC ENTHALPY',6X,'PRESSURE',//,10X,'(SEC)',,
$6X,'(LBM/SQ.FT-SEC)',4X,'(BTU/LBM)',6X,'(LBF/SQ.FT)',/)
IF(METRIC.EQ.1)WRITE(9,850)IBP,DESCRP
850 FORMAT(1H,'BODY POINT NUMBER = ',15,5X,A72,//,10X,'TIME',9X,
$'FILM COEF.',5X,'REC ENTHALPY',6X,'PRESSURE',//,10X,'(SEC)',,
$7X,'(KGM/SQM-SEC)',4X,'(JOULES/KGM)',5X,'(N/SQ.M)',/)
901 CONTINUE
IC=IC+1
C READ LANMIN DATA
READ(7,701)TM1(IC),HC1(IC),HAW1(IC),PRES1(IC)
IF(TM1(IC).GE.0.0)WRITE(9,751)TM1(IC),HC1(IC),
$HAW1(IC),PRES1(IC)
IF(METRIC.EQ.0)GO TO 500
HC1(IC)=HC1(IC)/4.8824
HAW1(IC)=HAW1(IC)/2.32456E3
PRES1(IC)=PRES1(IC)/47.88
500 CONTINUE
751 FORMAT(1H,4(6X,E10.4))
IF(TM1(IC).GE.0.0)GO TO 901
REWIND (UNIT=7)
GO TO 1001
1000 CONTINUE
WRITE(9,752)LBP
752 FORMAT(1H,'CANNOT FIND BODY POINT ',15)
1001 CONTINUE
RETURN
END

```

C SUBROUTINE DIST(I1,J1,I2,J2,D)  
C SUBROUTINE TO COMPUTE DISTANCE BETWEEN TWO POINTS GIVEN  
C COORDINATES XX(I,J),YY(I,J)  
C I = SURFACE NO  
C J = 1 OR 2 ; END POINTS  
C  
COMMON/FACT/XX(10,2),YY(10,2)  
X1=XX(I1,J1)  
Y1=YY(I1,J1)  
X2=XX(I2,J2)  
Y2=YY(I2,J2)  
D=SQRT((X1-X2)\*\*2+(Y1-Y2)\*\*2)  
RETURN  
END

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SUBROUTINE HEATN(TIME,HC,HAW,PRES,ISV)  
C THIS ROUTINE DETERMINES FILM COEFFICIENT, ADIABATIC WALL ENTHALPY,  
C AND PRESSURE AS A FUNCTION OF TIME  
PARAMETER (NMB10=100)  
COMMON/ENVIR/TM1(NMB10),HC1(NMB10),HAW1(NMB10),PRES1(NMB10)  
11=ISV  
100 CONTINUE  
12=11+1  
IF(TM1(12).GT.TIME)GO TO 50  
11=11+1  
GO TO 100  
50 CONTINUE  
ISV=11  
DT=TM1(12)-TM1(11)  
DINC=(TIME-TM1(11))/DT  
HC=HC1(11)+(HC1(12)-HC1(11))\*DINC  
HAW=HAW1(11)+(HAW1(12)-HAW1(11))\*DINC  
PRES=PRES1(11)+(PRES1(12)-PRES1(11))\*DINC  
RETURN  
END

SUBROUTINE HONEY

C SUBROUTINE TO COMPUTE EQUIVALENT THERMAL CONDUCTIVITY, CAPACITY,  
C AND MASS OF HEXAGONAL HONEYCOMB STRUCTURE

```

COMMON/GAP/T1,T2,TH1,TH2,TH3,TH,P,H,M1,M2,M3,TOL,BET,SIG,
$ XM,CAP1,CAP2,XK
COMMON/PRESS/PRES
F12=.1
F13=.9

```

C SET GEOMETRIC PARAMETERS

```

D=H*3.0/(2.0*SQRT(3.0))
CPFT2=2.0*SQRT(3.0)/(H**2*3.0)
WPFT2=CPFT2*3.0
DWAL=D*2.0/3.0
VOL=(TH1+TH2)+WPFT2*(TH*TH3*DWAL)
A1=(D/3.0*H/2.0)/2.0
A2=A1
A3=DWAL*TH
T3=(T1+T2)/2.0
CON=1.0E8
CONK=CON*DWAL*TH3
T30=T3
DO 100 I=1,100
CALL PROP(T1,PRES,M1,RHO1,CP1,XK1,EP1)
CALL PROP(T2,PRES,M2,RHO2,CP2,XK2,EP2)
CALL PROP(T3,PRES,M3,RHO3,CP3,XK3,EP3)

```

C COMPUTE RADIANT INTERCHANGE FACTORS

```

F1=(1.0/A1)*(1.0/EP1-1.0)
F2=(1.0/A3)*(1.0/EP3-1.0)
F3=1.0/(A1*F13)
A1F13=1.0/(F1+F2+F3)
F2=(1.0/A2)*(1.0/EP2-1.0)
F3=1.0/(A1*F12)
A1F12=1.0/(F1+F2+F3)

```

C SET CONDUCTORS

```

C1=4.0*TH1*5.0*DWAL*XK1/H
C2=C1*XK2/XK1
C3=2.0*XK3*TH3*DWAL/TH
XC1=C1*CONK*C3/(C1*CONK+C1*C3+CONK*C3)
XC2=C2*CONK*C3/(C2*CONK+C2*C3+CONK*C3)
R1=2.0*A1F13*SIG*(T1**2+T3**2)*(T1+T3)
R2=2.0*A1F13*SIG*(T2**2+T3**2)*(T1+T3)
R3=2.0*A1F12*SIG*(T1**2+T2**2)*(T1+T2)

```

C ITERATE ON CELL WALL TEMPERATURE - T3

```

T3N=(T1*(XC1+R1)+T2*(XC2+R2))/(XC1+R1+XC2+R2)
T3=(1.0-BET)*T3+BET*T3N
TEST=ABS(T3-T30)/T3
IF(TEST.LT.TOL)GO TO 200
T30=T3
TT3=T3
100 CONTINUE
GO TO 300
200 CONTINUE
T12=ABS(T1-T2)
T13=ABS(T1-T3)

```

C COMPUTE TOTAL HEAT TRANSFER

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Q=(T12\*R3+T13\*(R2+XC2))\*WPFT2  
C COMPUTE CONDUCTIVITY, MASS, AND CAPACITANCE  
XK=Q/T12  
XM=TH1\*RHO1+TH2\*RHO2+RH03\*WPFT2\*TH\*TH3\*DVAL  
CAP1=(VOL/2.0)\*RHO1\*CP1  
CAP2=(VOL/2.0)\*RHO2\*CP2  
300 CONTINUE  
RETURN  
END

```

C SUBROUTINE INPGEQ
C SUBROUTINE FOR INTERACTIVE INPUT OF DATA FOR EXITS CODE
C INCLUDES INTERACTIVE INPUT OF STRUCTURES
PARAMETER (NMB1=20,NMB2=6,NMB3=3,NMB4=6,NMB6=10,NMB7=100)
COMMON/GAP/T1,T2,TH1,TH2,TH3,TH,P,H,M1,M2,M3,TOL,BET,SIG,XM,
$      CAP1,CAP2,XK
COMMON/INIT/TSTART,TSTOP,TIMPT,DTIM,NBP,NEXT,METRIC,
$ METRIK,METRIX,NSTP,IPFLAG,TINI(NMB1),SINKT(NMB1),XFIJ(NMB1),
$ MBP(NMB1),IN1,IN2
COMMON/TIME/NNDS,CONV,CRAD,STAB,ISBFG,NCDS
COMMON/LD/LS(NMB1,NMB2),XP(NMB1,NMB2,NMB4),MATS(NMB1,NMB2,NMB3),
$ NS(NMB1)
COMMON/PICT/NN1S(NMB1,NMB2)
COMMON/TITL2/CHAR3
COMMON/TITLE/CHAR2,CHAR1,FNAM1,FNAM3
CHARACTER*20 CHAR(7),FNAM1,FNAM2,FNAM3
CHARACTER*10 CHAR3(NMB7),CHAR2(NMB6)
CHARACTER*13 CHAR1(NMB6)
INTEGER ANS1,ANS2,ANS4,ANS5,ANS6,ANS7,ANS8,ANS9,ANS10,STRFLG
DATA CHAR/
1      'SLAB          ,
2      'RADIATION GAP  ,
3      'HONEY COMB    ,
4      'CORRUGATED   ,
5      'Z STANDOFF   ,
6      'THIN SKIN     ,
7      'ABLATOR SUBLIMER  ,
9      WRITE(IN2,10)
10     FORMAT(1H,'WHAT IS THE MINIVER INPUT DATA FILE NAME ??')
11     READ(IN,20,END=1234)FNAM2
20     FORMAT(A20)
12     OPEN(UNIT=7,NAME=FNAM2,TYPE='OLD',ERR=9,RECORDSIZE=80)
22     WRITE(IN2,23)
23     FORMAT(1H,'WHAT IS THE STRUCTURE FILE NAME ??')
24     READ(IN,20,END=1234)FNAM3
25     WRITE(IN2,30)
30     FORMAT(1H,'WHAT IS THE NAME OF THE OUTPUT FILE ??')
31     READ(IN,20,END=1234)FNAM1
32     OPEN(UNIT=9,NAME=FNAM1,TYPE='NEW',ERR=22,RECORDSIZE=132)

```

## DEFAULT VALUES FOR CONTROL PARAMETERS

```

DTIM=10.0
STAB=2.0
TOL=.001
BET=0.5
NEXT=20
NSTP=3000
IPFLAG=1
METRIC=0
METRIK=0
METRIX=0

```

C SET INITIAL, FINAL, AND DELTA PRINT TIMES

```
40 WRITE(11IN2,50)
50 FORMAT(1H ,!WHAT IS THE INITIAL TIME(SEC) ?)
```

60 READ(IIN,\*,ERR=40,END=1234)TSTART  
 70 WRITE(IIN2,70)  
 70 FORMAT(1H,'WHAT IS THE FINAL TIME(SEC) ?')  
 80 READ(IIN,\*,ERR=60,END=1234)TSTOP  
 80 WRITE(IIN2,80)  
 90 FORMAT(1H,'WHAT IS THE TIME(SEC) BETWEEN PRINTOUTS ?')  
 90 READ(IIN,\*,ERR=80,END=1234)T.MPT  
 100 WRITE(IIN2,110)  
 110 FORMAT(1H,'DO YOU WANT TO RESET CONTROL PARAMETERS ?')  
 110 READ(IIN,120,ERR=100,END=1234)ANS1  
 120 FORMAT(A1)  
 130 FORMAT(E20.10)  
 140 FORMAT(I10)  
 IF(ANS1.NE.1.HY)GO TO 190  
 C -----  
 C -----  
 150 WRITE(IIN2,170)DTIM  
 170 FORMAT(1H,'RÉSOLUTION: DEFAULT = ',F4.1,' NEW VALUE = ')  
 READ(IIN,130,ERR=160,END=1234)A1  
 IF(A1.GT..00001)DTIM=A1  
 180 WRITE(IIN2,190)STAB  
 190 FORMAT(1H,'STABILITY: DEFAULT = ',F3.1,' NEW VALUE = ')  
 READ(IIN,130,ERR=180,END=1234)A2  
 IF(A2.GT..00001)STAB=A2  
 200 WRITE(IIN2,210)TOL  
 210 FORMAT(1H,'ITERATION TOLERENCE: DEFAULT = ',F4.3,' NEW VALUE = ')  
 READ(IIN,130,ERR=200,END=1234)A3  
 IF(A3.GT..00001)TOL=A3  
 220 WRITE(IIN2,230)BET  
 230 FORMAT(1H,'RELAXATION FACTOR: DEFAULT = ',F3.1,' NEW VALUE = ')  
 READ(IIN,130,ERR=220,END=1234)A4  
 IF(A4.GT..00001)BET=A4  
 240 WRITE(IIN2,250)NEXT  
 250 FORMAT(1H,'NUMBER OF STEPS BETWEEN PARAMETER CALC.: DEFAULT = ',I2,  
 \$ ' NEW VALUE = ')  
 READ(IIN,140,ERR=240,END=1234)K1  
 IF(K1.GT.0)NEXT=K1  
 260 WRITE(IIN2,270)NSTP  
 270 FORMAT(1H,'MAXIMUM NUMBER OF ITERATIONS: DEFAULT = ',I4,  
 \$ ' NEW VALUE = ')  
 READ(IIN,140,ERR=260,END=1234)K2  
 IF(K2.GT.0)NSTP=K2  
 WRITE(IIN2,278)  
 278 FORMAT(/,  
 \$ 16X,'ENGLISH(DEFAULT)',5X,'METRIC',//,  
 \$ 1X,'TEMPÉRATURE',4X,'DEG F',16X,'DEG K',//,  
 \$ 1X,'LENGTH',9X,'INCHES',15X,'CM',//,  
 \$ 1X,'ENERGY',9X,'BTU',18X,'JOULES',//,  
 \$ 1X,'MASS',11X,'LBM',18X,'KGM',//)  
 275 WRITE(IIN2,277)FNAM2  
 277 FORMAT(1H,'ARE THE UNITS OF ',A20,' IN ENGLISH OR METRIC ?')  
 READ(IIN,120,ERR=275,END=1234)ANS8  
 IF(ANS8.EQ.1.HM)METRIC=1  
 281 WRITE(IIN2,282)  
 282 FORMAT(1H,'DO YOU WANT OUTPUT DATA IN ENGLISH OR METRIC ?')

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```
READ(IIN,120,ERR=281,END=1234)ANS8
IF(ANS8.EQ.1HM)METRIK=1
283 WRITE(IIN2,284)
284 FORMAT(1H,'DO YOU WANT INPUT DATA IN ENGLISH OR METRIC ?')
READ(IIN,120,ERR=283,END=1234)ANS8
IF(ANS8.EQ.1HM)METRIX=1
280 WRITE(IIN2,290)
290 FORMAT(1H,'DO YOU WANT ADDITIONAL PRINTOUT ?')
READ(IIN,120,ERR=280,END=1234)ANS7
IF(ANS7.EQ.1HY)IPFLAG=0
C END OF CONTROL PARAMETER LOOP
C -----
150 CONTINUE
300 CONTINUE
310 WRITE(IIN2,320)
320 FORMAT(1H,'WHAT IS THE TOTAL NUMBER OF BODY POINTS ?')
READ(IIN,*,ERR=310,END=1234)NBP
IF(NBP.GT.NMB1)WRITE(IIN2,330)
330 FORMAT(1X,'ERROR---NUMBER OF BODY PTS. EXCEEDS DIMENSIONING')
IF(NBP.GT.NMB1)GO TO 300
DO 1000 IB=1,NBP
C DEFINE STRUCTURE AND INITIAL CONDITIONS FOR THE CURRENT BODY POINT
340 CONTINUE
IF(IB.EQ.1)WRITE(IIN2,350)
IF(IB.NE.1)WRITE(IIN2,360)
360 FORMAT(1H,'WHAT IS THE NEXT BODY PT. NUMBER ?')
350 FORMAT(1H,'WHAT IS THE BODY POINT NUMBER ?')
READ(IIN,*,ERR=340,END=1234)MBP(IB)
IF(IB.EQ.1)GO TO 370
C IF STRUCTURE SAME AS FOR PREVIOUS B.P. USE SAME DATA
390 WRITE(IIN2,380)MBP(IB),MBP(IB-1)
380 FORMAT(1H,'DOES BODY PT. ',IB,', HAVE THE SAME DATA?',
$      ' REQUIREMENTS AS BODY PT. ',IB-1,', ?')
READ(IIN,120,ERR=390,END=1234)ANS5
IF(ANS5.EQ.1HY)GO TO 890
370 CONTINUE
C RESET TIME OR CONTROL PARAMETERS ?
C YES(Y) = RESET TIMING PARAMETERS
C TIME(T) = RESET TIMING PARAMETERS
C CONTROL(C) = RESET CONTROL PARAMETERS
410 WRITE(IIN2,420)
420 FORMAT(1H,'DO YOU WANT TO RESET THE TIME OR CONTROL ',
$'PARAMETERS ?')
READ(IIN,120,ERR=410,END=1234)ANS6
IF(ANS6.EQ.1HY)GO TO 40
IF(ANS6.EQ.1HT)GO TO 40
IF(ANS6.EQ.1HC)GO TO 160
C DEFINE INITIAL TEMPERATURE DATA FOR BODY POINT
430 WRITE(IIN2,400)MBP(IB)
400 FORMAT(1H,'WHAT IS THE INITIAL TEMPERATURE OF BODY PT. ',
$IB,', ?')
READ(IIN,*,ERR=430,END=1234)TINI(IB)
IF(METRIX.EQ.0)TINI(IB)=TINI(IB)+459.6
IF(METRIX.EQ.1)TINI(IB)=TINI(IB)*1.8
440 WRITE(IIN2,450)MBP(IB)
```

```

450 FORMAT(1H , 'WHAT IS THE SINK TEMPERATURE OF BODY PT. ',  

$15,' ?')  

READ(IIN,* ,ERR=440,END=1234)SINKT(IB)  

IF(METRIX.EQ.0)SINKT(IB)=SINKT(IB)+459.6  

IF(METRIX.EQ.1)SINKT(IB)=SINKT(IB)*1.8  

460 WRITE(IIN2,470)MBP(IB)  

470 FORMAT(1H , 'WHAT IS THE VIEW FACTOR FOR BODY PT. ',15,' ?')  

READ(IIN,* ,ERR=460,END=1234)XFIJ(IB)  

STRFLG=0  

C DEFINE STRUCTURE  

472 WRITE(IIN2,473)MBP(IB)  

473 FORMAT(1H , 'DOES THE STRUCTURE FOR BODY PT. ',15,  

$ ' EXIST IN THE STRUCTURE FILE ?')  

READ(IIN,120,ERR=472,END=1234)ANS9  

C OBTAIN STRUCTURE DATA FROM STRUCTURE FILE  

IF(ANS9.EQ.1HY)CALL STRUCT(1,IB)  

IF(ANS9.EQ.1HY)GO TO 830  

474 WRITE(IIN2,475)MBP(IB)  

475 FORMAT(1H , 'DO YOU WANT TO ADD THE STRUCTURE FOR BODY PT. ',  

$ 15,' TO THE STRUCTURE FILE ?')  

READ(IIN,120,ERR=474,END=1234)ANS10  

IF(ANS10.EQ.1HY)STRFLG=1  

480 CONTINUE  

C DEFINE NUMBER OF LAYERS  

490 WRITE(IIN2,500)MBP(IB)  

500 FORMAT(1H , 'HOW MANY LAYERS AT BODY PT. ',15,' ?')  

READ(IIN,* ,ERR=490,END=1234)NS(IB)  

IF(NS(IB).GT.NMB2)WRITE(IIN2,510)  

510 FORMAT(1X,'ERROR---NUMBER OF LAYERS EXCEEDS ARRAY DIMENSIONING')  

IF(NS(IB).GT.NMB2)GO TO 480  

C LOOP 1 - NUMBER OF LAYERS  

DO 2000 KK=1,NS(IB)  

C DEFINE STRUCTURE TYPE AND DIMENSIONS FOR EACH LAYER  

520 WRITE(IIN2,530)  

530 FORMAT(///, '           STRUCTURE TYPE   ---   NUMBER  ', //,  

$           '           SLAB           1  ', //,  

$           '           RADIATION GAP    2  ', //,  

$           '           HONEYCOMB      3  ', //,  

$           '           CORRUGATED      4  ', //,  

$           '           Z STANDOFF      5  ', //,  

$           '           THIN SKIN       6  ', //,  

$           '           ABLATOR SUBLIMER  7  ', //, //)  

540 WRITE(IIN2,550)KK,MBP(IB)  

550 FORMAT(1H , 'WHAT IS THE STRUCTURE TYPE NUMBER ',  

$ ' FOR LAYER ',12,' OF BODY PT. ',15,' ?')  

READ(IIN,* ,ERR=540,END=1234)LS(IB,KK)  

GO TO(560,570,580,590,600,610,620),LS(IB,KK)  

560 WRITE(IIN2,630)KK,MBP(IB)  

READ(IIN,* ,ERR=560,END=1234)MATS(IB,KK,1),XP(IB,KK,1)  

GO TO 640  

570 CONTINUE  

DO 3000 LLL=1,2  

650 WRITE(IIN2,660)LLL,KK,MBP(IB)  

READ(IIN,* ,ERR=650,END=1234)MATS(IB,KK,LLL),XP(IB,KK,LLL)  

3000 CONTINUE

```

670 WRITE(1IN2,680)KK,MBP(1B)  
680 FORMAT(1H,'WHAT IS THE STRUCTURE HEIGHT FOR LAYER ',  
\$ '12,', ' OF BODY PT. ',15,', '?' )  
READ(1IN,\*,ERR=670,END=1234)XP(1B,KK,4)  
GO TO 640  
580 CONTINUE  
DO 4000 LLL=1,3  
690 WRITE(1IN2,690)LLL,KK,MBP(1B)  
READ(1IN,\*,ERR=690,END=1234)MATS(1B,KK,LLL),XP(1B,KK,LLL)  
4000 CONTINUE  
700 WRITE(1IN2,710)KK,MBP(1B)  
READ(1IN,\*,ERR=700,END=1234)XP(1B,KK,4),XP(1B,KK,6)  
GO TO 640  
590 CONTINUE  
DO 4500 LLL=1,3  
720 WRITE(1IN2,720)LLL,KK,MBP(1B)  
READ(1IN,\*,ERR=720,END=1234)MATS(1B,KK,LLL),XP(1B,KK,LLL)  
4500 CONTINUE  
730 WRITE(1IN2,740)KK,MBP(1B)  
READ(1IN,\*,ERR=730,END=1234)XP(1B,KK,4),XP(1B,KK,5)  
GO TO 640  
600 CONTINUE  
DO 5000 LLL=1,3  
750 WRITE(1IN2,750)LLL,KK,MBP(1B)  
READ(1IN,\*,ERR=750,END=1234)MATS(1B,KK,LLL),XP(1B,KK,LLL)  
5000 CONTINUE  
760 WRITE(1IN2,770)KK,MBP(1B)  
READ(1IN,\*,ERR=760,END=1234)(XP(1B,KK,LLL+3),LLL=1,3)  
GO TO 640  
610 CONTINUE  
780 WRITE(1IN2,630)KK,MBP(1B)  
READ(1IN,\*,ERR=780,END=1234)MATS(1B,KK,1),XP(1B,KK,4)  
GO TO 640  
620 CONTINUE  
790 WRITE(1IN2,630)KK,MBP(1B)  
READ(1IN,\*,ERR=790,END=1234)MATS(1B,KK,1),XP(1B,KK,1)  
GO TO 640  
630 FORMAT(1H,'WHAT IS THE MAT. IDENTIFIER AND THE MAT.',  
\$ ' THICKNESS',/, ' FOR LAYER ',12,', ' OF BODY PT. ',15,', '?' )  
660 FORMAT(' WHAT IS THE MAT. IDENTIFIER AND THICKNESS OF MAT. ',  
\$ '12,/,' FOR LAYER ',12,', ' OF BODY PT. ',15,', '?' )  
710 FORMAT(1H,'WHAT IS THE STRUCTURE HEIGHT AND CELL DIMENSIONS',  
\$ ' OF LAYER ',12,', ' OF BODY PT. ',15,', '?' )  
740 FORMAT(1H,'WHAT IS THE STRUCTURE HEIGHT AND PITCH FOR LAYER ',  
\$ '12,', ' OF BODY PT. ',15,', '?' )  
770 FORMAT(1H,'WHAT IS THE STRUCTURE HEIGHT,PITCH,AND FLANGE',  
\$ ' WIDTH FOR LAYER ',12,', ' OF BODY PT. ',15,', '?' )  
640 CONTINUE  
800 WRITE(1IN2,665)KK,MBP(1B)  
665 FORMAT(1H,'ARE THERE ANY CORRECTIONS FOR LAYER ',  
\$ '12,', ' OF BODY POINT ',15,', '?' )  
READ(1IN,810,ERR=800,END=1234)ANS2  
810 FORMAT(A1)  
IF(ANS2.EQ.'1') GO TO 520  
2000 CONTINUE /\* CONTINUE WITH NEXT LAYER

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830 CONTINUE  
IF(1B.GT.1)GO TO 855  
C  
C OBTAIN MATERIAL NAMES TO MATCH MATERIAL NUMBERS(USED BY PICTURE)  
OPEN(UNIT=8,NAME='INP1.DAT',TYPE='OLD',RECORDSIZE=132)  
6000 CONTINUE  
READ(8,840)MNUMB  
IF(MNUMB.EQ.0)GO TO 6000  
IF(MNUMB.LT.0)GO TO 850  
BACKSPACE (UNIT=8)  
READ(8,840)MNUMB,CHAR3(MNUMB)  
840 FORMAT(13,11X,A10)  
GO TO 6000  
850 CONTINUE  
REWIND (UNIT=8)  
CLOSE(UNIT=8,STATUS='KEEP')  
C  
C  
IF(ANS9.EQ.1HY)GO TO 911  
855 CONTINUE  
DO 910 INC1=1,NMB2  
DO 910 INC2=1,NMB4  
IF(METRIX.EQ.0)XP(1B,INC1,INC2)=XP(1B,INC1,INC2)/12.  
IF(METRIX.EQ.1)XP(1B,INC1,INC2)=XP(1B,INC1,INC2)/2.54/12.  
910 CONTINUE  
911 CONTINUE  
C DISPLAY PICTURE OF COMPLETE STRUCTURE ON SCREEN  
CALL PICTUR(5,1B)  
860 WRITE(1IN2,870)MP(1B)  
870 FORMAT(//,1H,'ARE THERE ANY CORRECTIONS FOR BODY PT. 1,15,1 ?')  
READ(1IN,810,ERR=860,END=1234)ANS4  
IF(ANS4.EQ.1HY)GO TO 370  
C ADD STRUCTURE TO STRUCTURE FILE  
IF(STRFLG.EQ.1)CALL STRUCT(2,1B)  
GO TO 820  
890 CONTINUE  
C IF STRUCTURE FOR BODY POINT MATCHES STRUCTURE FOR PREVIOUS  
C BODY POINT THEN USE SAME STRUCTURE DATA  
TINI(1B)=TINI(1B-1)  
SINKT(1B)=SINKT(1B-1)  
XFIJ(1B)=XFIJ(1B-1)  
NS(1B)=NS(1B-1)  
DO 7000 KKKK=1,NS(1B-1)  
LS(1B,KKKK)=LS(1B-1,KKKK)  
DO 8000 LLLL=1,6  
XP(1B,KKKK,LLL)=XP(1B-1,KKKK,LLL)  
8000 CONTINUE  
DO 9000 MMMM=1,3  
MATS(1B,KKKK,MMM)=MATS(1B-1,KKKK,MMM)  
9000 CONTINUE  
7000 CONTINUE  
IF(ANS5.EQ.1HY)GO TO 911  
GO TO 830  
820 CONTINUE  
1000 CONTINUE

/\* CONTINUE WITH NEXT BODY POINT

880 WRITE(1 FN2,880)  
FORMAT(//,1X,'  
RETURN  
1234 CONTINUE  
STOP  
END

MODEL COMPLETE - - - - - GONE TO EXECUTE!)

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SUBROUTINE INTF(X,P,N,Y)

C SUBROUTINE TO INTERPOLATE MONO AND BIVARIATE TABLES FOR PROPERTIES  
PARAMETER (NMB8=41,NMB9=41,NMB11=20,NMB12=8)  
COMMON/DTA/CC(NMB8,NMB9),BSV(NMB8,NMB11,NMB12)

C CHECK TO SEE IF TABLE IS BIVARIATE  
IF(CC(N,2).LT.0.0)GO TO 2000

C MONOVARIATE INTERPOLATION - DATA STORED IN CC(N,J)  
J=CC(N,1)  
J1=J#2  
XLST=CC(N,2)  
XHST=CC(N,J1)  
IF(X.EQ.XLST)GO TO 700  
IF(X.LT.XLST)GO TO 100  
IF(X.GT.XHST)GO TO 200  
JT=4

500 CONTINUE  
IF(CC(N,JT)=X)300,600,400

300 CONTINUE  
JT=JT+2  
GO TO 500

400 CONTINUE  
N1=JT-2  
N2=N1+1  
N3=JT  
N4=N3+1  
GO TO 900

500 CONTINUE  
NT=JT+1  
Y=CC(N,NT)  
GO TO 1000

700 CONTINUE  
Y=CC(N,3)  
GO TO 1000

100 CONTINUE  
N1=2  
N2=3  
N3=4  
N4=5  
GO TO 900

200 CONTINUE  
N1=2#J-2  
N2=N1+1  
N3=2#J  
N4=N3+1

900 CONTINUE

C COMPUTE PROPERTY  
SL=(CC(N,N4)-CC(N,N2))/(CC(N,N3)-CC(N,N1))  
Y=CC(N,N2)+SL\*(X-CC(N,N1))

1000 CONTINUE  
GO TO 3000

2000 CONTINUE

C BIVARIATE TABLES

C INDEPENDENT VARIABLES IN CC(N,J)

C DEPENDENT VARIABLES IN BSV(N,JT,IL)

N9=-CC(N,2)+2.0  
N8=4  
450 CONTINUE  
IF(CC(N,N8)-P)351,352,353  
351 CONTINUE  
IF(N8.GE.N9)GO TO 375  
N8=N8+1  
GO TO 450  
352 CONTINUE  
GO TO 375  
353 CONTINUE  
N7=N8-1  
GO TO 560  
375 CONTINUE  
N7=N8  
PFAC=0.0  
GO TO 561  
560 CONTINUE  
C FIND INCREMENT IN PRESSURE DIRECTION  
PFAC=(P-CC(N,N7))/(CC(N,N8)-CC(N,N7))  
561 CONTINUE  
L8=-CC(N,2)+4.0  
L9=-CC(N,2)+2.0+CC(N,1)  
550 CONTINUE  
IF(CC(N,L8)-X)751,752,753  
751 CONTINUE  
IF(L8.GE.L9)GO TO 775  
L8=L8+1  
GO TO 550  
752 CONTINUE  
GO TO 775  
753 CONTINUE  
L7=L8-1  
GO TO 760  
775 CONTINUE  
L7=L8  
TFAC=0.0  
GO TO 761  
760 CONTINUE  
C FIND INCREMENT IN TEMPERATURE DIRECTION  
TFAC=(X-CC(N,L7))/(CC(N,L8)-CC(N,L7))  
761 CONTINUE  
IR=N8-2  
IL=N7-2  
JT=L7-N9  
JB=L8-N9  
F1=BSV(N,JT,IL)+PFAC\*(BSV(N,JT,IR)-BSV(N,JT,IL))  
F2=BSV(N,JB,IL)+PFAC\*(BSV(N,JB,IR)-BSV(N,JB,IL))  
C FIND PROPERTY  
Y=F1+TFAC\*(F2-F1)  
3000 CONTINUE  
RETURN  
END

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SUBROUTINE LOAD(MP,IS,IT1,IT2)  
C SUBROUTINE TO LOAD GEOMETRIC AND MATERIAL IDENTIFICATION  
C INTO COMMON GAP  
PARAMETER (NMB1=20,NMB2=6,NMB3=3,NMB4=6,NMB5=40)  
COMMON/GAP/T1,T2,X(6),M(3),TOL,BET,SIG,XM,CAP1,CAP2,XK  
COMMON/LD/LS(NMB1,NMB2),XP(NMB1,NMB2,NMB4),MATS(NMB1,NMB2,NMB3),  
\$ NS(NMB1)  
COMMON/ARA/T(NMB5),TO(NMB5),C(NMB5),CD(NMB5),ICD(NMB5),L(NMB5,2)  
DT=25.0  
T1=TO(IT1)+DT  
T2=TO(IT2)-DT  
C LOAD MATERIALS  
DO 100 I=1,3  
M(I)=MATS(MP,IS,I)  
100 CONTINUE  
C LOAD GEOMETRY  
DO 200 J=1,6  
X(J)=XP(MP,IS,J)  
200 CONTINUE  
RETURN  
END

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C SUBROUTINE NODE

C THIS SUBROUTINE SETS UP THE NODAL NETWORK

PARAMETER (NMB1=20, NMB2=6, NMB3=3, NMB4=6, NMB5=40, NMB6=10)

COMMON /NODES/NN, I, TT, TINIT, TSINK, F1J, TMPMAX(NMB6)

COMMON/GAP/T1, T2, TH1, TH2, TH3, TH, P, H, M1, M2, M3, TOL, BET, SIG, XM,

S CAP1, CAP2, XK

COMMON/INIT/TSTART, TSTOP, TIMPT, DTIM, NBP, NEXT, METRIC,

S METRIK, METRIX, NSTP, IPFLAG, TINI(NMB1), SINKT(NMB1), XF1J(NMB1),

S MBP(NMB1), LIN, LIN2

COMMON/TAX/ TK(NMB2), XX(NMB5)

COMMON/TIME/NNDS, CONV, CRAD, STAB, ISBFG, NCDS

COMMON/ARA/T(NMB5), TO(NMB5), C(NMB5), CD(NMB5), ICD(NMB5), L(NMB5,2)

COMMON/LD/LS(NMB1, NMB2), XP(NMB1, NMB2, NMB4), MATS(NMB1, NMB2, NMB3),

S NS(NMB1)

COMMON/PICT/NNIS(NMB1, NMB2)

COMMON/TITLE/CHAR2, CHAR1, FNAM1, FNAM3

COMMON/PRESS/PR<sub>E</sub>S

CHARACTER\*4 UNIT1(2)

CHARACTER\*6 UNIT2(2)

CHARACTER\*13 CHAR1(NMB6)

CHARACTER\*10 CHAR2(NMB6)

CHARACTER\*20 CHAR(7), FNAM1, FNAM3

DIMENSION XXX(NMB5)

DATA UNIT1// IN.!, CM.!

DATA UNIT2// DEG F!, DEG K!

DATA CHAR/

1 'SLAB

2 'RADIATION GAP

3 'HONEY COMB

4 'CORRUGATED

5 'Z STANDOFF

6 'THIN SKIN

7 'ABLATOR SUBLIMER

IC=0

METRIC=METRIK+1

DO 7000 J=1, NN

IST=LS(1, J)

IF(IST.EQ.1) GO TO 140

IF(IST.EQ.7) GO TO 140

GO TO 120

140 CONTINUE

C DIVIDE SLAB OR ABLATOR INTO NX-1 LAYERS

THKNS=XP(1, J, 1)

MA=MATS(1, J, 1)

CALL PROP(TT, PRES, MA, R0, CP, XK, EP)

DX=SQRT(DTIM\*2.0\*XK/(R0\*CP))

NX=THKNS/DX+1

IF(NX+1.GE.NMB5) GO TO 999

NNIS(1, J)=NX

TK(J)=THKNS/FLOAT(NX)

C ASSIGN NODE NUMBERS AND INITIAL TEMPERATURES, NODE NUMBERS FOR

C EACH CONDUCTOR, NODE POSITIONS FOR SLAB AND ABLATOR

DO 130 K=1, NX

IC=IC+1

ICD(IC)=J

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IU=IC
IL=IC+1
L(IC,1)=IU
L(IC,2)=IL
TO(IU)=TINIT
TO(IL)=TINIT
T(IU)=TO(IU)
T(IL)=TO(IL)
XX(IL)=XX(IU)+TK(J)
130 CONTINUE
GO TO 7000
120 CONTINUE
C ASSIGN NODE NUMBERS, INITIAL TEMPERATURES, NODE NUMBERS FOR EACH
C CONDUCTOR, NODE POSITIONS FOR ALL OTHER STRUCTURE TYPES
IC=IC+1
ICD(IC)=J
IU=IC
IL=IC+1
L(IC,1)=IU
L(IC,2)=IL
TO(IU)=TINIT
TO(IL)=TINIT
T(IU)=TO(IU)
T(IL)=TO(IL)
XX(IL)=XX(IU)+XP(I,J,4)
7000 CONTINUE
NNDS=IL
NCDS=IC
WRITE(9,769)
WRITE(9,768)MBP(I)
IF(METRIK.EQ.0)ATINIT=TINIT-459.6
IF(METRIK.EQ.1)ATINIT=TINIT/1.8
IF(METRIK.EQ.0)ATSINK=TSINK-459.6
IF(METRIK.EQ.1)ATSINK=TSINK/1.8
WRITE(9,721)ATINIT,UNIT2(METRC),ATSINK,UNIT2(METRC),FIJ
DO 137 KK=1,NCDS
NOD1=L(KK,1)
IF(METRIK.EQ.0)XXX(NOD1)=XX(NOD1)*12.
IF(METRIK.EQ.1)XXX(NOD1)=XX(NOD1)*12.*2.54
WRITE(9,770)NOD1,XXX(NOD1),UNIT1(METRC)
WRITE(9,771)KK
JST=ICD(KK)
KST=LS(I,JST)
WRITE(9,772)KST,CHAR(KST)
C WRITE OUT DESCRIPTION OF NETWORK
DO 429 LL=1,3
MA=MATS(I,JST,LL)
IF(MA.EQ.0)GO TO 429
WRITE(9,773)LL,CHAR2(MA)
429 CONTINUE
773 FORMAT(1H ,20X,'MATERIAL ',12,' = ',A10)
NOD2=L(KK,2)
IF(METRIK.EQ.0)XXX(NOD2)=XX(NOD2)*12.
IF(METRIK.EQ.1)XXX(NOD2)=XX(NOD2)*12.*2.54
WRITE(9,774)NOD2,XXX(NOD2),UNIT1(METRC)

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```
137    CONTINUE
721    FORMAT(1H ,5X,'TINIT = ',F7.2,A6,4X,'TSINK = ',F7.2,A6,4X,
$      'FIJ = ',F12.3)
768    FORMAT(1H ,28X,'BODY POINT',15)
769    FORMAT(1H1,25X,'STRUCTURE DEFINITION')
770    FORMAT(//,1H ,15X,'NODE NUMBER = ',15,7X,
$      'DISTANCE FROM SURFACE = ',E15.6,A4)
771    FORMAT(1M ,20X,'CONDUCTOR NUMBER = ',15)
772    FORMAT(1H ,20X,'STRUCTURE TYPE = ',19,5X,A20)
774    FORMAT(1H ,15X,'NODE NUMBER = ',15,7X,
$      'DISTANCE FROM SURFACE = ',E15.6,A4)
    RETURN
999    CONTINUE
    WRITE(11IN2,998)
    WRITE(9,998)
998    FORMAT(1X,'ERROR---THE NUMBER OF NODES EXCEEDS THE ARRAY ',
$      'DIMENSIONING.')
    STOP
    END
```

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SUBROUTINE PICTUR(1IN3,MP)

C SUBROUTINE THATS PRINTS A PICTURE OF THE STRUCTURE AT A BODY POINT  
PARAMETER (NMB1=20,NMB2=6,NMB3=3,NMB4=6,NMB6=10,NMB7=100)  
COMMON/LD/LS(NMB1,NMB2),XP(NMB1,NMB2,NMB4),MATS(NMB1,NMB2,NMB3),  
\$ NS(NMB1)  
COMMON/INIT/TSTART,TSTOP,TIMPT,DTIM,NBP,NEXT,METRIC,  
\$ METRIK,METRIX,NSTP,IPFLAG,TINI(NMB1),SINKT(NMB1),XF1J(NMB1),  
\$ MBP(NMB1),IIN,IIN2  
COMMON/PICT/NNIS(NMB1,NMB2)  
COMMON/TITLE/CHAR2M,CHAR1,FNAM1,FNAM3  
COMMON/TITL2/CHAR3  
CHARACTER\*4 UNIT(2)  
CHARACTER\*20 CHAR(7),FNAM1,FNAM3  
CHARACTER\*10 CHAR2(NMB7),CHAR3(NMB7),CHAR2M(NMB6)  
CHARACTER\*13 CHAR1(NMB6)  
DIMENSION ZXP(NMB1,NMB2,NMB4)  
INTEGER ANS1,ANS2,ANS4,ANS5  
DATA UNIT/' IN.!', ' CM.'/  
C NAME OF EACH STRUCTURE TYPE  
DATA CHAR/  
1 'SLAB  
2 'RADIATION GAP  
3 'HONEY COMB  
4 'CORRUGATED  
5 'Z STANDOFF  
6 'THIN SKIN  
7 'ABLATOR SUBLINER  
C CONVERT UNITS OF MAT DIMENSIONS FOR PICTURE  
DO 300 INC1=1,NMB2  
DO 300 INC2=1,NMB4  
IF(METRIK.EQ.0)ZXP(MP,INC1,INC2)=XP(MP,INC1,INC2)\*12.  
IF(METRIK.EQ.1)ZXP(MP,INC1,INC2)=XP(MP,INC1,INC2)\*12.\*2.54  
300 CONTINUE  
METRC=METRIK+1  
IF(IIN3.EQ.9)GO TO 3  
C IF PICTURE DISPLAYED TO SCREEN(NODES NOT INCLUDED)  
DO 2 IKJ=1,NMB7  
CHAR2(IKJ)=CHAR3(IKJ)  
2 CONTINUE  
GO TO 1  
3 CONTINUE  
C IF PICTURE PRINTED TO OUTPUT FILE(NODES INCLUDED)  
DO 4 IKJ=1,NMB6  
CHAR2(IKJ)=CHAR2M(IKJ)  
4 CONTINUE  
1 CONTINUE  
C CREATE THE PICTURE  
IJK=1  
WRITE(1IN3,619)MBP(MP)  
C TOP LAYER BOUNDARY  
IF(IIN3.EQ.5)WRITE(1IN3,620)  
IF(IIN3.EQ.9)WRITE(1IN3,720)IJK  
IJK=IJK+1  
C LOOP 1 - NUMBER OF LAYERS FOR THIS BODY POINT  
C J - LAYER NUMBER

C MP = BODY POINT NUMBER  
C IIN3 = 5 DISPLAY TO SCREEN (NODES AVAILABLE)  
C IIN3 = 9 PRINT TO OUTPUT FILE (NODES NOT AVAILABLE)  
DO 700 J=1,NS(MP)

C SLAB  
IF(LS(MP,J).EQ.1.AND.IIN3.EQ.5)WRITE(IIN3,621)CHAR2(MATS(MP,J,1)),  
\$ CHAR(LS(MP,J)),ZXP(MP,J,1),UNIT(METRC)  
IF(LS(MP,J).EQ.1.AND.IIN3.EQ.9)GO TO 100

C ABLATOR SUBLIMER  
IF(LS(MP,J).EQ.7.AND.IIN3.EQ.5)WRITE(IIN3,621)CHAR2(MATS(MP,J,1)),  
\$ CHAR(LS(MP,J)),ZXP(MP,J,1),UNIT(METRC)  
IF(LS(MP,J).EQ.7.AND.IIN3.EQ.9)GO TO 100

C RADIATION GAP  
IF(LS(MP,J).EQ.2)WRITE(IIN3,622)ZXP(MP,J,1),UNIT(METRC),  
\$ CHAR2(MATS(MP,J,1)),CHAR(LS(MP,J)),ZXP(MP,J,4),UNIT(METRC),  
\$ ZXP(MP,J,2),UNIT(METRC),CHAR2(MATS(MP,J,2))

C HONEY COMB  
IF(LS(MP,J).EQ.3)WRITE(IIN3,623)ZXP(MP,J,1),UNIT(METRC),  
\$ CHAR2(MATS(MP,J,1)),CHAR2(MATS(MP,J,3)),CHAR(LS(MP,J)),  
\$ ZXP(MP,J,4),UNIT(METRC),ZXP(MP,J,2),UNIT(METRC),  
\$ CHAR2(MATS(MP,J,2))

C CORRUGATED  
IF(LS(MP,J).EQ.4)WRITE(IIN3,624)ZXP(MP,J,1),UNIT(METRC),  
\$ CHAR2(MATS(MP,J,1)),CHAR2(MATS(MP,J,3)),CHAR(LS(MP,J)),  
\$ ZXP(MP,J,4),UNIT(METRC),ZXP(MP,J,2),UNIT(METRC),  
\$ CHAR2(MATS(MP,J,2))

C Z STANDOFF  
IF(LS(MP,J).EQ.5)WRITE(IIN3,625)ZXP(MP,J,1),UNIT(METRC),  
\$ CHAR2(MATS(MP,J,1)),CHAR2(MATS(MP,J,3)),CHAR(LS(MP,J)),  
\$ ZXP(MP,J,4),UNIT(METRC),ZXP(MP,J,2),UNIT(METRC),  
\$ CHAR2(MATS(MP,J,2))

C THIN SKIN  
IF(LS(MP,J).EQ.6)WRITE(IIN3,626)CHAR2(MATS(MP,J,1)),  
\$ CHAR(LS(MP,J)),ZXP(MP,J,4),UNIT(METRC)

200 CONTINUE

C BOTTOM LAYER BOUNDARY  
IF(IIN3.EQ.5)WRITE(IIN3,620)  
IF(IIN3.EQ.9)WRITE(IIN3,720)IJK  
IJK=IJK+1  
GO TO 700

100 CONTINUE

C SLAB / ABLATOR SUBLIMER (CONTINUED)  
NX=NNIS(MP,J)-1  
IF(NX.LE.0)GO TO 11  
NX02=NX/2+1  
DO 10 K=1,NX  
IF(K.EQ.1.AND.K.LT.NX02)WRITE(IIN3,800)IJK  
IF(K.GT.1.AND.K.LT.NX02)WRITE(IIN3,801)IJK  
IF(K.EQ.NX02)WRITE(IIN3,802)IJK,CHAR2(MATS(MP,J,1)),  
\$ CHAR(LS(MP,J)),ZXP(MP,J,1),UNIT(METRC)  
IF(K.GT.NX02.AND.K.LT.NX)WRITE(IIN3,801)IJK  
IF(K.GT.NX02.AND.K.EQ.NX)WRITE(IIN3,803)IJK  
IJK=IJK+1

10 CONTINUE  
11 CONTINUE

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SUBROUTINE PROP(T1,P,MAT,RHO,CP,XK,EP)  
UNITS ARE BTU,FT,SEC,OR,LBM  
TABLES ARE DENSITY,SP.HT.,CONDUCTIVITY,EMISSIVITY  
IN THAT ORDER AND REPEATED

T1 =TEMP.  
MAT=MATERIAL NUMBER  
RHO=DENSITY  
CP =SPECIFIC HEAT  
XC =CONDUCTIVITY  
EP =EMISSIVITY

```
MDEN=(MAT-1)*4+1
MSCP=MDEN+1
MCON=MSCP+1
MEP=MCON+1
CALI. INTP(T1,P,MDEN,RHO)
CALL INTP(T1,P,MSGP,CP)
CALL INTP(T1,P,MCON,XK)
CALL INTP(T1,P,MEP,EP)
RETURN
END
```

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SUBROUTINE RGAP

C SUBROUTINE COMPUTES EQUIVALENT THERMAL CONDUCTANCE THROUGH  
C RADIATION GAP

COMMON/GAP/T1,T2,TH1,TH2,TH3,TH,P,H,M1,M2,M3,TOL,BET,SIG,  
S XM,CAP1,CAP2,XK  
COMMON/PRESS/PRES

TT1=T1  
TT2=T2  
TAO=T1  
TBO=T2  
DO 100 I=1,100

C FIND PROPERTIES OF UPPER AND LOWER SURFACES

CALL PROP(TT1,PRES,M1,RHO1,CP1,XK1,EP1)  
CALL PROP(TT2,PRES,M2,RHO2,CP2,XK2,EP2)

F1=1.0/EP1  
F2=1.0/EP2  
SF=1.0/(F1+F2-1.0)  
YK1=XK1/TH1  
YK2=XK2/TH2  
YK3=SIG\*SF\*(TAO\*\*2+TBO\*\*2)\*(TAO+TBO)

C COMPUTE INTERIOR SURFACE TEMPERATURE OF UPPER AND LOWER SURFACE

TA=((T1\*YK1+TBO\*YK3)/(YK1+YK3))\*BET+(1.0-BET)\*TAO  
TB=((T2\*YK2+TAO\*YK3)/(YK2+YK3))\*BET+(1.0-BET)\*TBO  
TEST1=ABS((TA-TAO)/TAO)  
TEST2=ABS((TB-TBO)/TBO)

C CHECK FOR CONVERGENCE

IF(TEST1.LT.TOL.AND.TEST2.LT.TOL)GO TO 200  
TT1=(T1+TA)/2.0  
TT2=(T2+TB)/2.0  
TAO=TA  
TBO=TB

100 CONTINUE  
GO TO 300

200 CONTINUE

C COMPUTE EQUIVALENT CONDUCTANCE

XK=YK1\*YK2\*YK3/(YK1\*YK2+YK1\*YK3+YK2\*YK3)  
XM=RHO1\*TH1+RHO2\*TH2  
CAP1=RHO1\*TH1\*CP1  
CAP2=RHO2\*TH2\*CP2

300 CONTINUE  
RETURN  
END

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SUBROUTINE SRIPP(NN)  
C SUBROUTINE TO FIND RADIATION INTERCHANGE FACTORS  
C GIVEN EMISSIVITIES, GEOMETRIC VIEW FACTORS, AREAS USING THE  
C NETWORK METHOD  
COMMON/SF/AR(10),EPP(10),F(10,10),ASF(10,10)  
DIMENSION EB(10),XJ(10),XJN(10),RHO(10)  
TOL=.001  
BETA=.5  
EMPOW=1000.0  
N=NN  
DO 100 I=1,N  
RI=FLOAT(I)  
EB(I)=EMPOW\*RI  
RHO(I)=1.0-EPP(I)  
XJ(I)=EB(I)  
100 CONTINUE  
C ITERATE ON RADIOSITY,XJ  
DO 500 M=1,50  
TESTM=1.0E-8  
DO 200 J=1,N  
SUMJF=0.0  
F1=EPP(J)/(1.0-RHO(J)\*F(J,J))\*EB(J)  
F2=RHO(J)/(1.0-RHO(J)\*F(J,J))  
DO 300 K=1,N  
IF(J.EQ.K)GO TO 300  
SUMJF=SUMJF+XJ(K)\*F(J,K)  
300 CONTINUE  
XJN(J)=(1.0-BETA)\*XJ(J)+BETA\*(F1+F2\*SUMJF)  
TEST=ABS(XJ(J)-XJN(J))/XJ(J)  
IF(TEST.GT.TESTM)TESTM=TEST  
XJ(J)=XJN(J)  
200 CONTINUE  
IF(TESTM.LT.TOL)GO TO 600  
500 CONTINUE  
600 CONTINUE  
C COMPUTE NET EXCHANGE BY RADIOSITY DIFFERENCE  
C COMPUTE AREA\*SCRIPT "F" BY DIVISION BY BLACK BODY EMISSIVE POWER  
DO 800 I=1,N  
DO 900 J=1,N  
ASF(I,J)=0.0  
IF(I.EQ.J)GO TO 900  
ASF(I,J)=AR(I)\*F(I,J)\*(XJ(I)-XJ(J))/(EB(I)-EB(J))  
900 CONTINUE  
800 CONTINUE  
RETURN  
END

SUBROUTINE STAND

C SUBROUTINE TO COMPUTE EQUIVALENT CONDUCTANCE, MASS, CAPACITANCE  
C OF Z STANDOFF STRUCTURE

COMMON/GAP/T1,T2,TH1,TH2,TH3,TH,P,H,M1,M2,M3,TOL,BET,SIG,  
\$ XM,CAP1,CAP2,XK  
COMMON/FACT/XX(10,2),YY(10,2)  
COMMON/SF/AR(10),EPP(10),F(10,10),ASF(10,10)  
COMMON/PRESS/PRES  
H2=TH/2.0  
P2=P/2.0  
CONK=1.0E8  
T3=(T1+T2)/2.0  
VOL=TH1+TH2+(2.0\*M+TH)\*TH3/P

C SET COORDINATES OF INTERIOR TO COMPUTE VIEW FACTORS

DO 100 I=1,4  
XT=0.0  
YT=0.0  
DO 200 J=1,2  
IF(I.EQ.1.AND.J.EQ.1)YT=TH  
IF(I.EQ.1.AND.J.EQ.2)XT=P  
IF(I.EQ.2.AND.J.EQ.2)XT=P  
IF(I.EQ.3.AND.J.EQ.1)XT=P  
IF(I.EQ.3.AND.J.EQ.2)YT=TH  
IF(I.EQ.4.AND.J.EQ.2)YT=TH  
XX(I,J)=XT  
YY(I,J)=YT

200 CONTINUE

100 CONTINUE

C COMPUTE INTERIOR VIEW FACTORS

CALL VFAC(4)

DO 1000 I=1,100  
T30=T3

CALL PROP(T1,PRES,M1,RHO1,CP1,XK1,EPP(1))  
CALL PROP(T2,PRES,M2,RHO2,CP2,XK2,EPP(2))  
CALL PROP(T3,PRES,M3,RHO3,CP3,XK3,EPP(3))  
EPP(4)=EPP(3)

IF(I.NE.1)GO TO 1001

C GET RADIATION INTERCHANGE FACTORS

CALL SRIPF(4)

1001 CONTINUE

C COMPUTE CONDUCTION PATH VALUE

C1A=XK1\*TH1/P2  
C1B=CONK\*TH3/2.0  
C1C=XK3\*TH3/(2.0\*M2)  
C1=C1A\*C1B\*C1C/(C1A\*C1B+C1A\*C1C+C1B\*C1C)  
DIS=P2-(H+TH3)/2.0  
C3A=XK1\*TH1/DIS  
C3B=CONK\*(H+TH3/2.0)  
C3C=XK3\*TH3/(2.0\*M2)  
C3=C3A\*C3B\*C3C/(C3A\*C3B+C3A\*C3C+C3B\*C3C)  
C2C=C3C  
C2B=C3B  
C2A=C1A\*XK2/XK1  
C2=C2A\*C2B\*C2C/(C2A\*C2B+C2A\*C2C+C2B\*C2C)  
C4A=C1A\*XK2/XK1

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C4B=C1B  
C4C=C1C  
 $C4 = C4A * C4B * C4C / (C4A * C4B + C4A * C4C + C4B * C4C)$   
C COMPUTE RADIATION PATH VALUES  
 $R1 = ASF(1,4) * SIG * (T1**2 + T3**2) * (T1 + T3)$   
 $R2 = ASF(1,3) * SIG * (T1**2 + T3**2) * (T1 + T3)$   
 $R3 = ASF(4,2) * SIG * (T3**2 + T2**2) * (T3 + T2)$   
 $R4 = ASF(3,2) * SIG * (T3**2 + T2**2) * (T3 + T2)$   
 $R5 = ASF(1,2) * SIG * (T1**2 + T2**2) * (T1 + T2)$   
C FIND NEW STANDOFF TEMPERATURE  
 $T3N = (T1 * (R1 + R2 + C1 + C3) + T2 * (C2 + R3 + C4 + R4)) /$   
\$  $(R1 + R2 + R3 + R4 + C1 + C2 + C3 + C4)$   
 $T3 = (1.0 - BET) * T3 + BET * T3N$   
 $TEST = ABS(T3 - T30) / T3$   
IF(TEST.LT.TOL) GO TO 2000  
 $T30 = T3$   
1000 CONTINUE  
2000 CONTINUE  
C FIND MASS, CAPACITORS, AND EQUIVALENT CONDUCTIVITY  
 $XM = TH1 * RH01 + TH2 * RH02 + RH03 * (2.0 * H + TH) * TH3 / P$   
 $CAP1 = XM / 2.0 * CP1$   
 $CAP2 = XM / 2.0 * CP2$   
 $T12 = ABS(T1 - T2)$   
 $T13 = ABS(T1 - T3)$   
 $Q = (T12 * R5 + T13 * (C2 + C4 + R3 + R4)) / P$   
 $XK = Q / T12$   
RETURN  
END

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SUBROUTINE STRUCT(M,N)

C SUBROUTINE THAT HANDLES STRUCTURE FILE

C EITHER ADDING NEW STRUCTURES OR COPYING OLD STRUCTURE

PARAMETER (NMB1=20,NMB2=6,NMB3=3,NMB4=6,NMB6=10)

COMMON/INIT/TSTART,TSTOP,TIMPT,DTIM,NBP,NEXT,METRIC,

\$ METRIK,METRIX,NSTP,IPFLAG,TINI(NMB1),SINKT(NMB1),XFIJ(NMB1),

\$ MBP(NMB1),IN,IN2

COMMON/TITLE/CHAR2,CHAR1,FNAM1,FNAM3

COMMON/LD/LS(NMB1,NMB2),XP(NMB1,NMB2,NMB4),MATS(NMB1,NMB2,NMB3),

\$ NS(NMB1)

CHARACTER\*10 CHAR2(NMB6)

CHARACTER\*13 CHAR1(NMB6)

CHARACTER\*20 FNAM1,FNAM3

CHARACTER\*80 LABEL1,LABEL2

IFLAG=-9999

OPEN(UNIT=10,NAME=FNAM3,TYPE='OLD',RECORDSIZE=132)

IF(M.EQ.2)GO TO 500

C LOOKUP OLD STRUCTURE FROM STRUCTURE FILE

10 WRITE(IN2,20)MBP(N)

20 FORMAT(1H,'WHAT IS THE STRUCTURE NUMBER FOR BODY PT. 1,15,?')

- READ(IN,\*,ERR=10,END=1234)ND

C LOCATE STRUCTURE BY STRUCTURE NUMBER

35 READ(10,40,END=9000)NA

40 FORMAT(15)

IF(NA.EQ.ND)GO TO 150

IF(NA.EQ.IFLAG)GO TO 9000

GO TO 35

C REACHED END OF FILE WITHOUT LOCATING STRUCTURE NUMBER

9000 WRITE(IN2,9001)ND

9001 FORMAT(1H,'UNABLE TO FIND STRUCTURE NUMBER 1,15)

REWIND (UNIT=10)

GO TO 10

150 CONTINUE

BACKSPACE (UNIT=10)

C OBTAIN STRUCTURE DATA FOR DESIRED STRUCTURE NUMBER

READ(10,55)NA,NS(N),N3,N4

55 FORMAT(15,5X,3(15,5X))

READ(10,60)LABEL1,LABEL2

60 FORMAT(5X,A80,/,5X,A80)

WRITE(IN2,70)NA,LABEL1,LABEL2

70 FORMAT(/,10X,'STRUCTURE NUMBER = 1,15,/,,1X,A80,,1X,A80)

DO 200 IK=1,NS(N)

READ(10,210,END=1000)LS(N,I),(MATS(N,I,J),J=1,N3),

\$ (XP(N,I,J),J=1,N4)

210 FORMAT(10X,15,315,6E15.7)

200 CONTINUE

GO TO 1000

C

C ADD NEW STRUCTURE TO STRUCTURE FILE

500 CONTINUE

510 WRITE(IN2,520)MBP(N)

520 FORMAT(1H,'WHAT IS THE STRUCTURE NUMBER FOR BODY PT. 1,15)

READ(IN,\*,ERR=510,END=1234)ND

C SEARCH THROUGH STRUCTURE FILE

551 DO 550 IK=1,1000

```
      READ(10,40,END=560)NB
C  CHECK TO SEE IF STRUCTURE NUMBER ALREADY EXISTS
    IF(ND.EQ.NB)WRITE(11IN2,530)ND
  530 FORMAT(1H , 'STRUCTURE NUMBER ',15,' ALREADY EXISTS.')
    IF(ND.EQ.NB)REWIND (UNIT=10)
    IF(ND.EQ.NB)GO TO 510
C  CHECK FOR END OF FILE
    IF(NB.EQ.IFLAG)GO TO 560
  550 CONTINUE
    GO TO 551
  560 BACKSPACE (UNIT=10)
C  ADD NEW STRUCTURE DATA
    N4=NMB4
    N3=NMB3
    WRITE(10,570)ND,NS(N),N3,N4
  570 FORMAT(15,5X,3(15,5X))
    WRITE(11IN2,580)MBP(N)
  580 FORMAT(1H , 'GIVE A TWO LINE DESCRIPTION OF THE STRUCTURE',
    $ ' FOR BODY PT. ',15)
    READ(11IN,590)LABEL1,LABEL2
  590 FORMAT(A80,/,A80)
    WRITE(10,600)LABEL1,LABEL2
  600 FORMAT(5X,A80,/,5X,A80)
    DO 700 I=1,NS(N)
    WRITE(10,610)LS(N,I),(MATS(N,I,J),J=1,N3),(XP(N,I,J),J=1,N4)
  610 FORMAT(10X,15,315,6E15.7)
  700 CONTINUE
    WRITE(10,800)IFLAG
  800 FORMAT(15)
  1000 CONTINUE
    CLOSE (UNIT=10,STATUS='KEEP')
    RETURN
  1234 CONTINUE
    STOP
    END
```

SUBROUTINE SUBPR(X,N,Y)

C SUBROUTINE TO RETURN TEMPERATURE OF SUBLIMATION AND HEAT OF  
C SUBLIMATION, Y, GIVEN PRESSURE, X.

C

C N = 1 RETURN TEMPERATURE  
C N = 2 RETURN PRESSURE  
C X = PRESSURE

C

PARAMETER (NMB9=41)  
COMMON/CSUB/CCS(2,NMB9)  
J=CCS(N,1)  
J1=J#2  
XLST=CCS(N,2)  
XHST=CCS(N,J1)  
IF(X.EQ.XLST)GO TO 700  
IF(X.LT.XLST)GO TO 100  
IF(X.GT.XHST)GO TO 200  
JT=4

500 CONTINUE

C SEARCH INDEPENDENT VARIABLE  
IF(CCS(N,JT)=X)300,600,400

300 CONTINUE

C GO BACK AND CHECK ANOTHER  
JT=JT+2  
GO TO 500

400 CONTINUE

C X LIES BETWEEN CCS(N,N1) AND CCS(N,N3)  
N1=JT-2  
N2=N1+1  
N3=JT  
N4=N3+1  
GO TO 900

600 CONTINUE

C X EQUAL TO A INDEPENDENT VARIABLE  
NT=JT+1  
Y=CCS(N,NT)  
GO TO 1000

700 CONTINUE

C X EQUAL TO LOWEST INDEPENDENT VARIABLE  
Y=CCS(N,3)  
GO TO 1000

100 CONTINUE

C X LESS THAN LOWEST INDEPENDENT VARIABLE, EXTRAPOLATE  
N1=2  
N2=3  
N3=4  
N4=5  
GO TO 900

200 CONTINUE

C X GREATER THAN HIGHEST INDEPENDENT VARIABLE, EXTRAPOLATE  
N1=2#J-2  
N2=N1+1  
N3=2#J  
N4=N3+1

900 CONTINUE

C FIND SLOPE AND INTERPOLATE

SL=(CCS(N,N4)-CCS(N,N2))/(CCS(N,N3)-CCS(N,N1))

Y=CCS(N,N2)+SL\*(X-CCS(N,N1))

1000 CONTINUE

RETURN.

END

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SUBROUTINE THINS  
C SUBROUTINE TO SET CAPACITANCE, MASS, AND CONDUCTOR OF INFINITELY  
C CONDUCTING OR THERMALLY THIN PLATE  
COMMON/GAP/T1,T2,TH1,TH2,TH3,TH,P,H,M1,M2,M3,TOL,BET,SIG,XM,  
S CAP1,CAP2,XK  
COMMON/PRESS/PRES  
T=(T1+T2)/2.0  
CALL PROP(T,PRES,M1,R0,CP,XK,EP)  
CAP1=R0\*CP\*TH/2.0  
CAP2=CAP1  
XM=R0\*TH  
XK=1.0E10  
RETURN  
END

SUBROUTINE TMSTEP(DTSM,1)

C SUBROUTINE TO COMPUTE STABLE TIME STEP  
PARAMETER (NMB1=20,NMB2=6,NMB3=3,NMB4=6,NMB5=40)  
COMMON/TIME/NNDS,CONV,CRAD,STAB,ISBFG,NCDS  
COMMON/ARA/T(NMB5),TO(NMB5),C(NMB5),CD(NMB5),ICD(NMB5),L(NMB5,2)  
COMMON/LD/LS(NMB1,NMB2),XP(NMB1,NMB2,NMB4),MATS(NMB1,NMB2,NMB3),  
S NS(NMB1)  
DTSM=1.0E30

C GENERAL HEAT BALANCE NODE  
DO 379 J=1,NNDS  
IF(J.NE.1)GO TO 380

C SURFACE NODE  
C1=CONV+CRAD  
C2=CD(1)  
GO TO 381

380 CONTINUE  
IF(J.NE.NNDS)GO TO 370

C LAST NODE  
C1=CD(NCDS)  
C2=0.0  
GO TO 381

370 CONTINUE

C GENERAL NODE  
JM1=J-1  
C1=CD(JM1)  
C2=CD(J)

381 CONTINUE

C IF ADJACENT CONDUCTORS ARE THIN SKIN, SKIP  
IF(C1.GT.1.0E9) GO TO 379  
IF(C2.GT.1.0E9) GO TO 379  
STEST=C(J)/((C1+C2)\*STAB)  
IF(STEST.LT.DTSM)DTSM=STEST

379 CONTINUE

C THIN SKIN ALGORITHM  
IF(ISBFG.EQ.0)GO TO 382  
DO 387 KAP=1,NCDS  
JJ=ICD(KAP)  
JN=LS(1,JJ)  
IF(JN.NE.6)GO TO 387  
N1=L(KAP,1)  
N2=L(KAP,2)  
KP1=KAP+1  
KM1=KAP-1  
IF(KAP.EQ.1)GO TO 388  
IF(KAP.EQ.NCDS)GO TO 389

C GENERAL THIN SKIN NODE  
STEST=(C(N1)+C(N2))/((CD(KM1)+CD(KP1))\*STAB)  
GO TO 390

388 CONTINUE

C SURFACE NODE  
STEST=(C(N1)+C(N2))/((CONV+CRAD+CD(KP1))\*STAB)  
GO TO 390

389 CONTINUE

C ADIABATIC BACK SIDE  
STEST=(C(N1)+C(N2))/(CD(KM1)\*STAB)

390 CONTINUE  
IF (STEST.LT.DTSM)DTSM=STEST  
387 CONTINUE  
382 CONTINUE  
RETURN  
END

SUBROUTINE VFAC(NN).

C           THEORY OF CROSSED STRINGS (PLANAR 2-D)

C           COMMON/FACT/XX(10,2),YY(10,2)  
COMMON/SF/AR(10),EPP(10),F(10,10),ASF(10,10)  
N=NN  
DO 100 I=1,N  
SUMF=0.0

C FIND AREA OF SURFACE I  
CALL DIST(I,1,I,2,AR(I))  
DO 200 J=1,N

C FIND LENGTHS OF CROSSED AND UNCROSSED STRINGS  
CALL DIST(I,1,J,1,D11)  
CALL DIST(I,1,J,2,D12)  
CALL DIST(I,2,J,1,D21)  
CALL DIST(I,2,J,2,D22)

C FIND WHICH ARE CROSSED  
S1=D11+D22  
S2=D12+D21  
A1=S1  
A2=S2  
IF(S1.GT.S2)GO TO 201  
A1=S2  
A2=S1

201       CONTINUE  
F(I,J)=0.0  
IF(I.EQ.J)GO TO 200  
F(I,J)=(A1-A2)/(2.0\*AR(I))

C SUM OF VIEW FACTORS SHOULD EQUAL ONE FOR ENCLOSURE  
SUMF=SUMF+F(I,J)

200       CONTINUE  
100       CONTINUE  
RETURN  
END